



Faculty of Automotive and Construction Machinery Engineering

WARSAW UNIVERSITY OF TECHNOLOGY

Theory of Machines and Automatic Control Winter 2019/2020

Lecturer: Sebastian Korczak, PhD Eng.

Lecture 10

Classification of basic automatic systems
with examples.

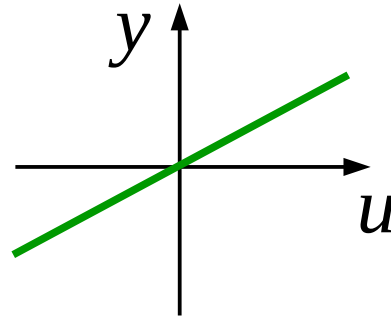
Classification of basic automatic systems

Element name	Transfer function
proportional	k
first order (inertial)	$\frac{k}{Ts + 1}$
integrator	$\frac{k}{s}$
differentiator	ks
differentiator with inertia	$\frac{ks}{Ts + 1}$
delay	$e^{-\tau s}$
second order (oscillator)	$\frac{k}{T_1^2 s^2 + T_2 s + 1}$

Proportional element

1. Element equation: $y(t) = ku(t)$ $u(t)$ - input, $y(t)$ - output

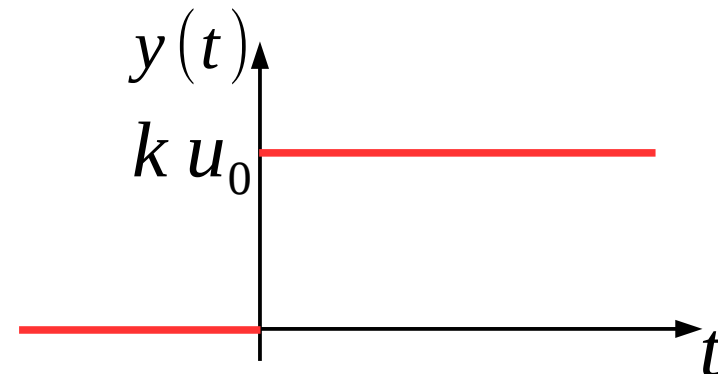
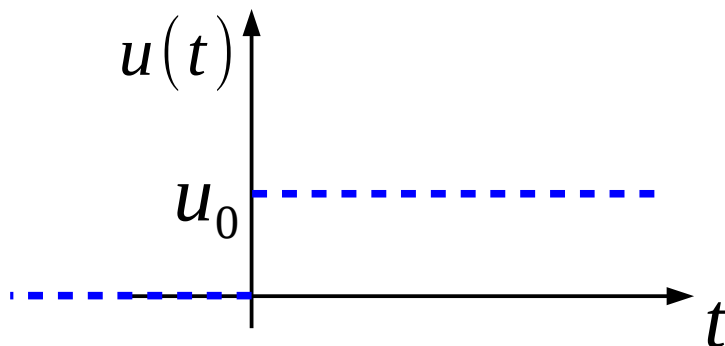
2. Static characteristic (steady state): $y = ku$ for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



for $k > 0$

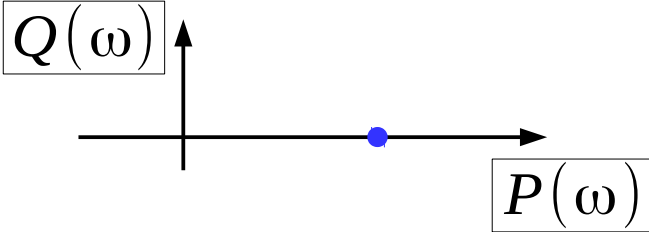
3. Transfer function: $H(s) = k$

4. Step response: $y(t) = k u_0 1(t)$ for $u(t) = u_0 1(t)$



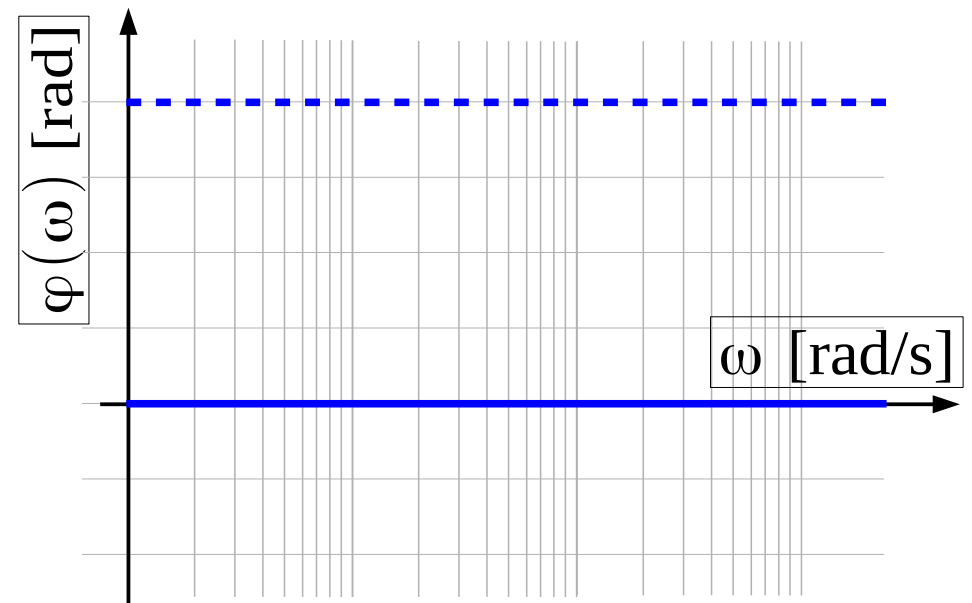
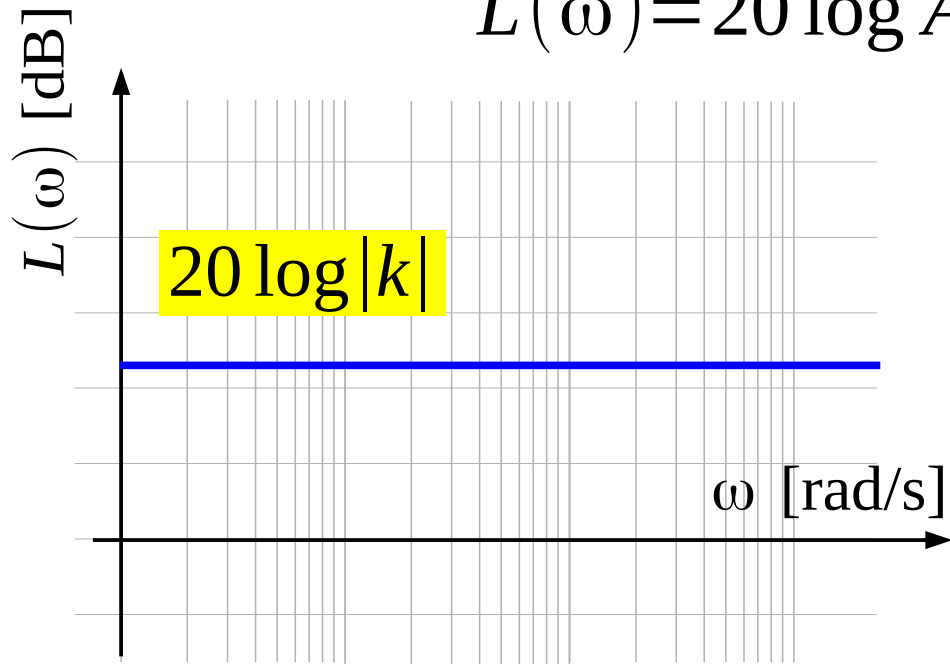
Proportional element

5. Frequency response: $H(j\omega) = k$ $P(\omega) = k, Q(\omega) = 0$

6. Nyquist plot:  for $k > 0$

The Nyquist plot shows a horizontal axis labeled $P(\omega)$ and a vertical axis labeled $Q(\omega)$. A single blue dot is located on the positive $P(\omega)$ axis, representing the frequency response for $k > 0$.

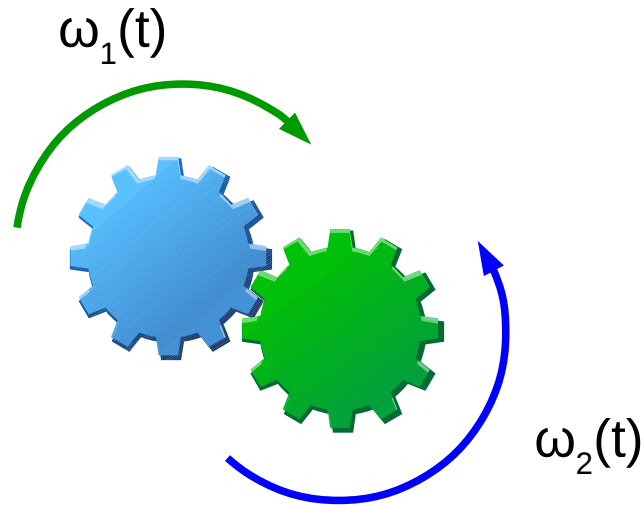
7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k|$ $\varphi(\omega) = \arctan \frac{Q}{P} = \begin{cases} 0, & \text{dla } k \geq 0 \\ \pi, & \text{dla } k < 0 \end{cases}$
 $L(\omega) = 20 \log A(\omega)$



Proportional element

Examples

1

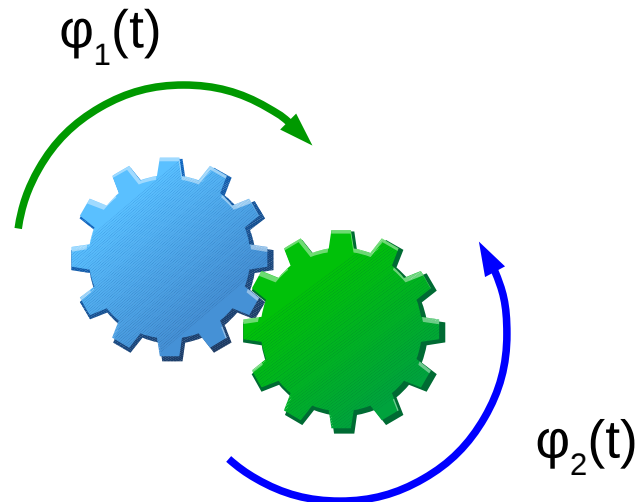


GEARBOX:

input – angular velocity $\omega_1(t)$

output – angular velocity $\omega_2(t)$

2



GEARBOX:

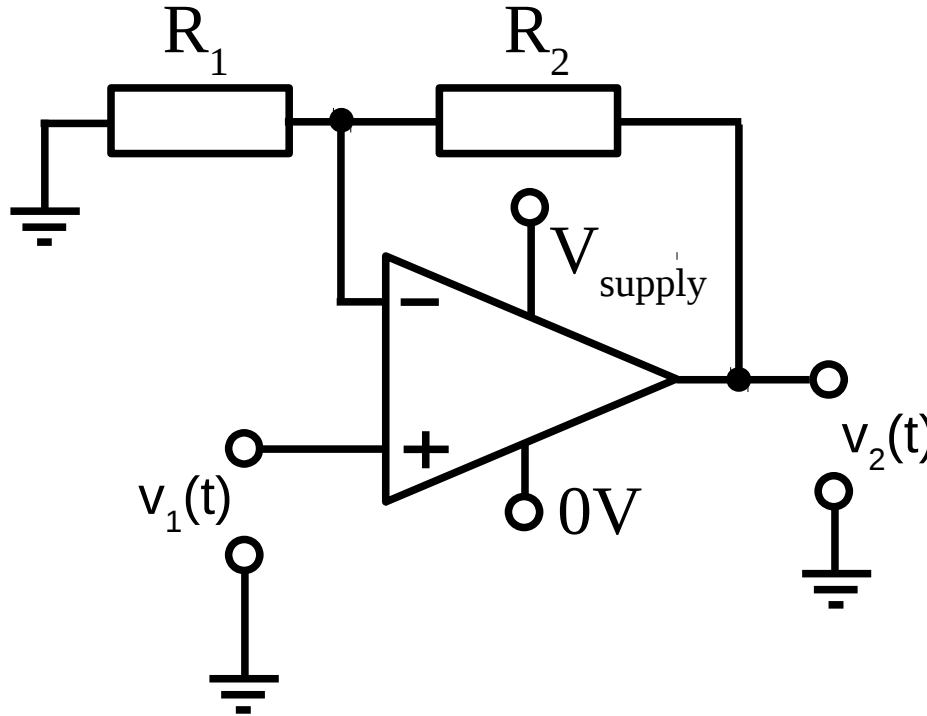
input – rotation angle $\varphi_1(t)$

output – rotation angle $\varphi_2(t)$

Proportional element

Examples

3



OPERATIONAL AMPLIFIER:
input – voltage $v_1(t)$
output – voltage $v_2(t)$

$$v_2(t) = v_1(t) \left(1 + \frac{R_2}{R_1} \right)$$

4

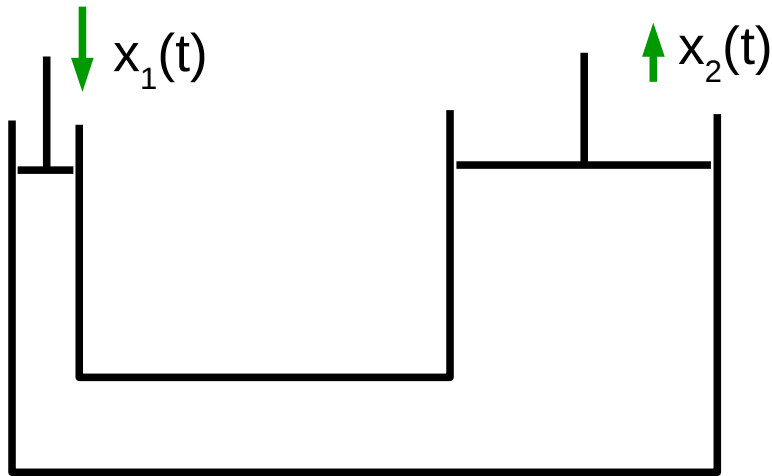


BEAM in steady state:
input – force F_1
output – force F_2

Proportional element

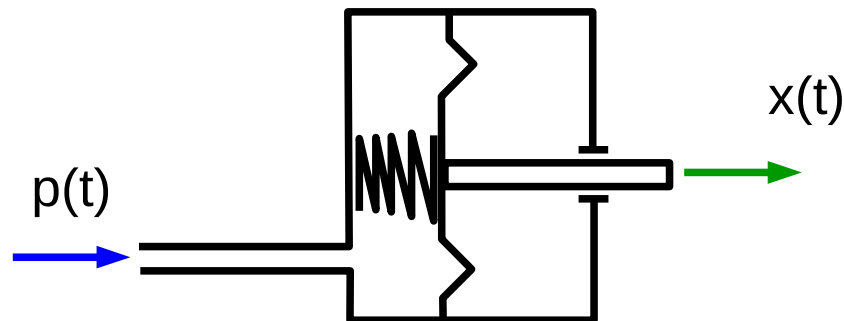
Examples

5



HYDRAULIC LEVER:
input – displacement $x_1(t)$
output – displacement $x_2(t)$

6



PRESSURE ACTUATOR:
input – pressure $p_1(t)$
output – displacement $x(t)$

First-order inertial element

1. Element equation: $T \frac{dy(t)}{dt} + y(t) = ku(t)$

$u(t)$ - input
 $y(t)$ - output

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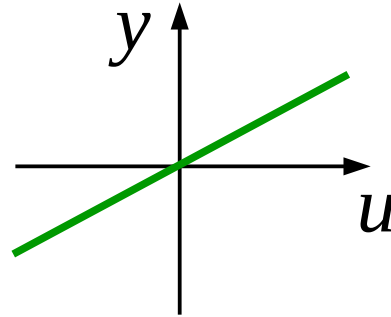
2. Static characteristic (steady state):

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for $k > 0$

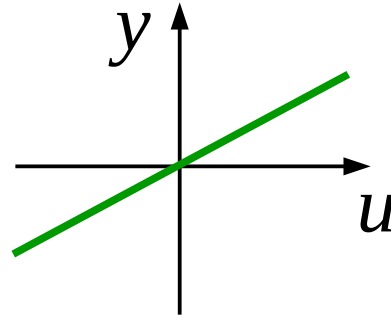
3. Transfer function:

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3. Transfer function: $H(s) = \frac{k}{Ts + 1}$

First-order inertial element

4. Step response:

First-order inertial element

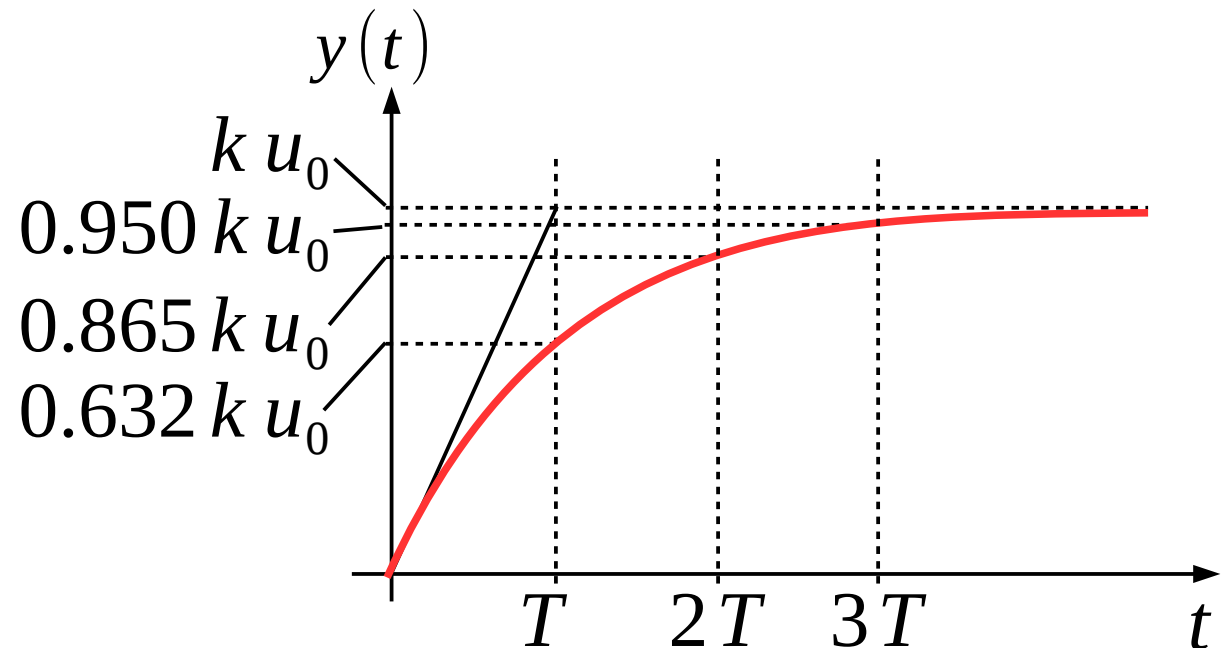
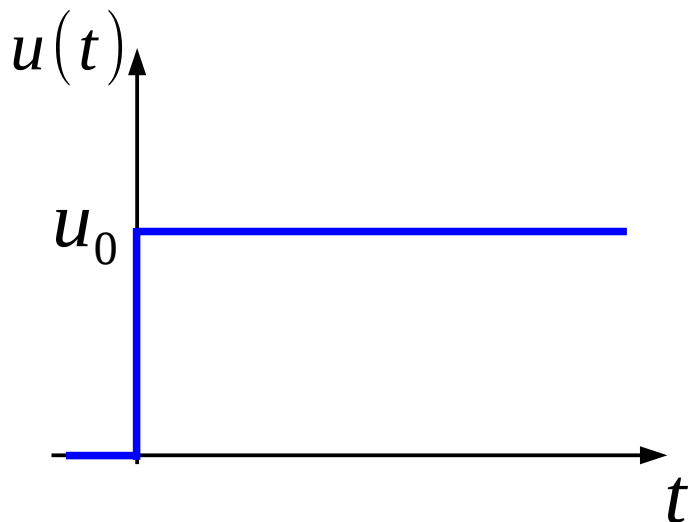
4. Step response:

$$\text{input: } u(t) = u_0 \mathbf{1}(t)$$

$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

$$\text{Laplace of output: } Y(s) = H(s) U(s) = \frac{k u_0}{s(Ts + 1)}$$

$$\text{output: } y(t) = L^{-1}\{Y(s)\} = k u_0 (1 - e^{-t/T})$$



First-order inertial element

5. Frequency response:

First-order inertial element

5. Frequency response: $H(j\omega) = \frac{k}{Tj\omega + 1}$

$$P(\omega) = \frac{k}{T^2\omega^2 + 1}, \quad Q(\omega) = \frac{-kT\omega}{T^2\omega^2 + 1}$$

