



# Faculty of Automotive and Construction Machinery Engineering

WARSAW UNIVERSITY OF TECHNOLOGY

## ***Theory of Machines and Automatic Control*** Winter 2019/2020

**Lecturer: Sebastian Korczak, PhD Eng.**

# Lecture 9

Frequency response.  
Classification of basic automatic systems.

# Transfer function