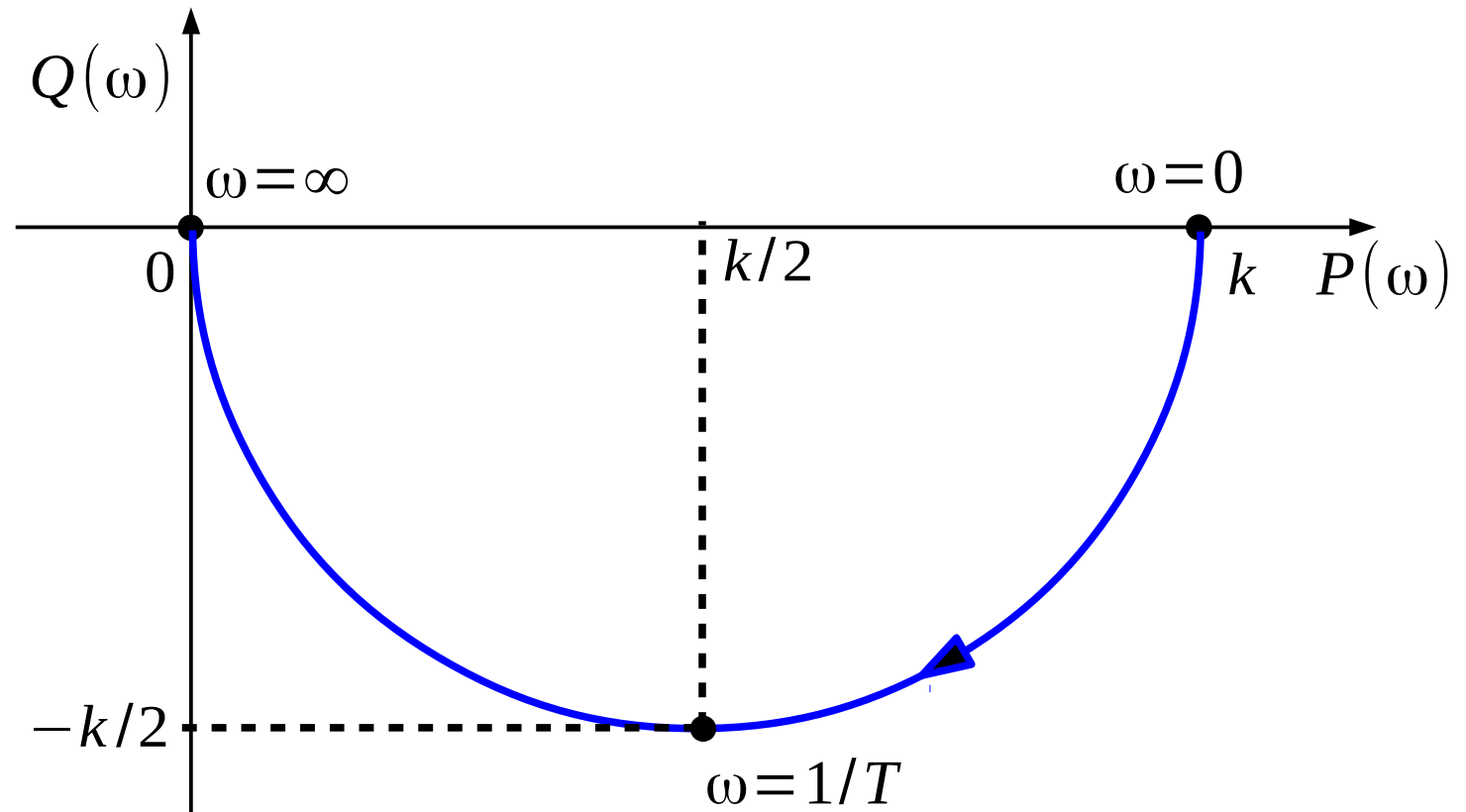
6. Nyquist plot:

First-order inertial element

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6. Nyquist plot:
for $k > 0$



First-order inertial element

7. Bode plot:

$$P(\omega) = \frac{k}{T^2 \omega^2 + 1}, \quad Q(\omega) = \frac{-k T \omega}{T^2 \omega^2 + 1}$$

First-order inertial element

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k| / \sqrt{T^2 \omega^2 + 1}$

$$L(\omega) = 20 \log A(\omega) = 20 \log |k| - 20 \log \sqrt{T^2 \omega^2 + 1}$$

$$\varphi(\omega) = \arctan \frac{Q}{P} = \arctan(-T \omega)$$

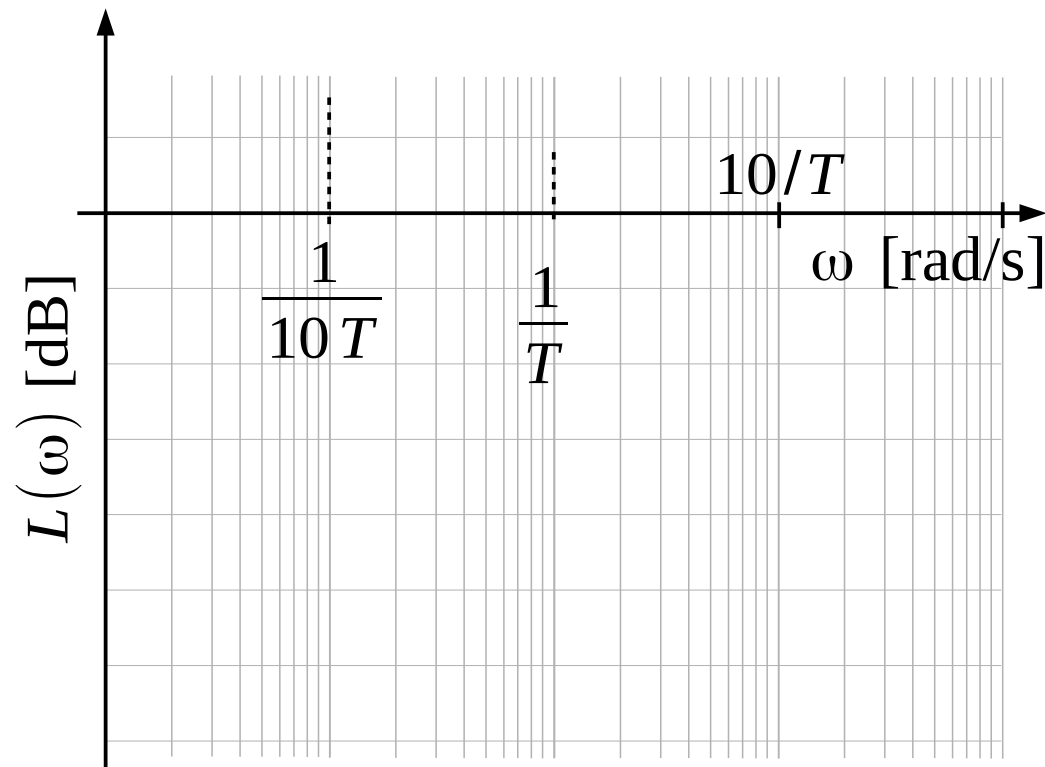
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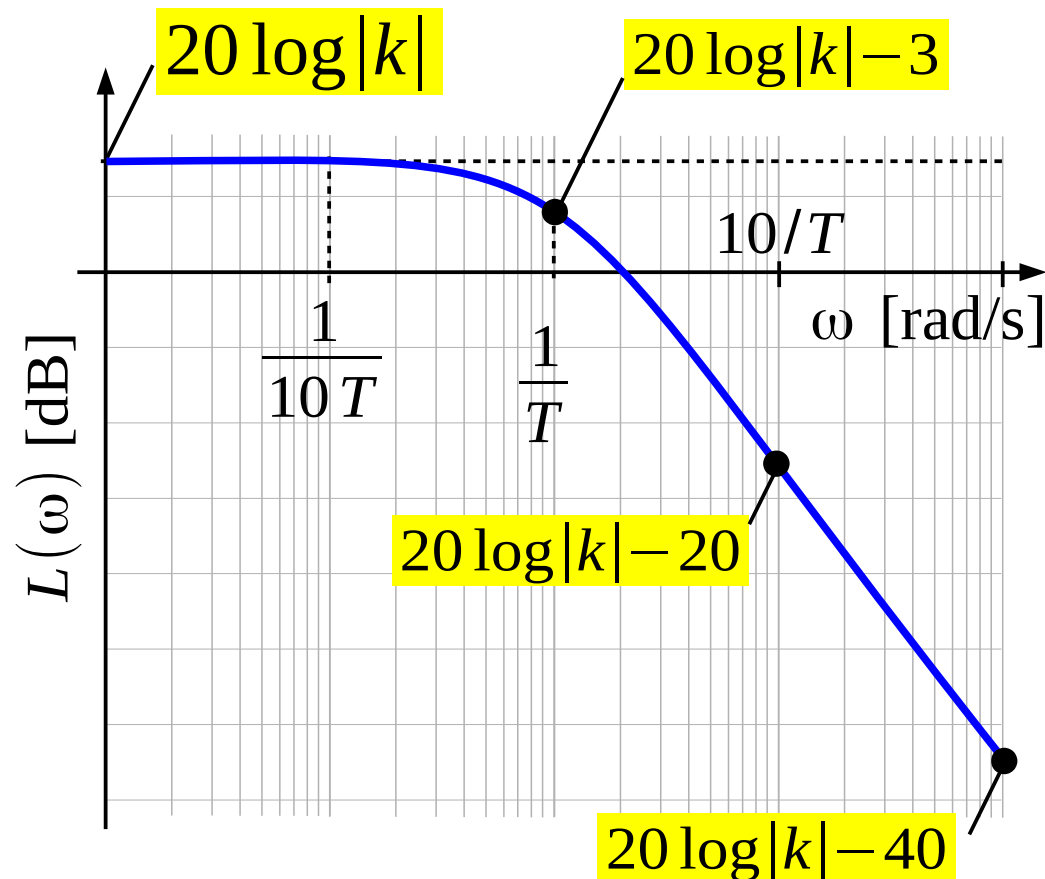
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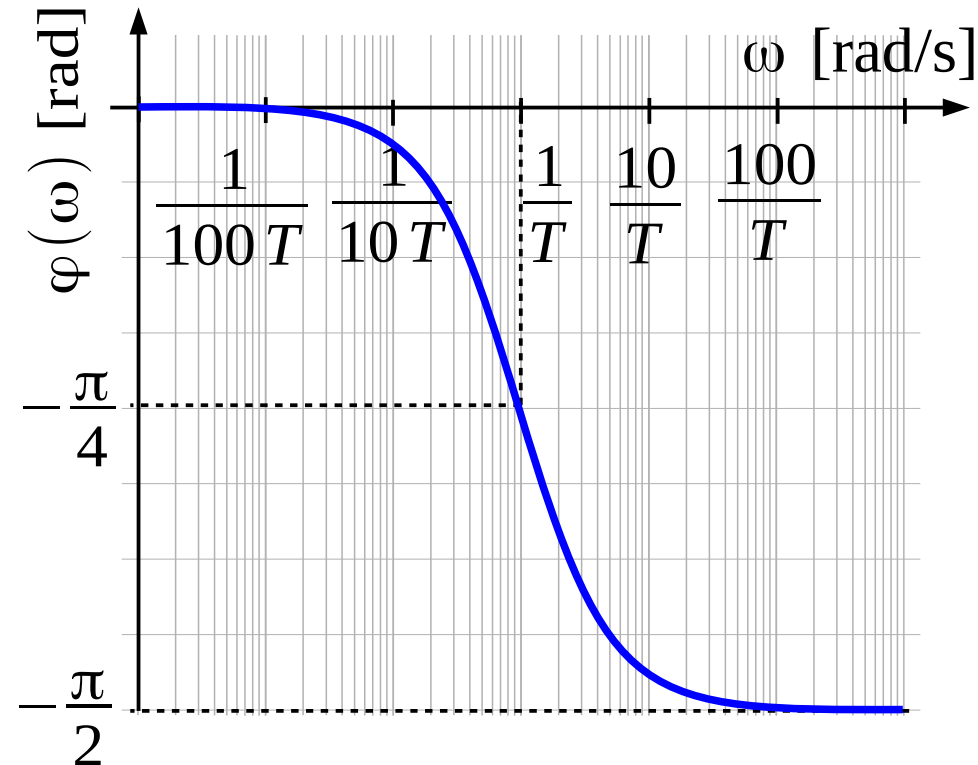
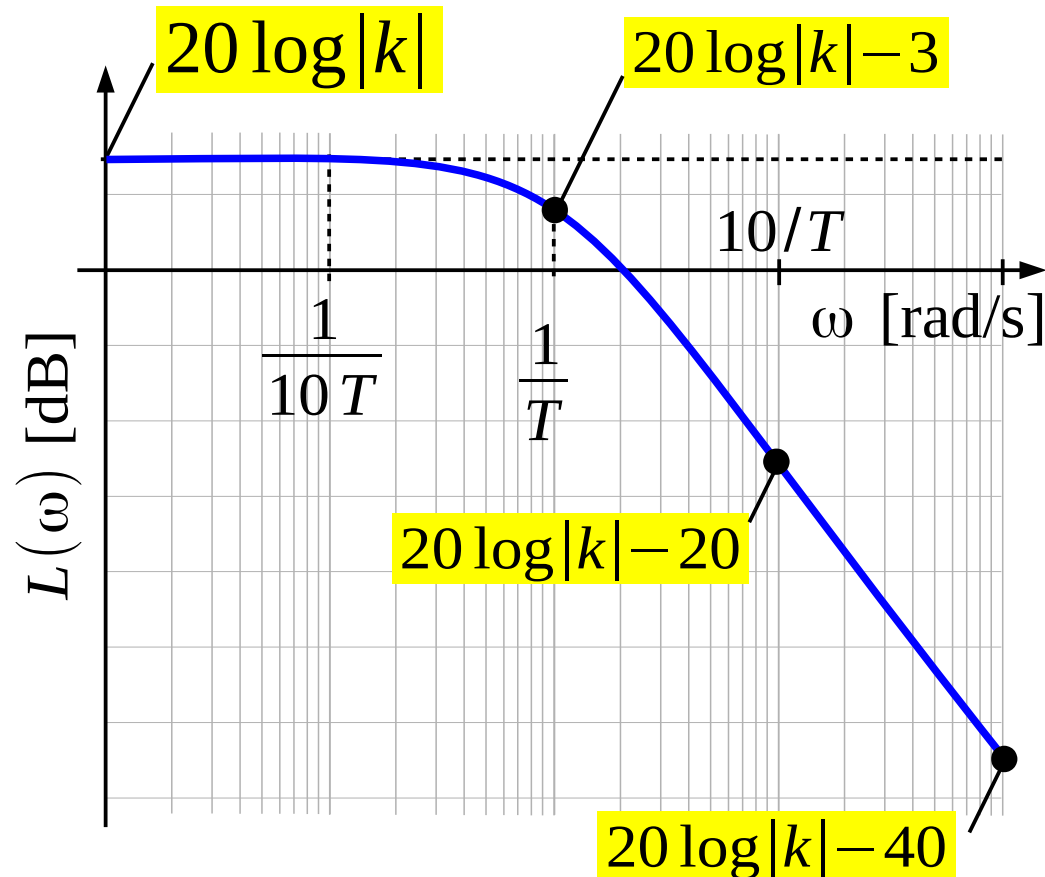


First-order inertial element

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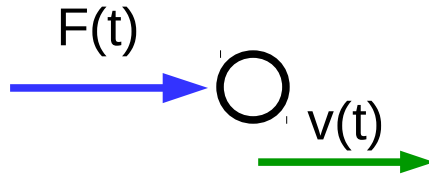
$$\varphi(\omega) = \arctan \frac{Q}{P} = \arctan(-T \omega) \quad \text{for } k > 0$$



First-order inertial element

Examples

1



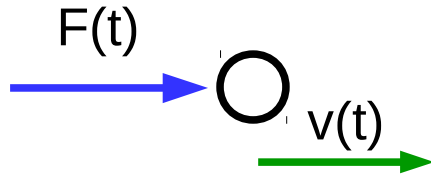
LINEAR MOTION OF A MATERIAL POINT WITH LINEAR DAMPING:
input – force $F(t)$
output – velocity $v(t)$

example: car is driving on a flat surface with air resistance proportional to its velocity, described using machine equation of motion, with assumption of constant reduced mass.

First-order inertial element

Examples

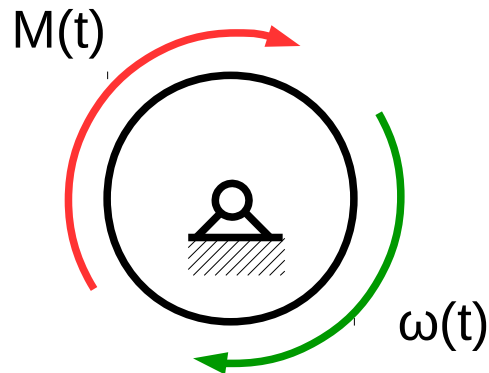
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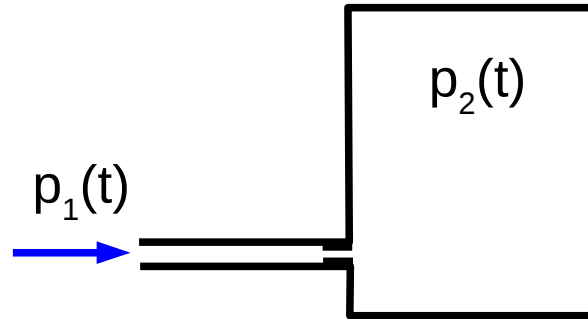


ANGULAR MOTION OF A RIGID BODY WITH LINEAR DAMPING:
input – torque $M(t)$
output – angular velocity $\omega(t)$

First-order inertial element

Examples

3

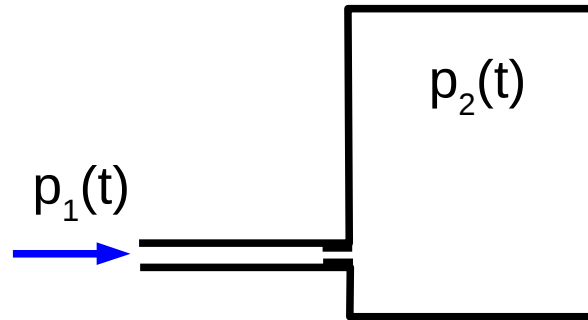


AIR CONTAINER:
input – pressure $p_1(t)$
output – pressure $p_2(t)$

First-order inertial element

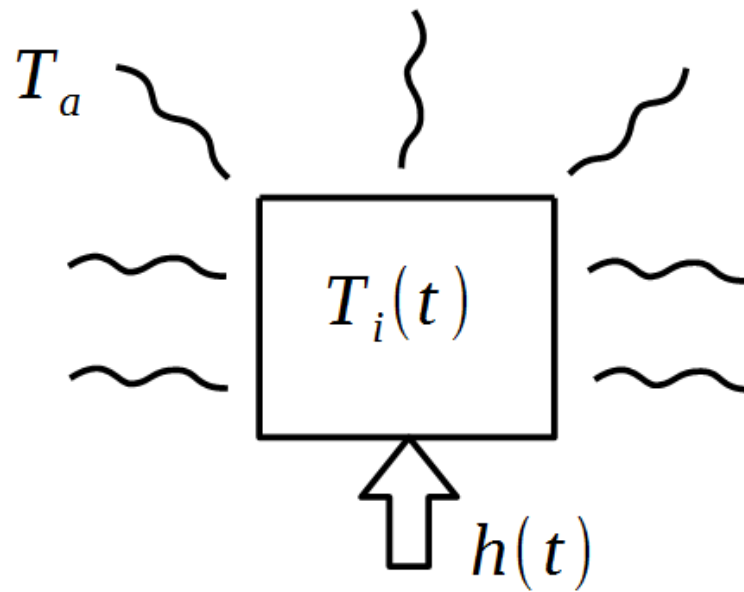
Examples

3



AIR CONTAINER:
input – pressure $p_1(t)$
output – pressure $p_2(t)$

4



HEATED OBJECT WITH SMALL
INERTIA:
input – heater power $h(t)$
output – object temperature $T_i(t)$

Integrator

1. Element equation: $\frac{dy(t)}{dt} = k u(t)$

$u(t)$ - input
 $y(t)$ - output

Integrator

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2. Static characteristic (steady state):

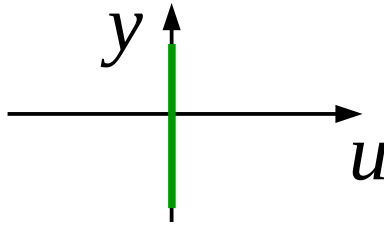
Integrator

1. Element equation: $\frac{dy(t)}{dt} = k u(t)$

$u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state): $u = 0$

for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function:

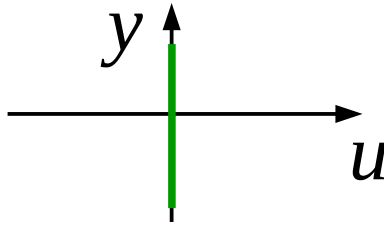
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3. Transfer function: $H(s) = \frac{k}{s}$

Integrator

4. Step response:

input: $u(t) = u_0 \mathbf{1}(t)$

Laplace of input: $U(s) = u_0 \frac{1}{s}$

Integrator

4. Step response:

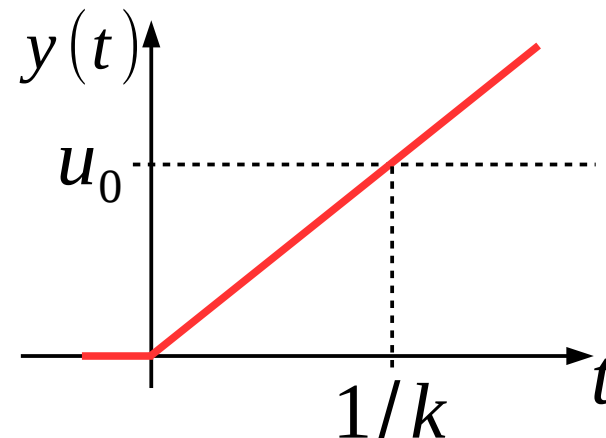
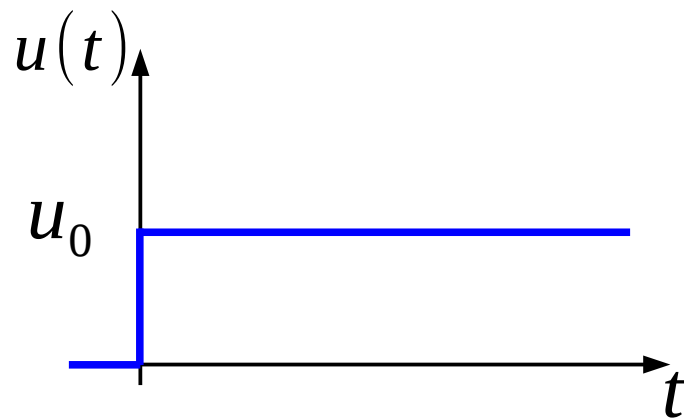
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for $k > 0$



Integrator

5. Frequency response:

Integrator

5. Frequency response: $H(j\omega) = \frac{k}{j\omega}$

$$P(\omega) = 0, \quad Q(\omega) = -\frac{k}{\omega}$$

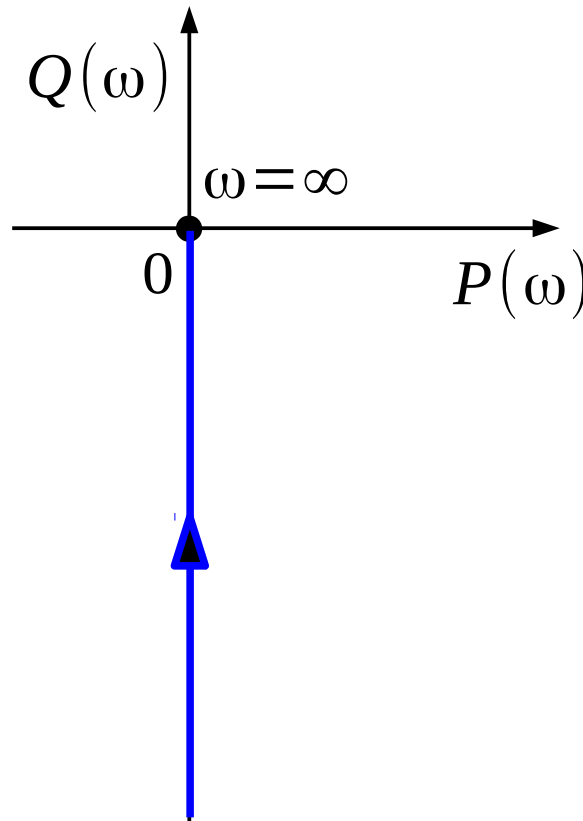
6. Nyquist plot:

Integrator

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for $k > 0$



Integrator

7. Bode plot:

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Integrator

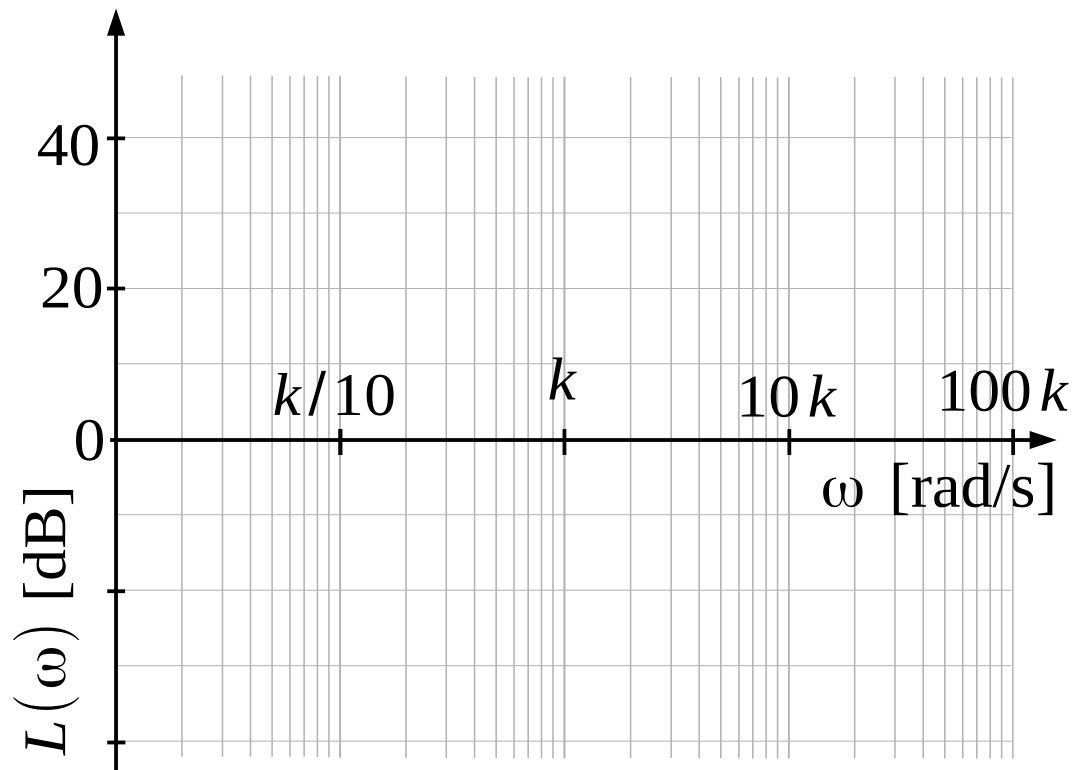
7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = \left| \frac{k}{\omega} \right|$

$$L(\omega) = 20 \log A(\omega) = 20 \log \left| \frac{k}{\omega} \right| \quad \varphi(\omega) = \arctan \frac{Q}{P} = \arctan(-\infty)$$

Integrator

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = \left| \frac{k}{\omega} \right|$ for $k > 0$

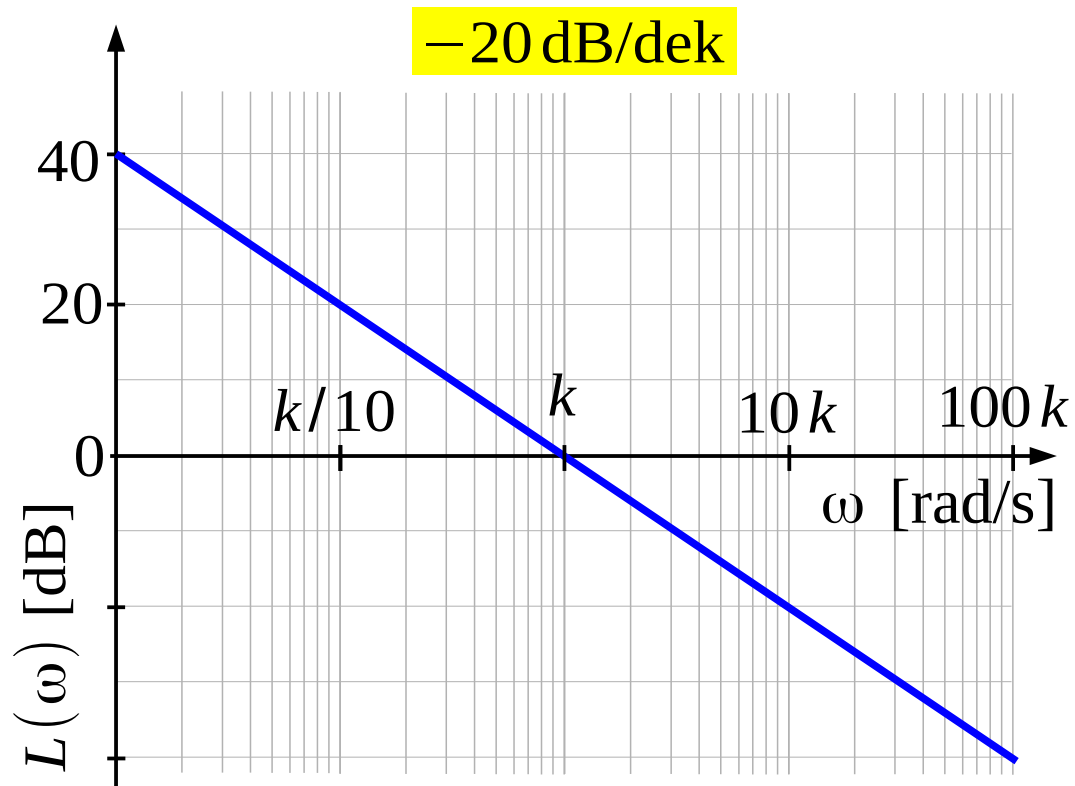
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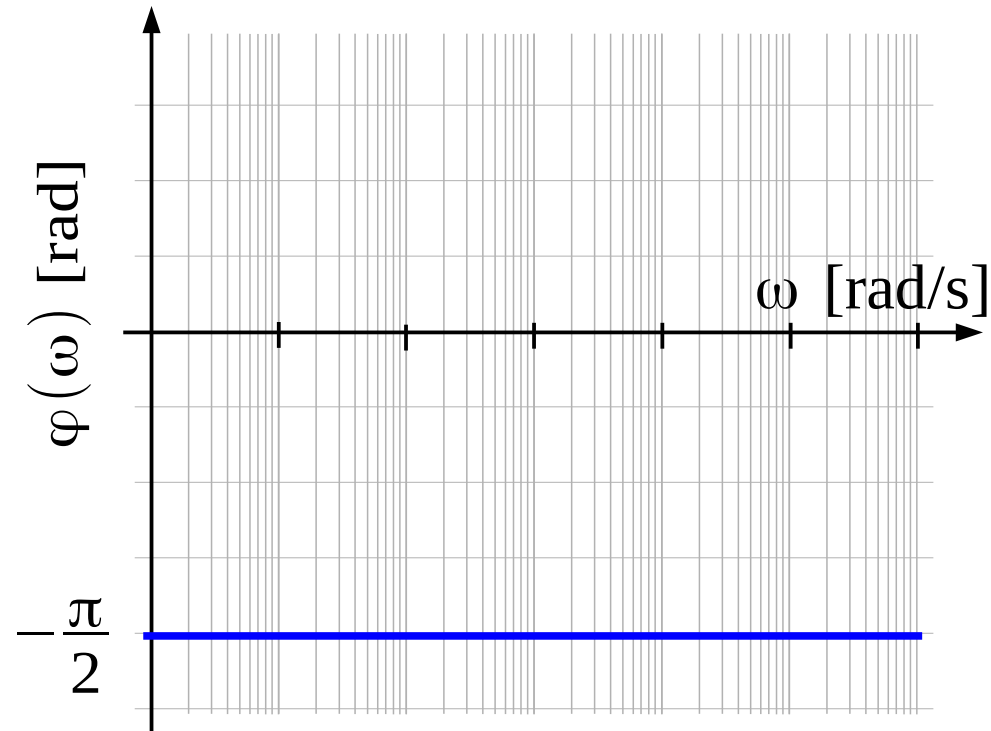
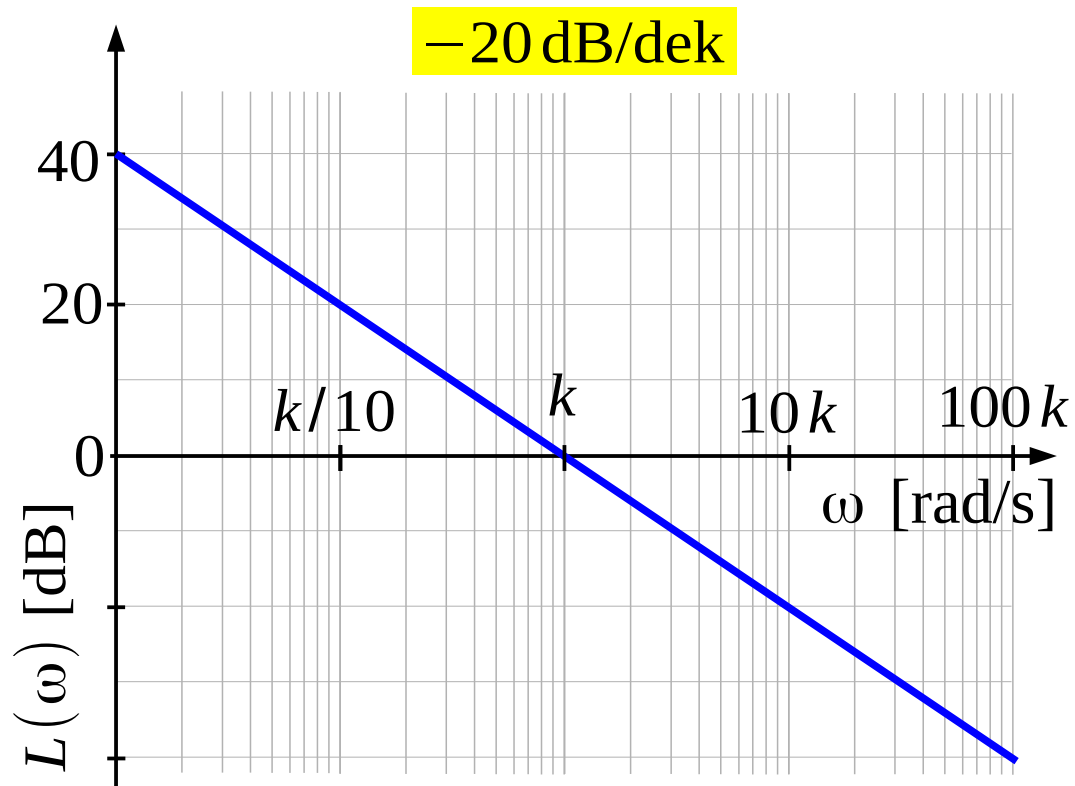
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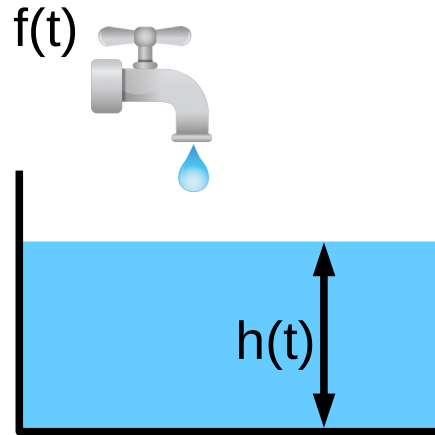
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Integrator

Examples

1

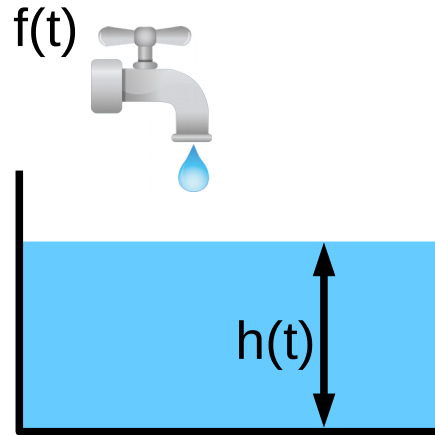


PRISM LIQUID TANK:
input – liquid inflow $f(t)$
output – liquid level $h(t)$

Integrator

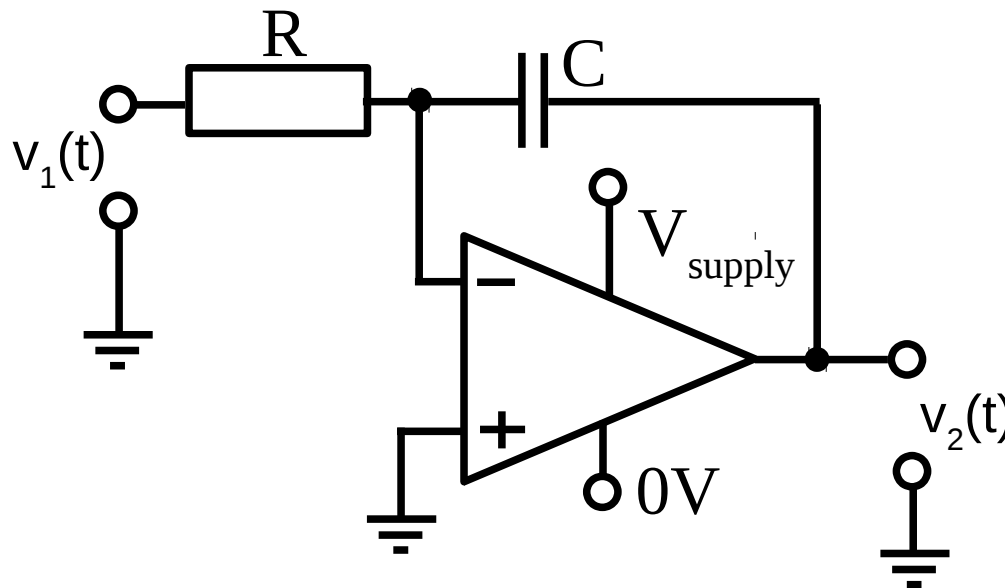
Examples

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PRISM LIQUID TANK:
input – liquid inflow $f(t)$
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②



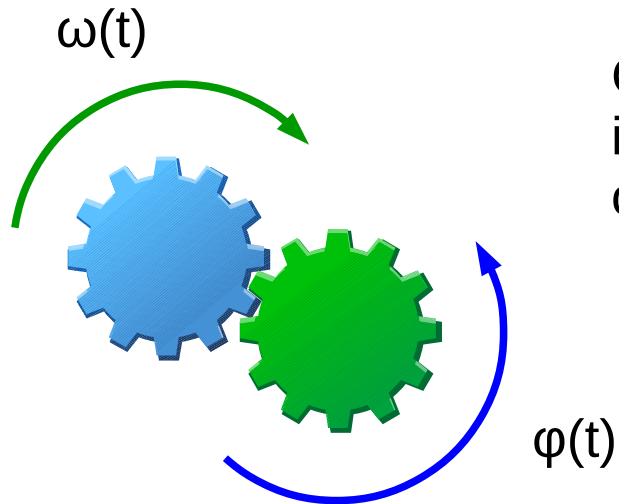
OPERATIONAL AMPLIFIER:
input – voltage $v_1(t)$
output – voltage $v_2(t)$

$$v_2(t) = \frac{1}{RC} \int_0^t v_1(t) dt$$

Integrator

Examples

3



GEARBOX:

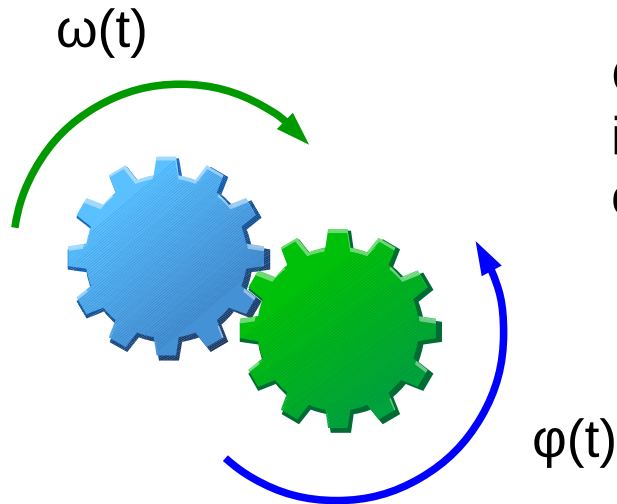
input – angular velocity $\omega(t)$

output – rotation angle $\phi(t)$

Integrator

Examples

3

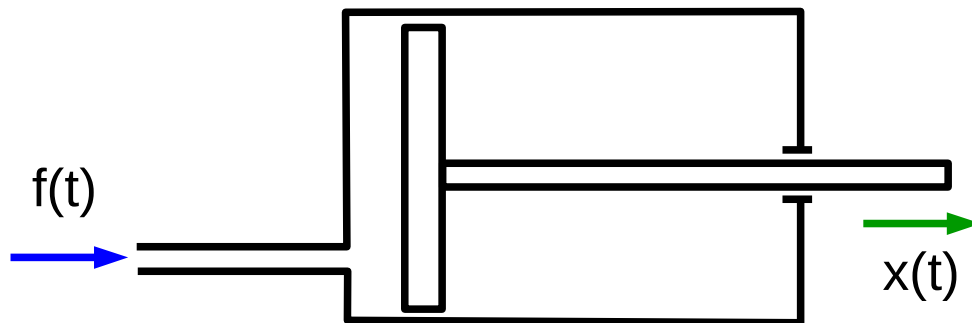


GEARBOX:

input – angular velocity $\omega(t)$

output – rotation angle $\phi(t)$

4



HYDRAULIC CYLINDER:

input – volume inflow $f(t)$

output – displacement $x(t)$

Differentiator

1. Element equation: $y(t) = k \frac{du(t)}{dt}$

$u(t)$ - input
 $y(t)$ - output

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2. Static characteristic (steady state):

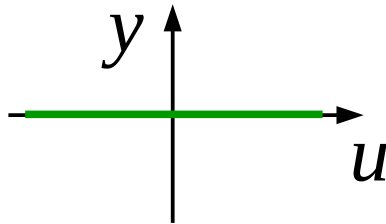
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for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function:

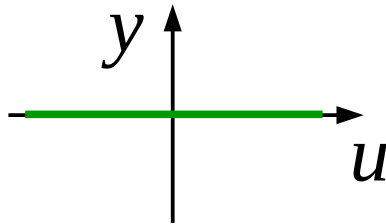
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Differentiator

4. Step response:

$$\text{input: } u(t) = u_0 1(t)$$

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Differentiator

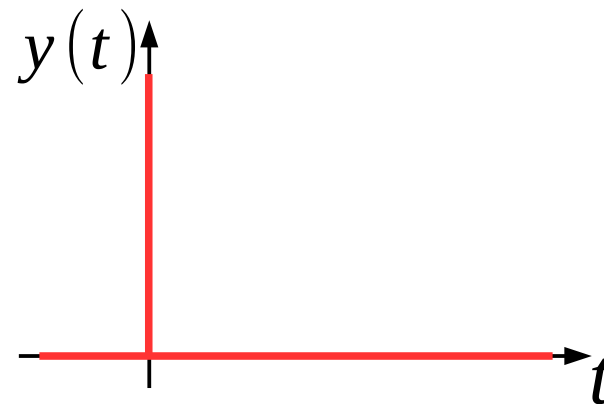
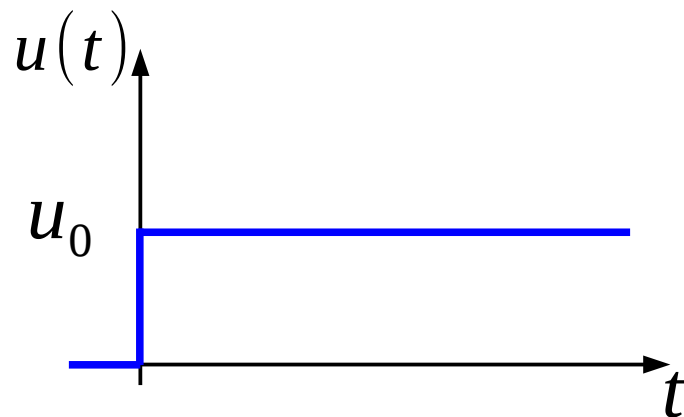
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$$\text{Laplace of output: } Y(s) = H(s) U(s) = k u_0$$

$$\text{output: } y(t) = L^{-1}\{Y(s)\} = k u_0 \delta(t)$$



Differentiator

5. Frequency response:

Differentiator

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$$P(\omega) = 0, \quad Q(\omega) = k\omega$$

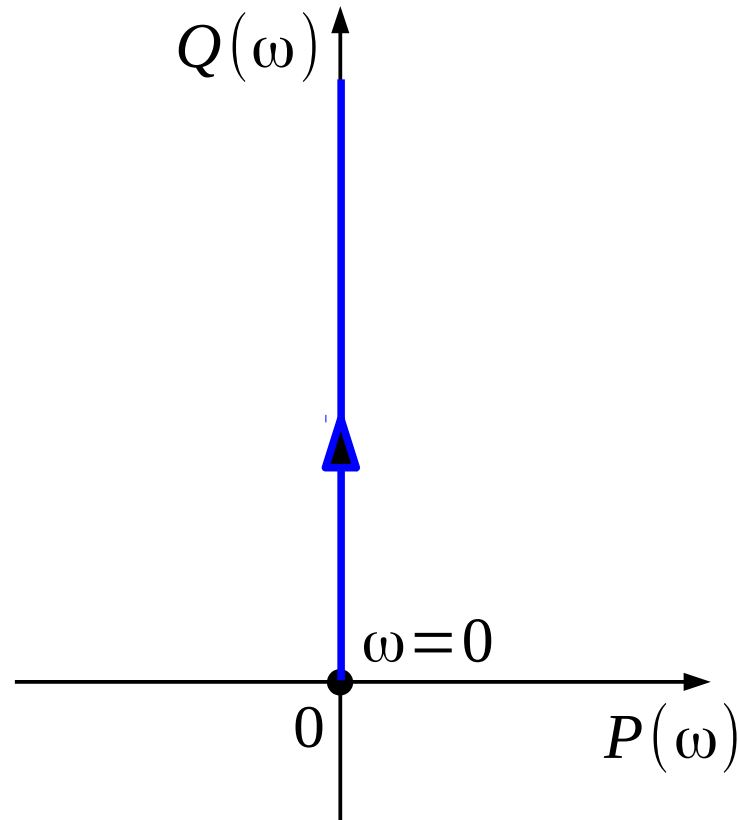
6. Nyquist plot:

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6. Nyquist plot:
for $k > 0$



Differentiator

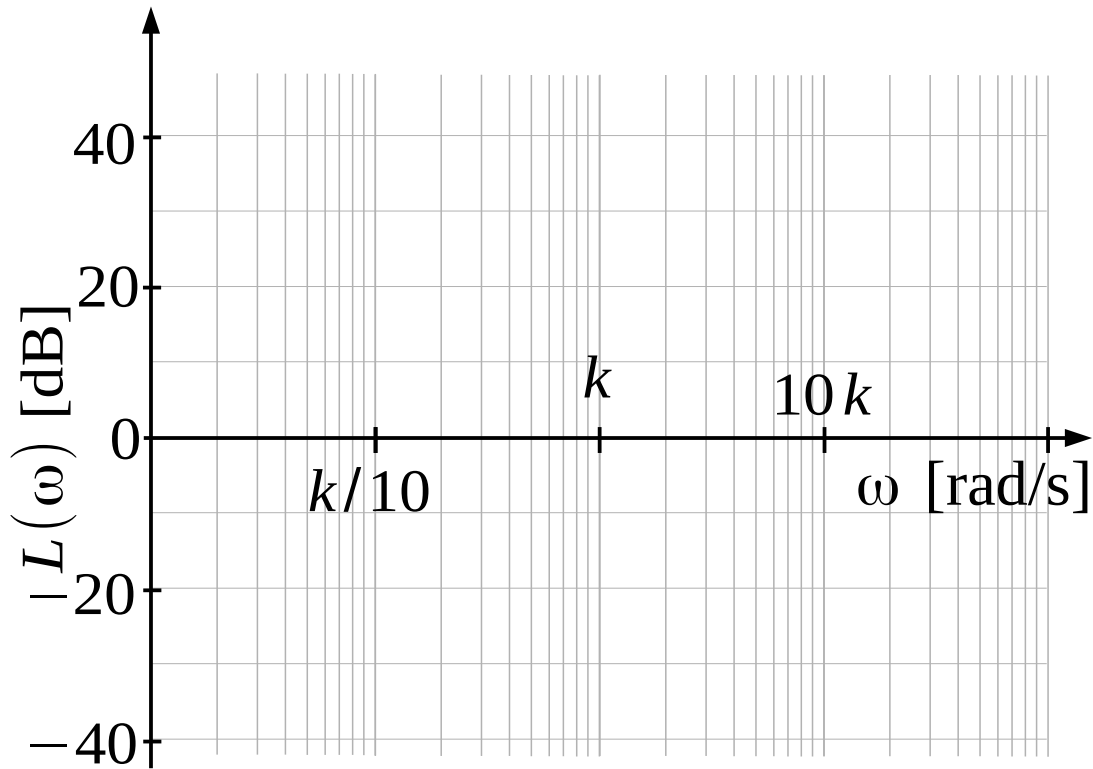
7. Bode plot:

$$P(\omega) = 0, \quad Q(\omega) = k\omega$$

Differentiator

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k \omega|$

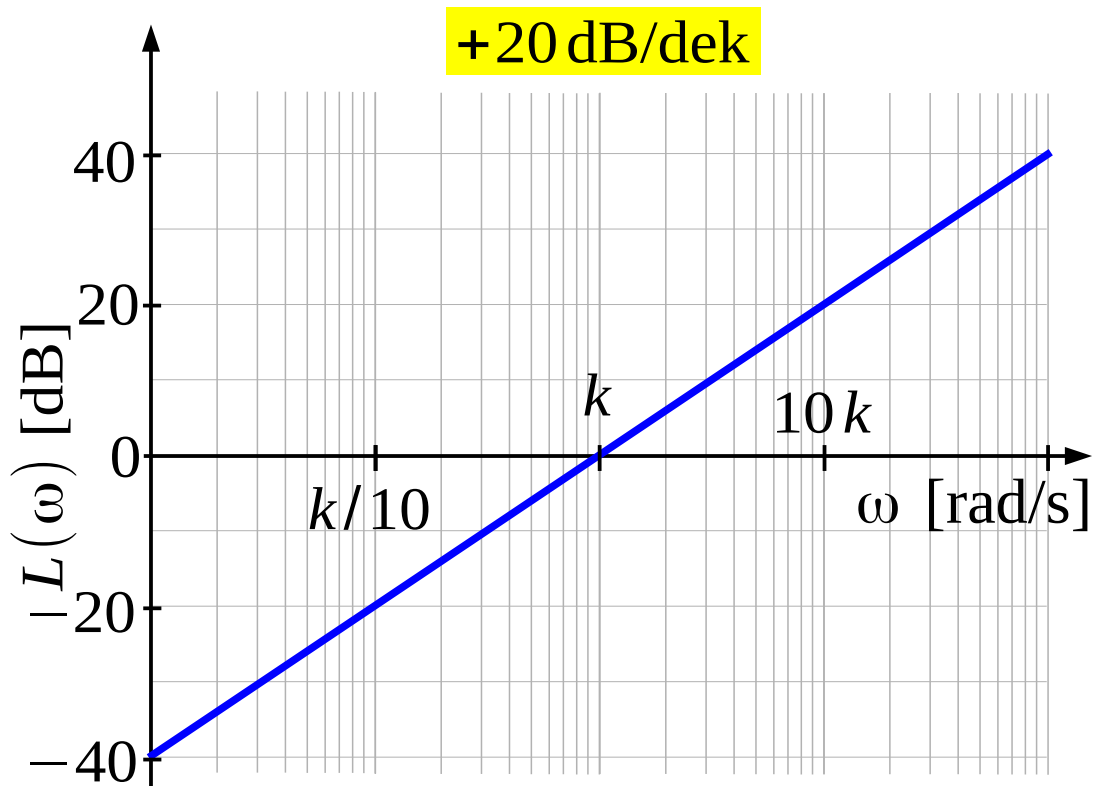
$$L(\omega) = 20 \log A(\omega) = 20 \log |k \omega| \quad \varphi(\omega) = \arctan \frac{Q}{P} = \arctan(\infty)$$



Differentiator

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k \omega|$

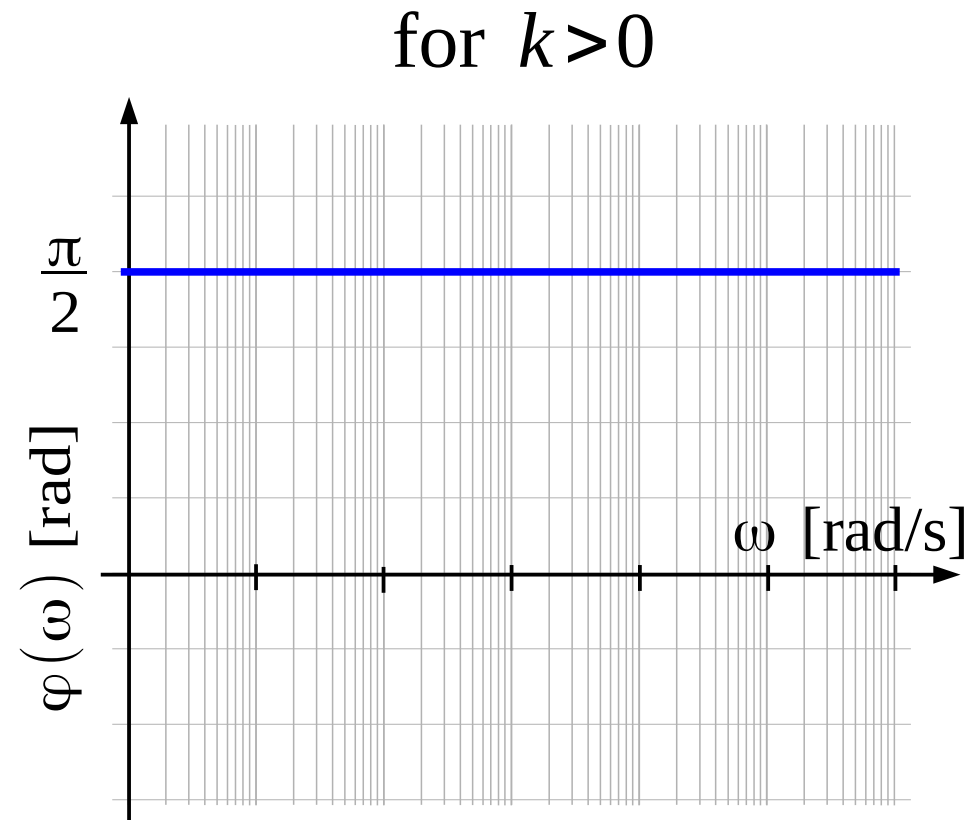
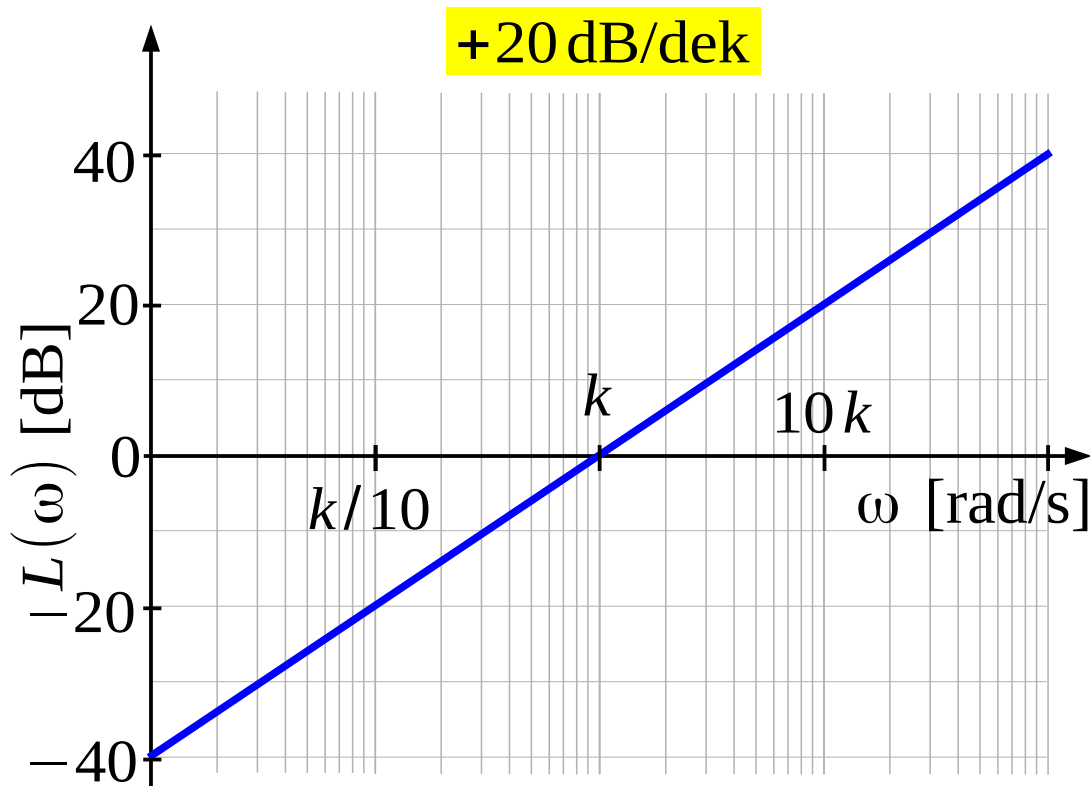
$$L(\omega) = 20 \log A(\omega) = 20 \log |k \omega| \quad \varphi(\omega) = \arctan \frac{Q}{P} = \arctan(\infty)$$



Differentiator

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k \omega|$

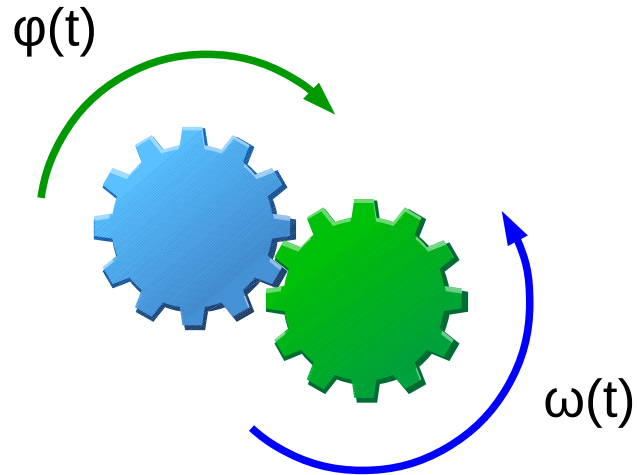
$$L(\omega) = 20 \log A(\omega) = 20 \log |k \omega| \quad \varphi(\omega) = \arctan \frac{Q}{P} = \arctan(\infty)$$



Differentiator

Examples

①



GEARBOX:

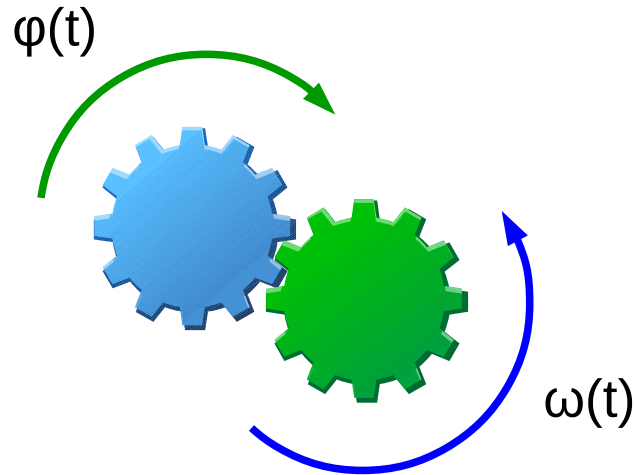
input – rotation angle $\varphi(t)$

output – angular velocity $\omega(t)$

Differentiator

Examples

①

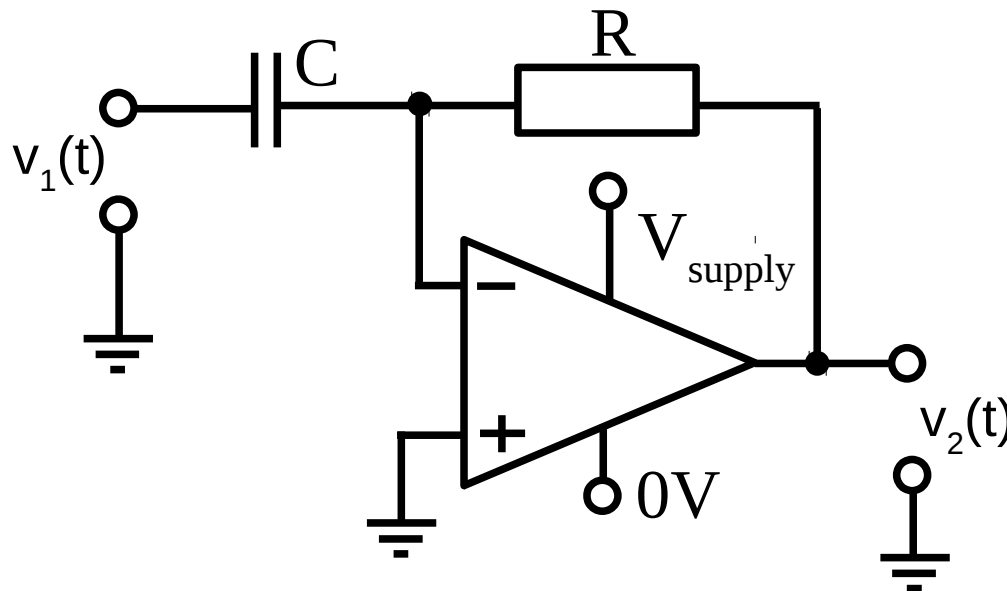


GEARBOX:

input – rotation angle $\varphi(t)$

output – angular velocity $\omega(t)$

②



OPERATIONAL AMPLIFIER:

input – voltage $v_1(t)$

output – voltage $v_2(t)$

$$v_2(t) = -RC \frac{dv_1(t)}{dt}$$

Real differentiator (derivative+1st order)

1. Element equation: $T \frac{dy(t)}{dt} + y(t) = k \frac{du(t)}{dt}$ $u(t)$ - input
 $y(t)$ - output

Real differentiator (derivative+1st order)

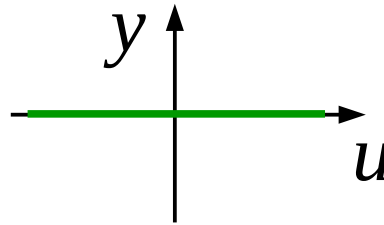
1. Element equation: $T \frac{dy(t)}{dt} + y(t) = k \frac{du(t)}{dt}$ $u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state):

Real differentiator (derivative+1st order)

1. Element equation: $T \frac{dy(t)}{dt} + y(t) = k \frac{du(t)}{dt}$ $u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state): $y = 0$ for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$

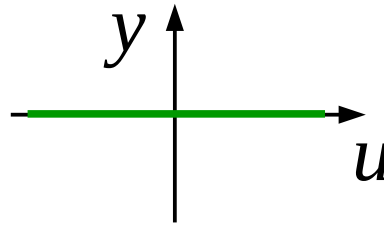


3. Transfer function:

Real differentiator (derivative+1st order)

1. Element equation: $T \frac{dy(t)}{dt} + y(t) = k \frac{du(t)}{dt}$ $u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state): $y = 0$ for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function: $H(s) = \frac{k s}{T s + 1}$

Real differentiator (derivative+1st order)

4. Step response:

$$\text{input: } u(t) = u_0 1(t)$$
$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

Real differentiator (derivative+1st order)

4. Step response:

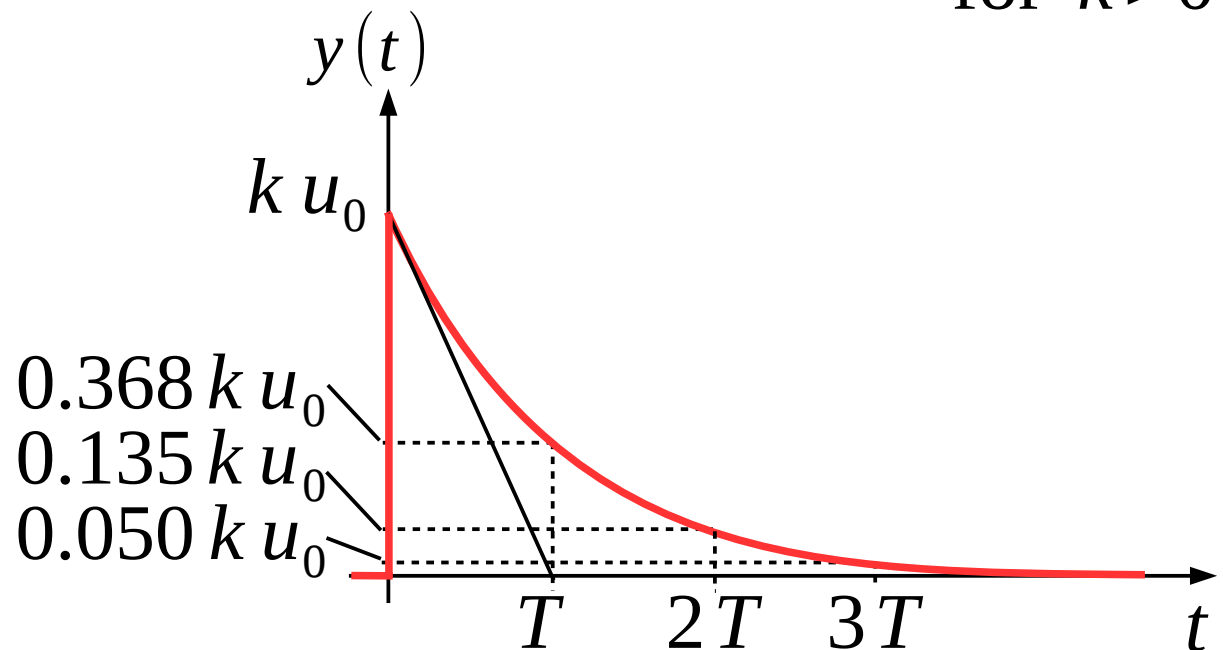
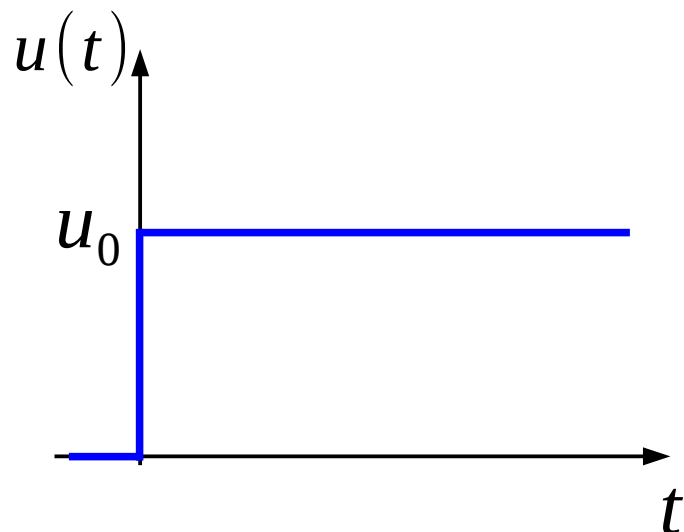
$$\text{input: } u(t) = u_0 1(t)$$

$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

$$\text{Laplace of output: } Y(s) = H(s) U(s) = \frac{k u_0}{Ts + 1}$$

$$\text{output: } y(t) = L^{-1}\{Y(s)\} = k u_0 e^{-t/T}$$

for $k > 0$



Real differentiator (derivative+1st order)

5. Frequency response:

Real differentiator (derivative+1st order)

5. Frequency response: $H(j\omega) = \frac{k j \omega}{T j \omega + 1}$

$$P(\omega) = \frac{k T \omega^2}{T^2 \omega^2 + 1}, \quad Q(\omega) = \frac{k \omega}{T^2 \omega^2 + 1}$$

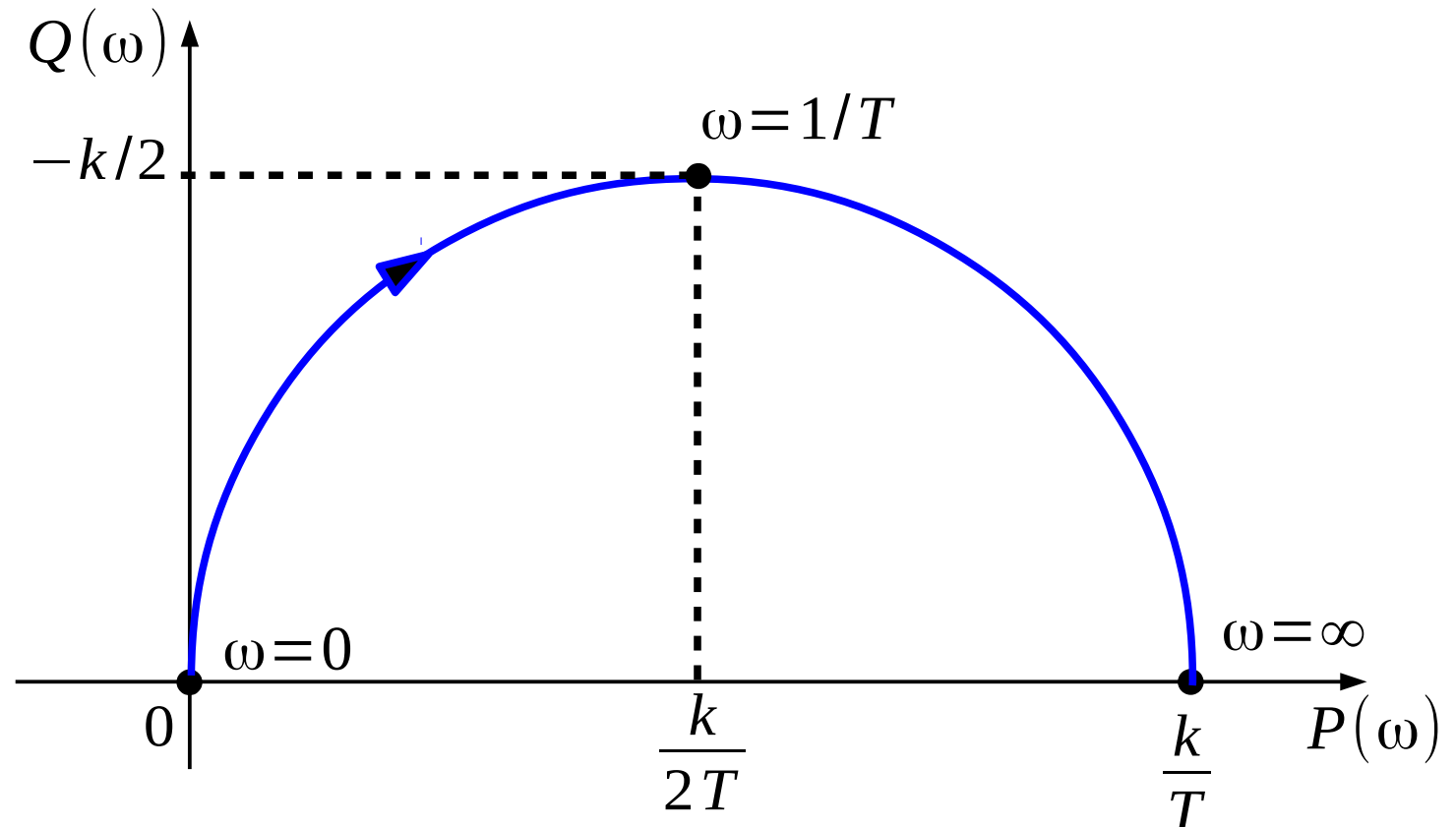
6. Nyquist plot:

Real differentiator (derivative+1st order)

5. Frequency response: $H(j\omega) = \frac{k j \omega}{T j \omega + 1}$

$$P(\omega) = \frac{k T \omega^2}{T^2 \omega^2 + 1}, \quad Q(\omega) = \frac{k \omega}{T^2 \omega^2 + 1}$$

6. Nyquist plot:
for $k > 0$



Real differentiator (derivative+1st order)

7. Bode plot:

$$P(\omega) = \frac{k T \omega^2}{T^2 \omega^2 + 1}, \quad Q(\omega) = \frac{k \omega}{T^2 \omega^2 + 1}$$

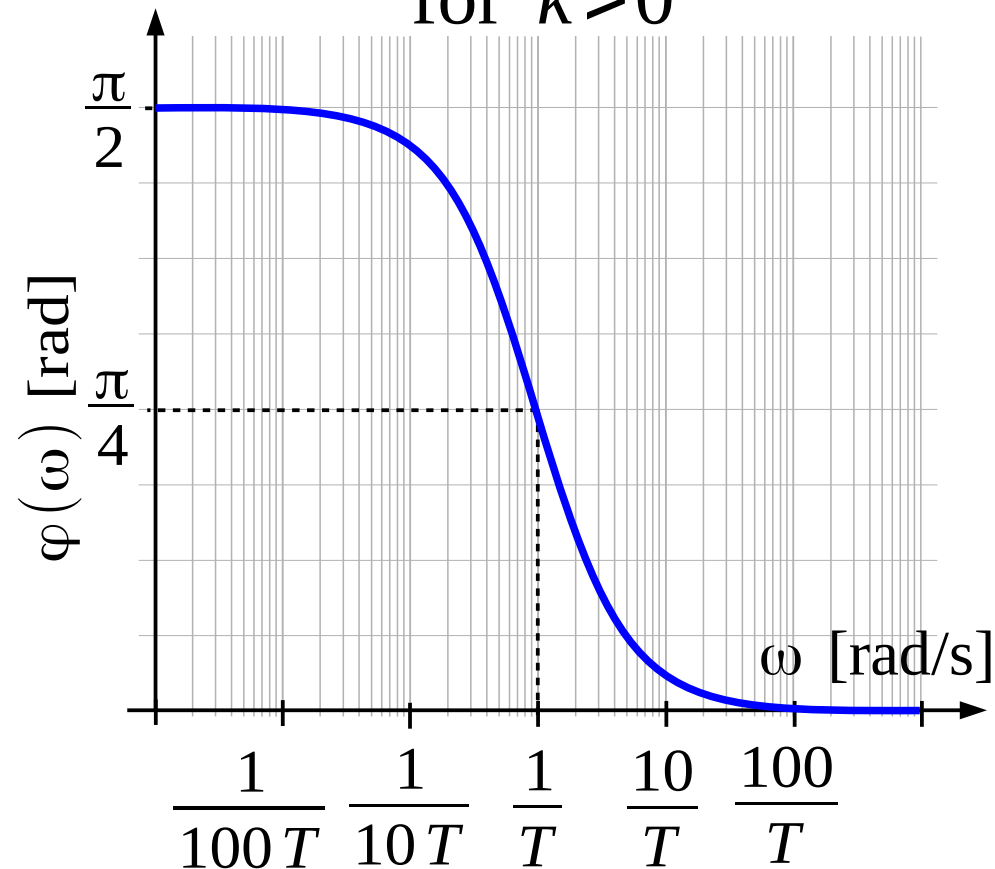
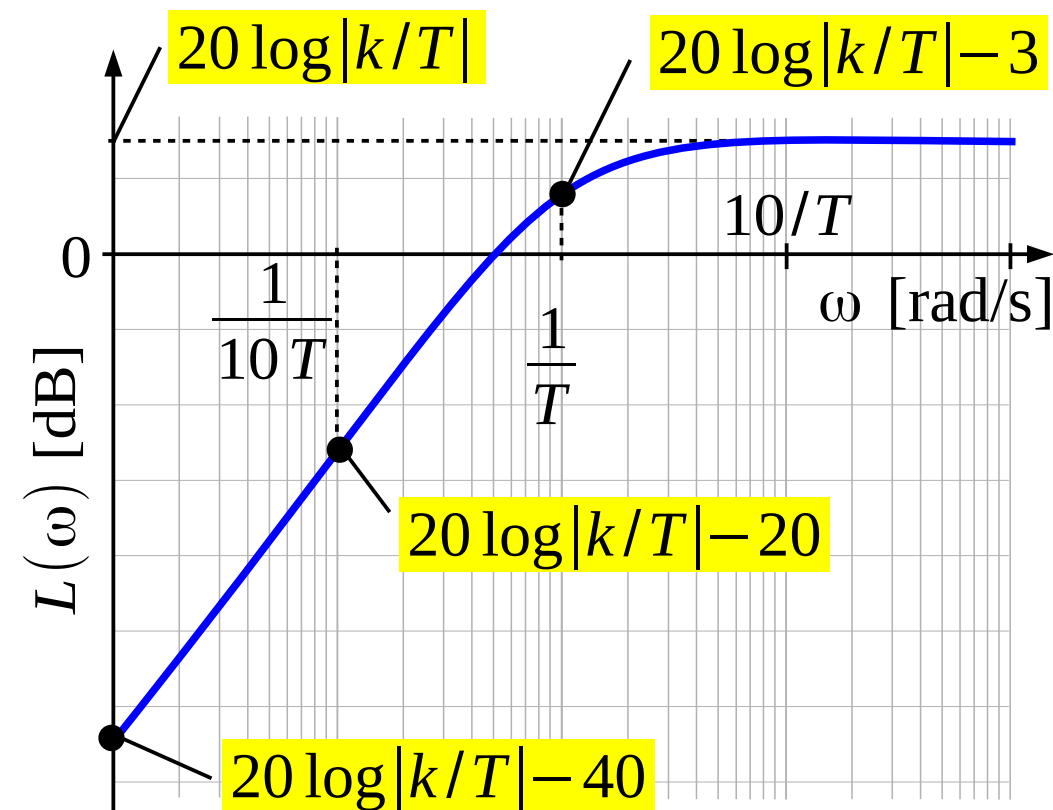
Real differentiator (derivative+1st order)

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = |k \omega| / \sqrt{T^2 \omega^2 + 1}$

$$L(\omega) = 20 \log A(\omega) = 20 \log |k \omega| - 20 \log \sqrt{T^2 \omega^2 + 1}$$

$$\varphi(\omega) = \arctan \frac{Q}{P} = \arctan \left(\frac{1}{T \omega} \right)$$

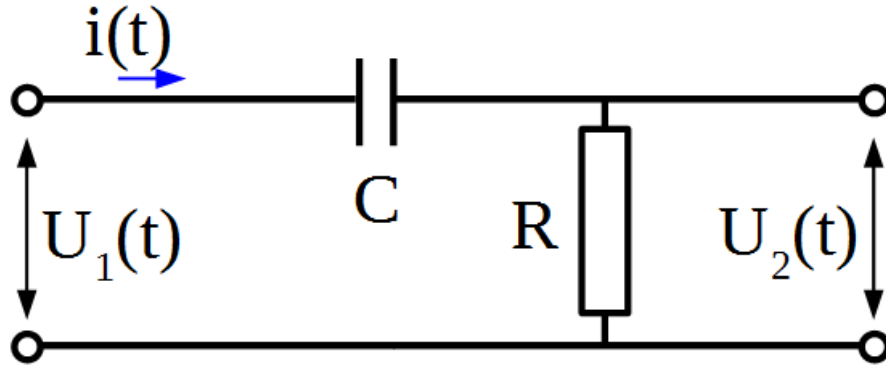
for $k > 0$



Real differentiator (derivative+1st order)

Examples

1



RC CIRCUIT:
input – voltage $u_1(t)$
output – voltage $u_2(t)$

Delay

1. Element equation: $y(t) = u(t - \tau)$

$u(t)$ - input
 $y(t)$ - output

Delay

1. Element equation: $y(t) = u(t - \tau)$

$u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state):

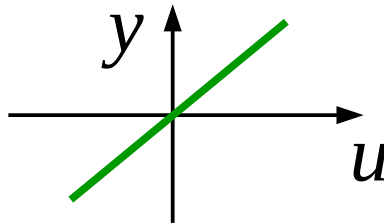
Delay

1. Element equation: $y(t) = u(t - \tau)$

$u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state): $y = u$

for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function:

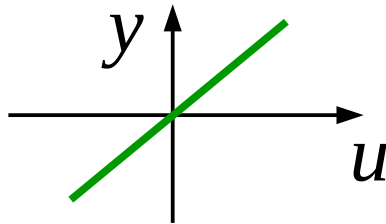
Delay

1. Element equation: $y(t) = u(t - \tau)$

$u(t)$ - input
 $y(t)$ - output

2. Static characteristic (steady state): $y = u$

for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function: $H(s) = e^{-\tau s}$

Delay

4. Step response: input: $u(t) = u_0 1(t)$
Laplace of input: $U(s) = u_0 \frac{1}{s}$

Delay

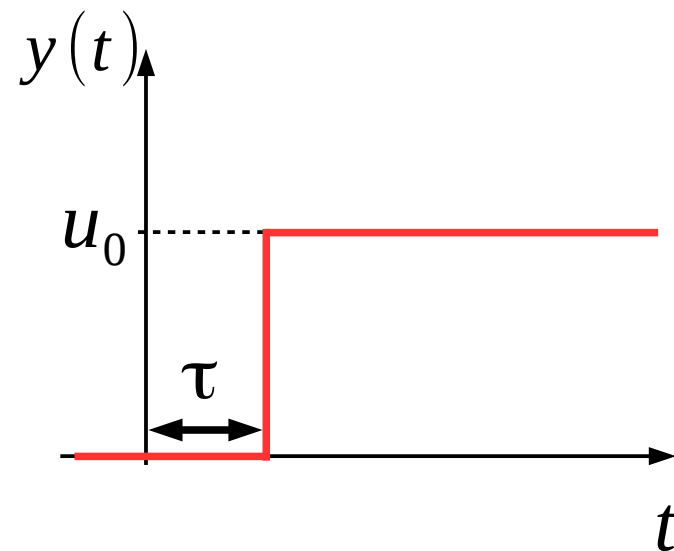
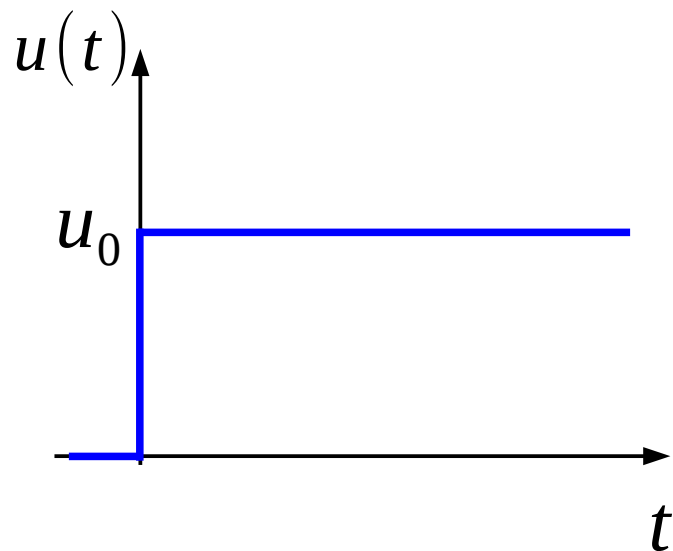
4. Step response:

$$\text{input: } u(t) = u_0 \mathbf{1}(t)$$

$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

$$\text{Laplace of output: } Y(s) = H(s)U(s) = \frac{u_0}{s} e^{-\tau s}$$

$$\text{output: } y(t) = L^{-1}\{Y(s)\} = u_0 \mathbf{1}(t - \tau)$$



Delay

5. Frequency response:

Delay

5. Frequency response: $H(j\omega) = e^{-\tau j\omega}$

$$e^{-jx} = \cos x - j \sin x$$

$$P(\omega) = \cos(\tau\omega), \quad Q(\omega) = -\sin(\tau\omega)$$

6. Nyquist plot:

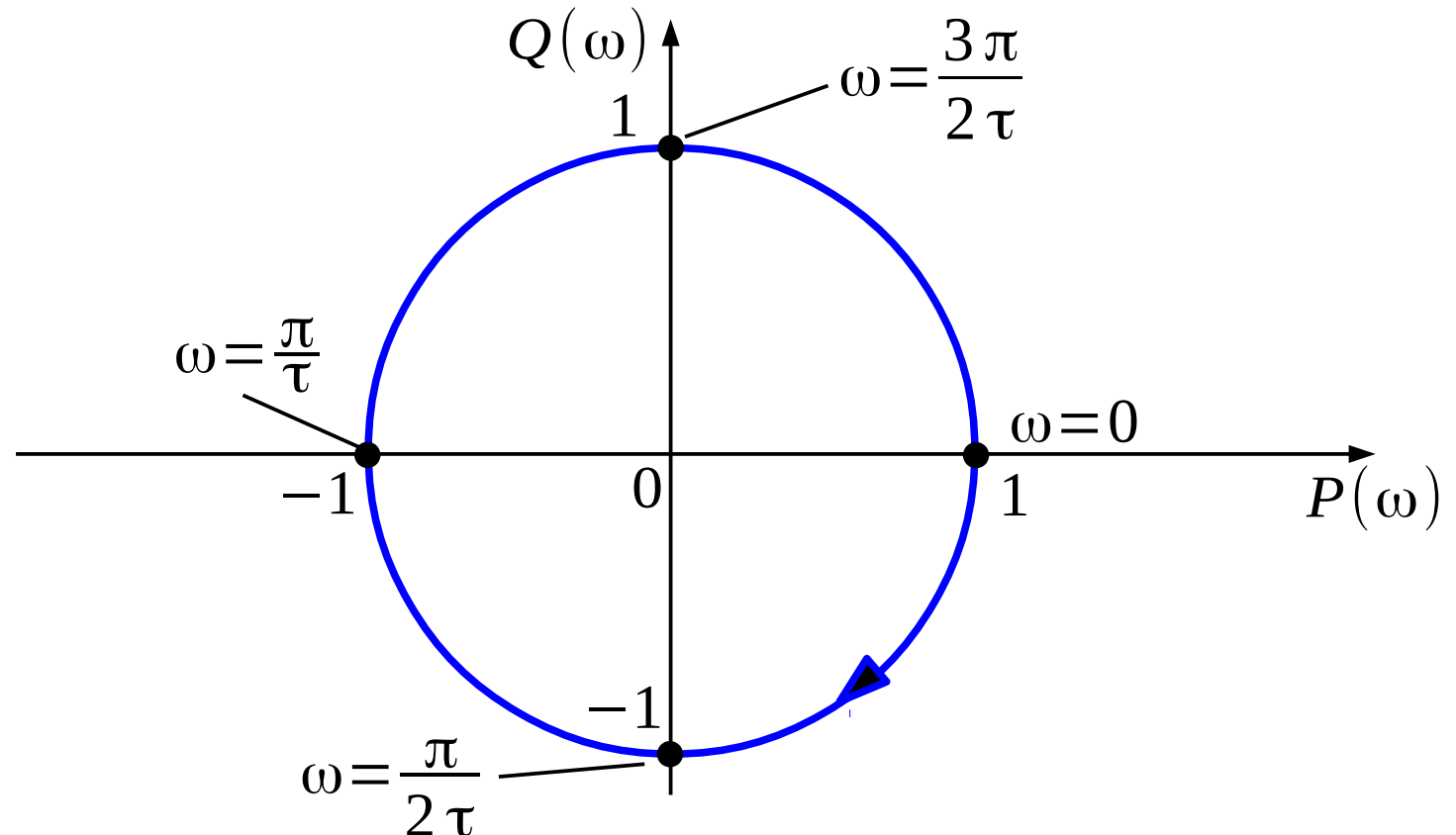
Delay

5. Frequency response: $H(j\omega) = e^{-\tau j\omega}$

$$e^{-jx} = \cos x - j \sin x$$

$$P(\omega) = \cos(\tau\omega), \quad Q(\omega) = -\sin(\tau\omega)$$

6. Nyquist plot:
for $k > 0$



Delay

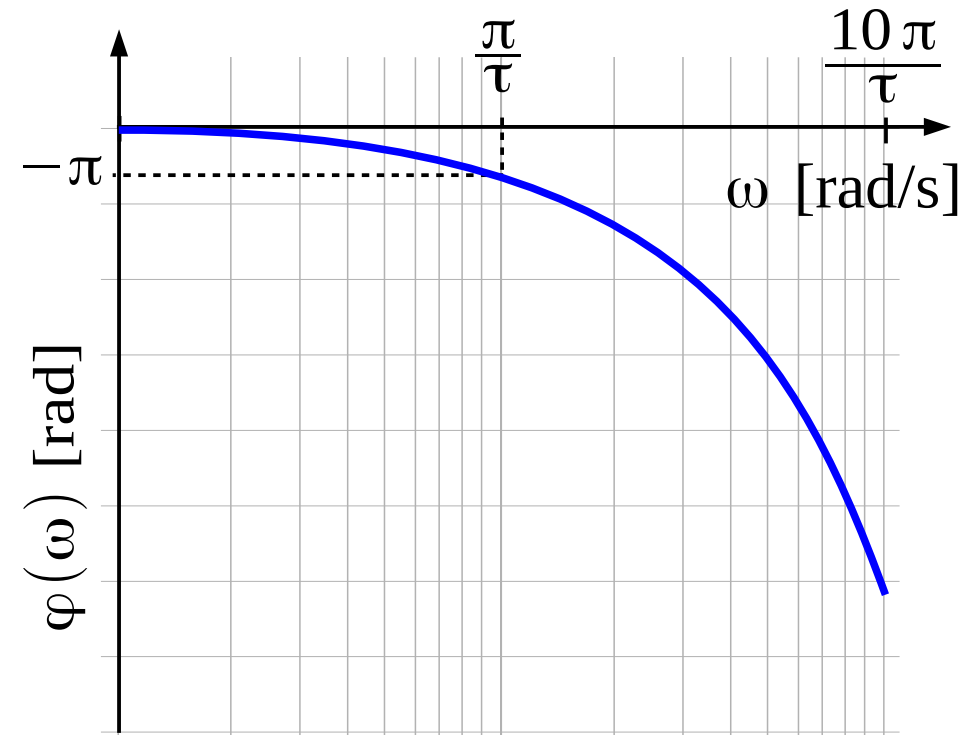
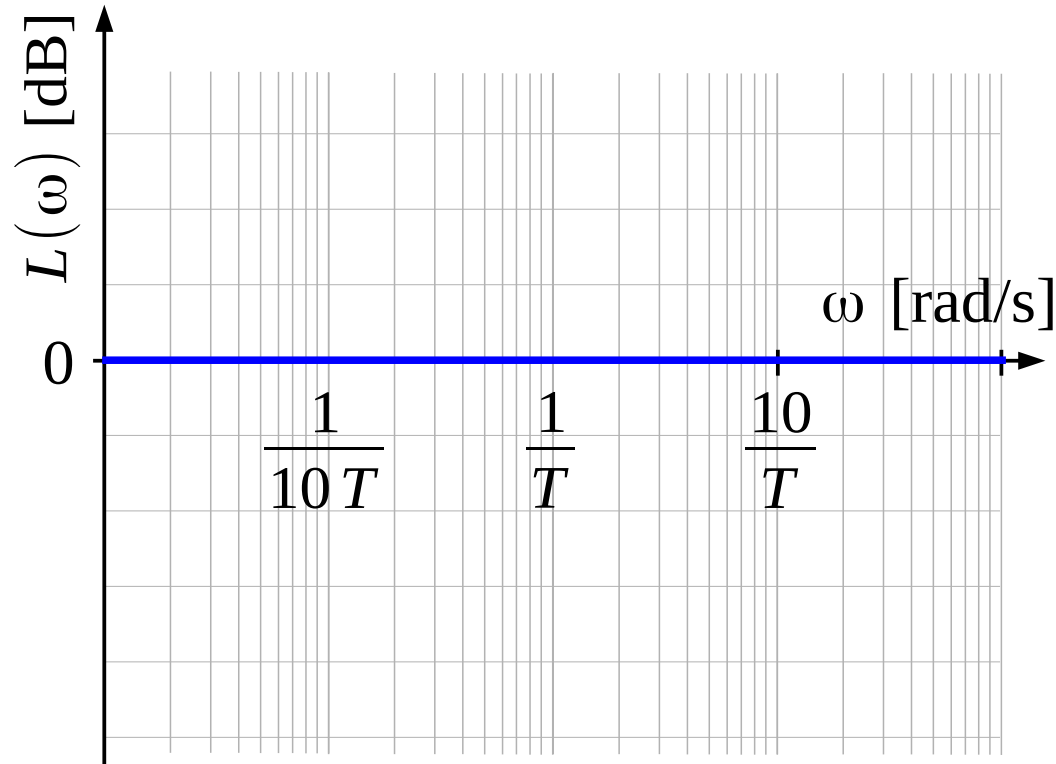
7. Bode plot: $P(\omega) = \cos(\tau \omega)$, $Q(\omega) = -\sin(\tau \omega)$

Delay

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2} = 1$

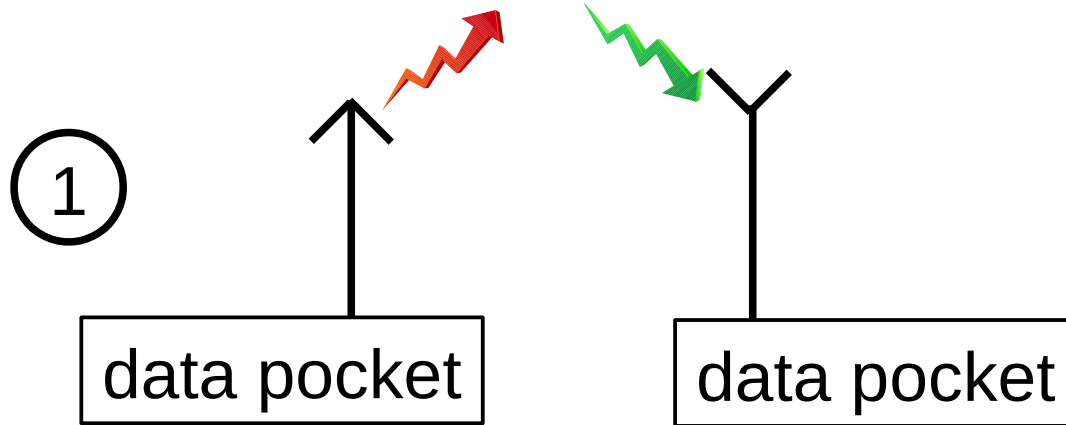
$$L(\omega) = 20 \log A(\omega) = 20 \log 1 = 0$$

$$\varphi(\omega) = \arctan \frac{Q}{P} = \arctan(-\tan(\tau\omega)) = -\tau\omega$$



Delay

Examples



WIRELESS TRANSMISSION:
input – sent data
output – received data

Second-order inertial element

1. Element equation:
$$T_1^2 \frac{d^2 y(t)}{dt^2} + T_2 \frac{dy(t)}{dt} + y(t) = k u(t)$$

Second-order inertial element

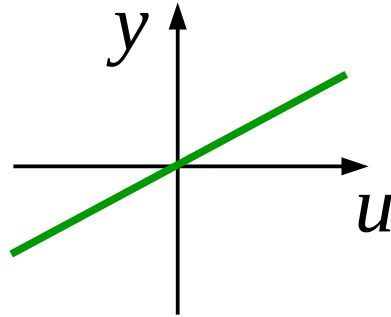
1. Element equation:
$$T_1^2 \frac{d^2 y(t)}{dt^2} + T_2 \frac{dy(t)}{dt} + y(t) = k u(t)$$

2. Static characteristic (steady state):

Second-order inertial element

1. Element equation: $T_1^2 \frac{d^2 y(t)}{dt^2} + T_2 \frac{dy(t)}{dt} + y(t) = k u(t)$

2. Static characteristic (steady state): $y = ku$ for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$

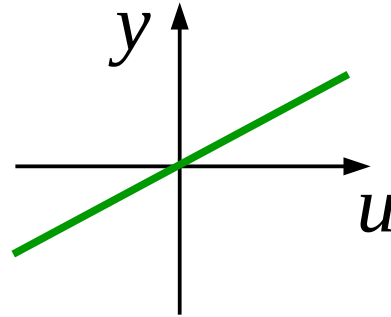


3. Transfer function:

Second-order inertial element

1. Element equation: $T_1^2 \frac{d^2 y(t)}{dt^2} + T_2 \frac{dy(t)}{dt} + y(t) = k u(t)$

2. Static characteristic (steady state): $y = ku$ for $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



3. Transfer function: $H(s) = \frac{k}{T_1^2 s^2 + T_2 s + 1}$

Second-order inertial element

4. Step response:

Second-order inertial element

4. Step response:

$$\text{input: } u(t) = u_0 \mathbf{1}(t)$$

$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

$$\text{Laplace of output: } Y(s) = H(s) U(s) = \frac{k u_0}{s(T_1^2 s^2 + T_2 s + 1)}$$

Second-order inertial element

4. Step response:

$$\text{input: } u(t) = u_0 \mathbf{1}(t)$$

$$\text{Laplace of input: } U(s) = u_0 \frac{1}{s}$$

$$\text{Laplace of output: } Y(s) = H(s) U(s) = \frac{k u_0}{s(T_1^2 s^2 + T_2 s + 1)}$$

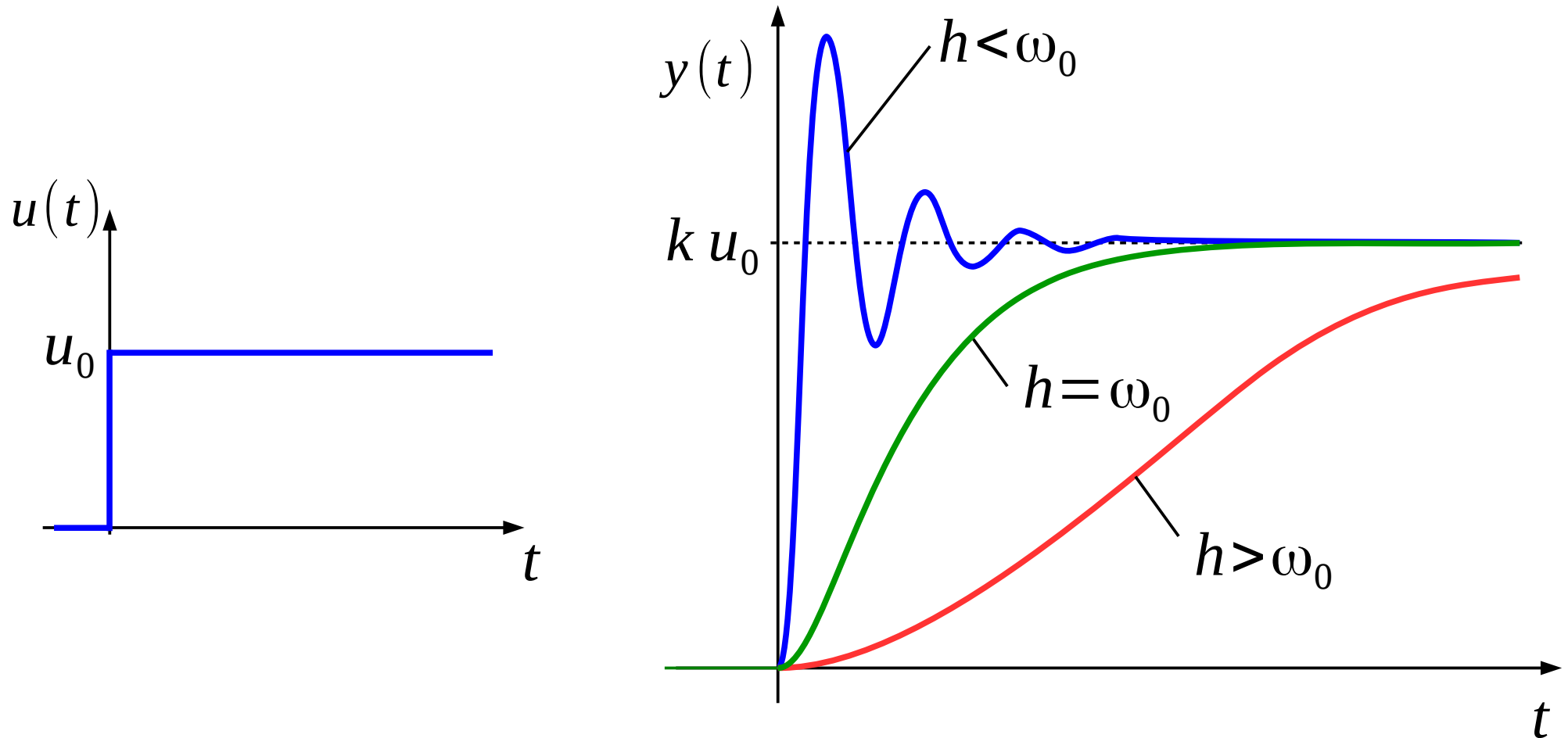
$$\text{output: } y(t) = L^{-1}\{Y(s)\} =$$

$$= \begin{cases} k u_0 \omega_0^2 \left(1 - e^{-ht} \left(\cos \omega t + \frac{h}{\omega} \sin \omega t \right) \right), & \text{for } h \leq \omega_0 \\ k u_0 \omega_0^2 \left(1 + e^{-ht} \left(\left(\frac{h+w}{2w} - 1 \right) e^{-wt} - \frac{h+w}{2w} e^{wt} \right) \right), & \text{for } h \geq \omega_0 \end{cases}$$

$$\text{where: } h = \frac{T_2}{2T_1^2}, \quad \omega_0 = \frac{1}{T_1}, \quad \omega = \sqrt{\omega_0^2 - h^2}, \quad w = \sqrt{h^2 - \omega_0^2}$$

Second-order inertial element

4. Step response:



Second-order inertial element

5. Frequency response:

Second-order inertial element

5. Frequency response: $H(j\omega) = \frac{k}{-T_1^2 \omega^2 + T_2 j\omega + 1}$

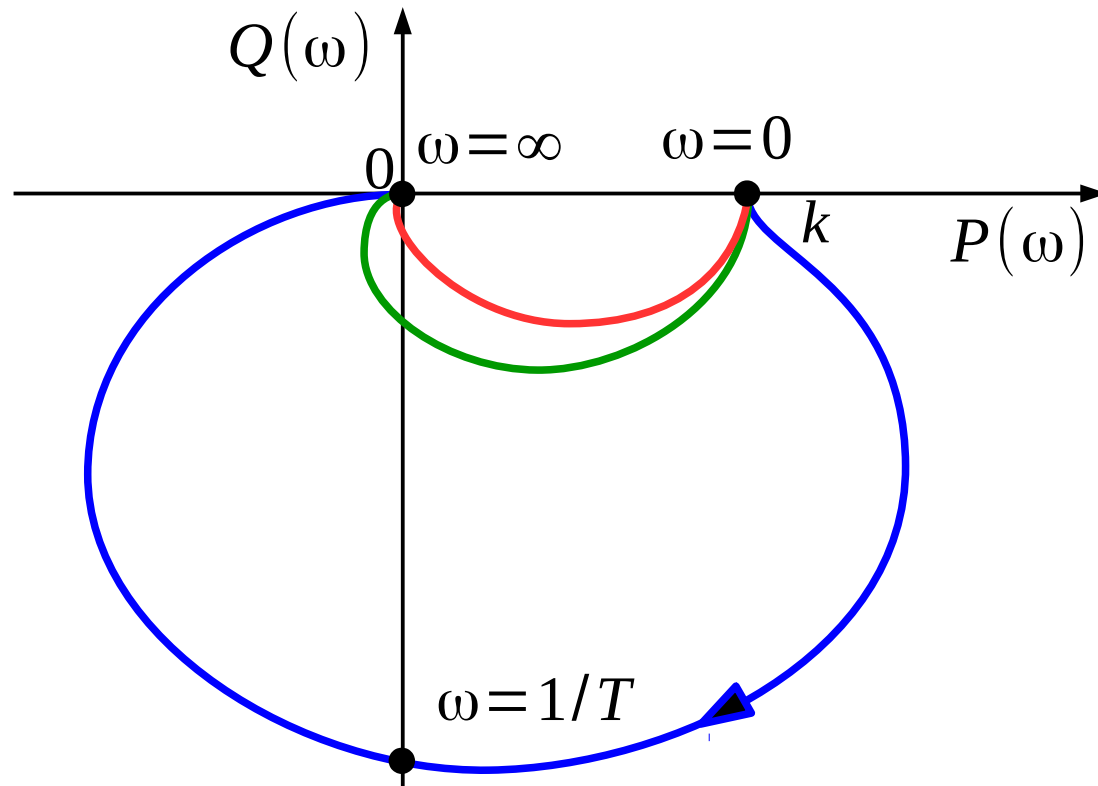
$$P(\omega) = \frac{k(1 - T_1^2 \omega^2)}{(1 - T_1^2 \omega^2)^2 + T_2^2 \omega^2}, \quad Q(\omega) = \frac{-k T_2 \omega}{(1 - T_1^2 \omega^2)^2 + T_2^2 \omega^2}$$

Second-order inertial element

5. Frequency response:
$$H(j\omega) = \frac{k}{-T_1^2 \omega^2 + T_2 j\omega + 1}$$

$$P(\omega) = \frac{k(1 - T_1^2 \omega^2)}{(1 - T_1^2 \omega^2)^2 + T_2^2 \omega^2}, \quad Q(\omega) = \frac{-k T_2 \omega}{(1 - T_1^2 \omega^2)^2 + T_2^2 \omega^2}$$

6. Nyquist plot:
for $k > 0$



- for $h < \omega_0$
- for $h = \omega_0$
- for $h > \omega_0$

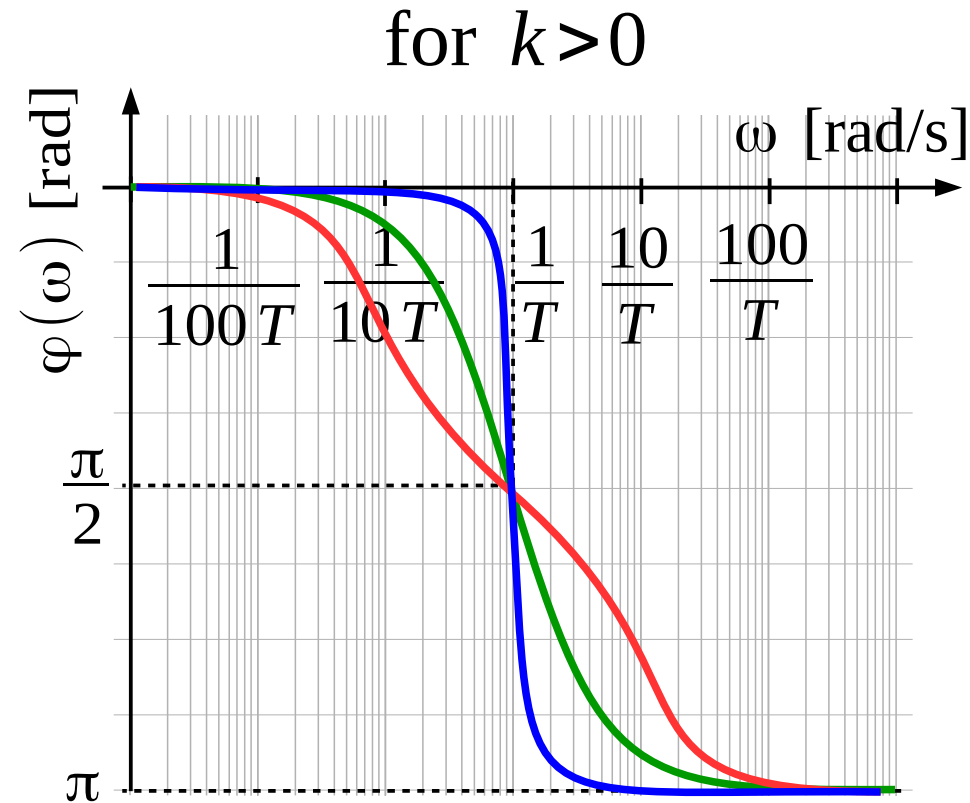
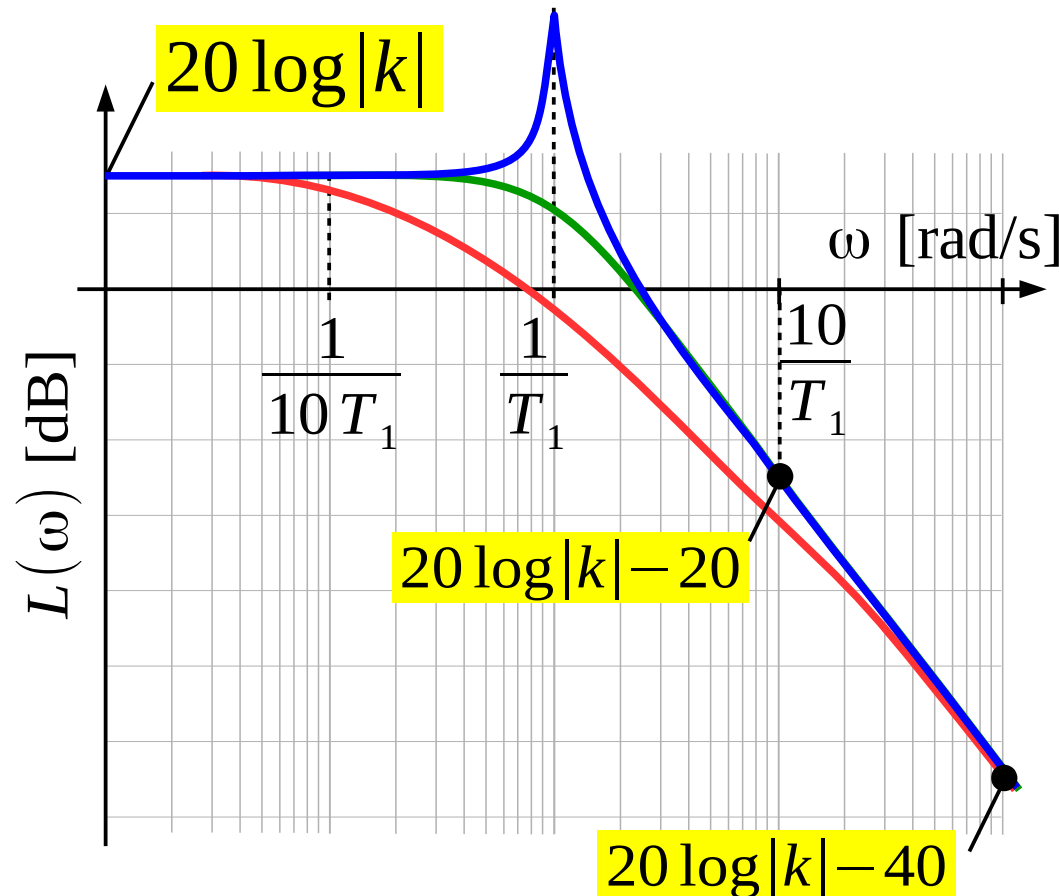
Second-order inertial element

7. Bode plot: $A(\omega) = \sqrt{P^2 + Q^2}$

$L(\omega) = 20 \log A(\omega)$

$\varphi(\omega) = \arctan \frac{Q}{P}$

- for $h < \omega_0$
- for $h = \omega_0$
- for $h > \omega_0$



Second-order inertial element

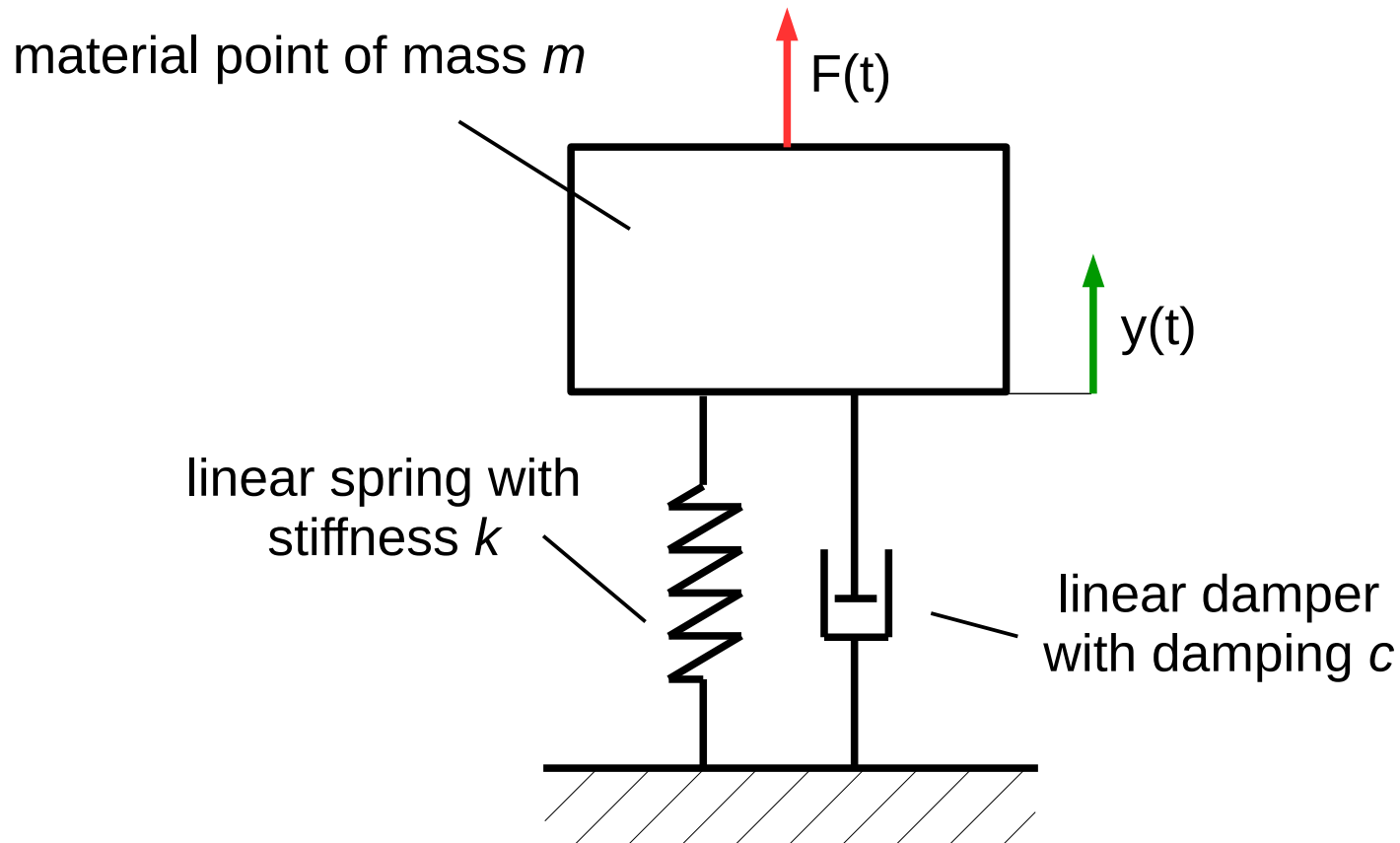
Examples

①

VIBRATING SYSTEM:

input – force $F(t)$

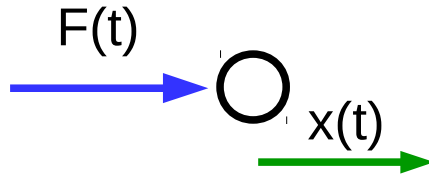
output – displacement $y(t)$



Second-order inertial element

Examples

②



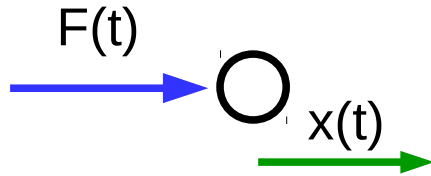
LINEAR MOTION OF A MATERIAL POINT WITH LINEAR DAMPING:
input – force $F(t)$
output – displacement $x(t)$

example: car driving on a flat surface with air resistance proportional to velocity, described using machine equation of motion, with assumption of constant reduced mass.

Second-order inertial element

Examples

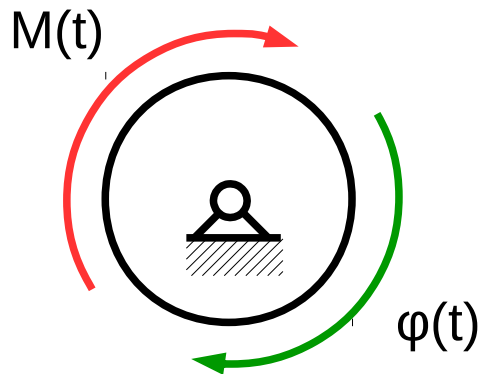
②



LINEAR MOTION OF A MATERIAL POINT WITH LINEAR DAMPING:
input – force $F(t)$
output – displacement $x(t)$

example: car driving on a flat surface with air resistance proportional to velocity, described using machine equation of motion, with assumption of constant reduced mass.

③

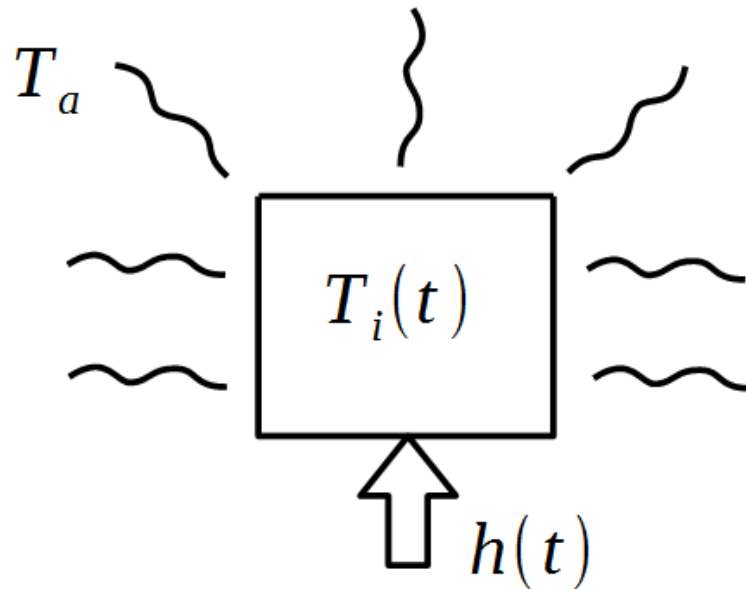


ANGULAR MOTION OF A RIGID BODY WITH LINEAR DAMPING:
input – torque $M(t)$
output – angle $\varphi(t)$

Second-order inertial element

Examples

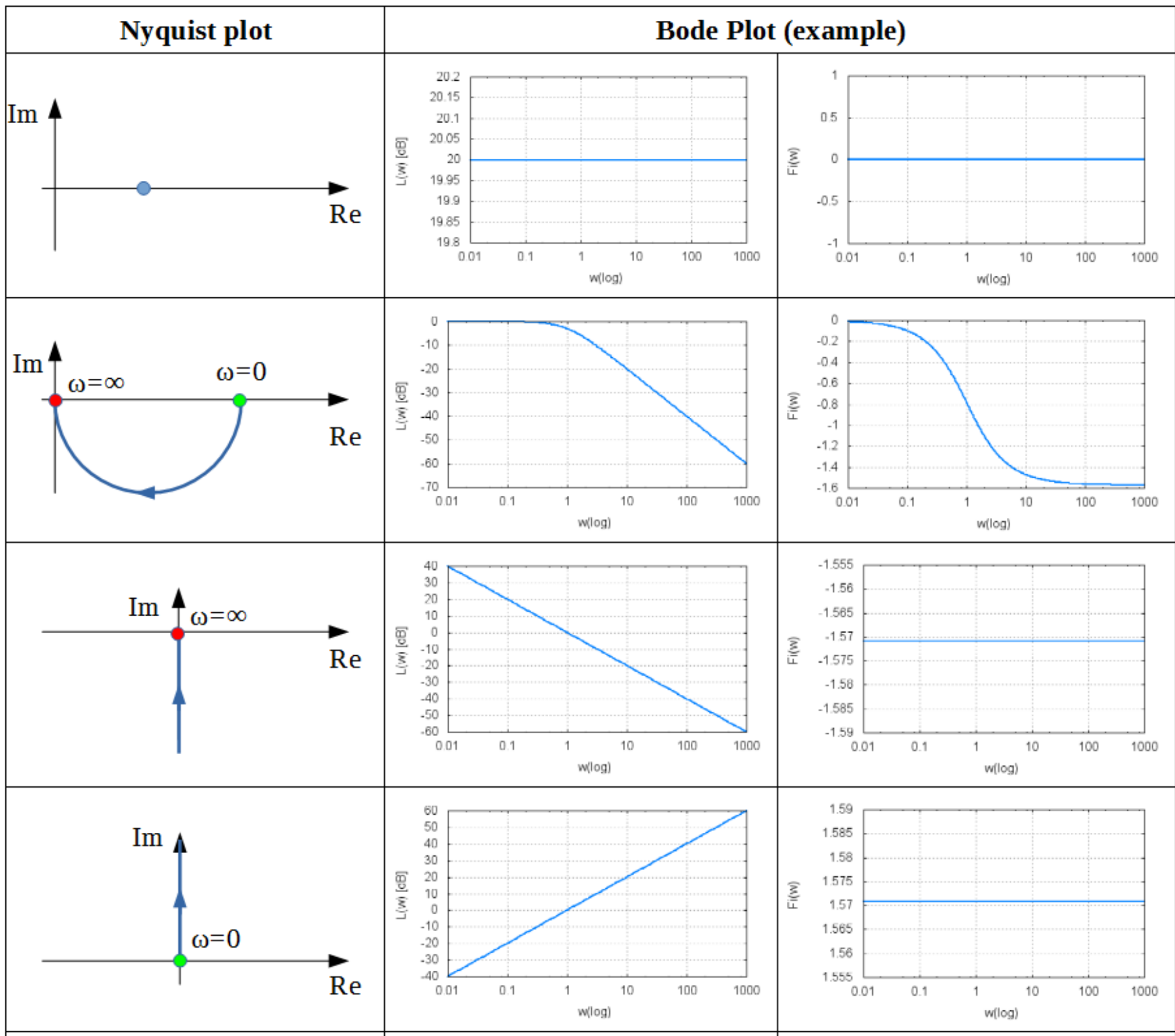
4

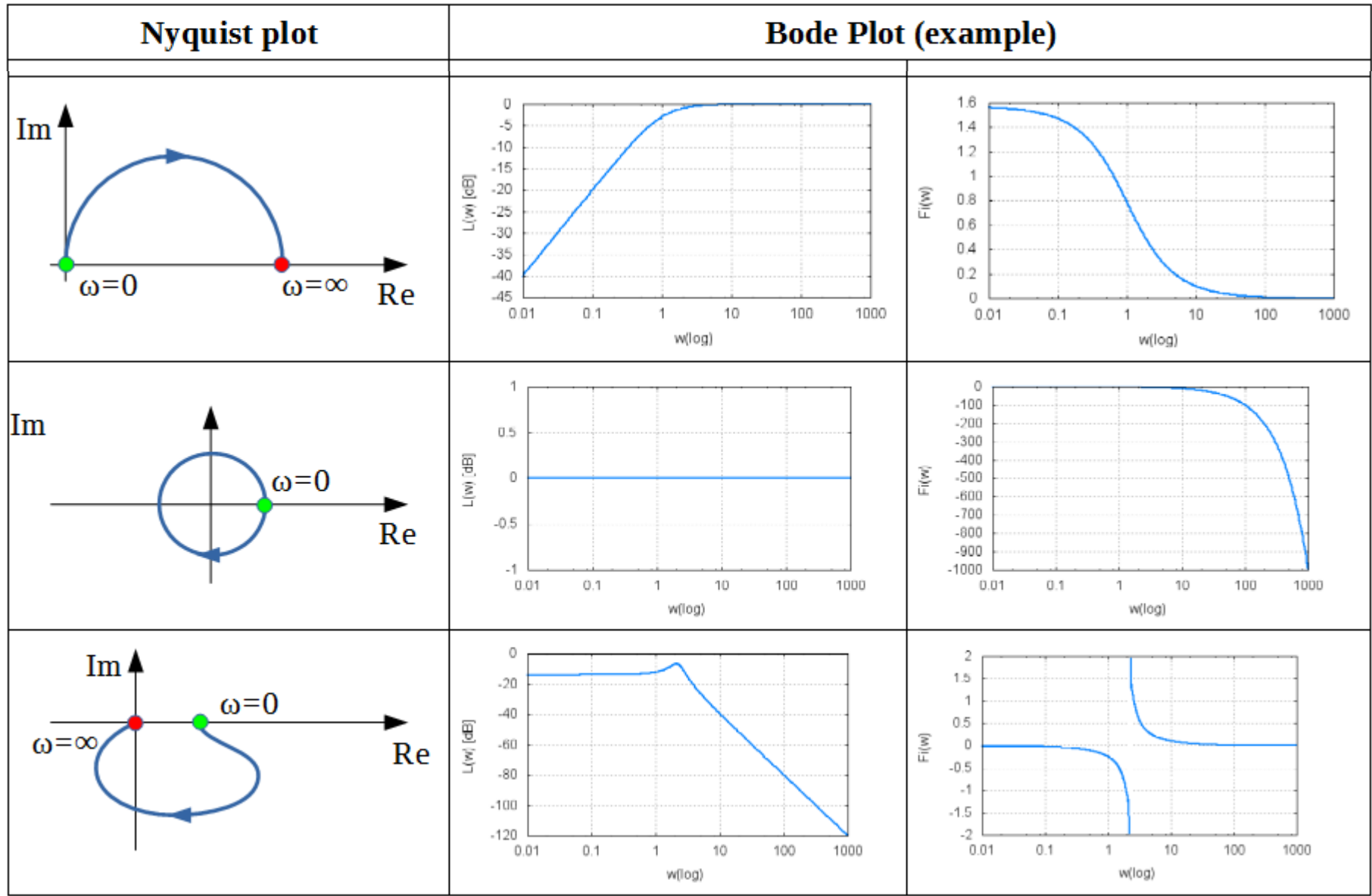


HEATED OBJECT WITH HIGH
INERTIA:
input – heater power $h(t)$
output – object temperature $T_i(t)$

Classification of basic automatic systems

Element name	Transfer function
proportional	k
first order (inertial)	$\frac{k}{Ts + 1}$
integrator	$\frac{k}{s}$
differentiator	ks
differentiator with inertia	$\frac{ks}{Ts + 1}$
delay	$e^{-\tau s}$
second order (oscillator)	$\frac{k}{T_1^2 s^2 + T_2 s + 1}$





Example

Calculate and sketch step response and Bode Plots for a system with transfer function $H(s) = 3/s$.

Example

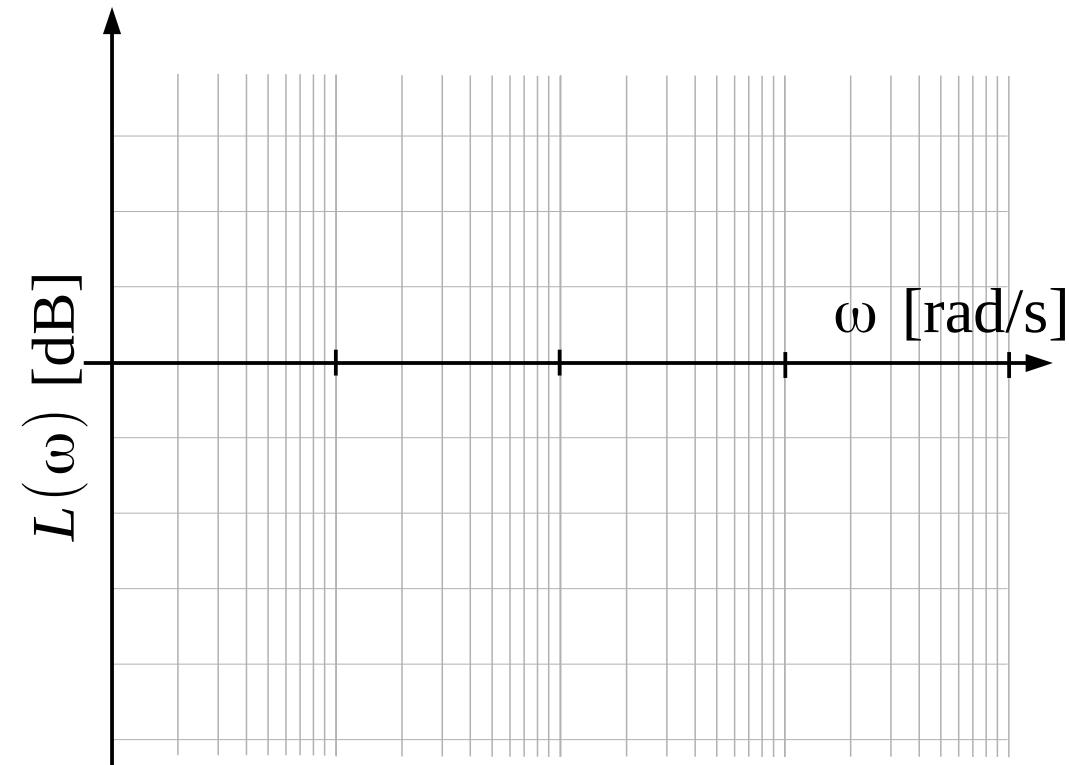
Calculate and sketch step response and Bode Plots for a system with transfer function $H(s) = 3/s$.

Example

Calculate and sketch step response and Bode Plots for a system with transfer function $H(s) = 3/s$.

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Calculate and sketch step response and Bode Plots for a system with transfer function $H(s) = 3/s$.



Example

Calculate and sketch step response and Bode Plots for a system with transfer function $H(s) = 3/s$.

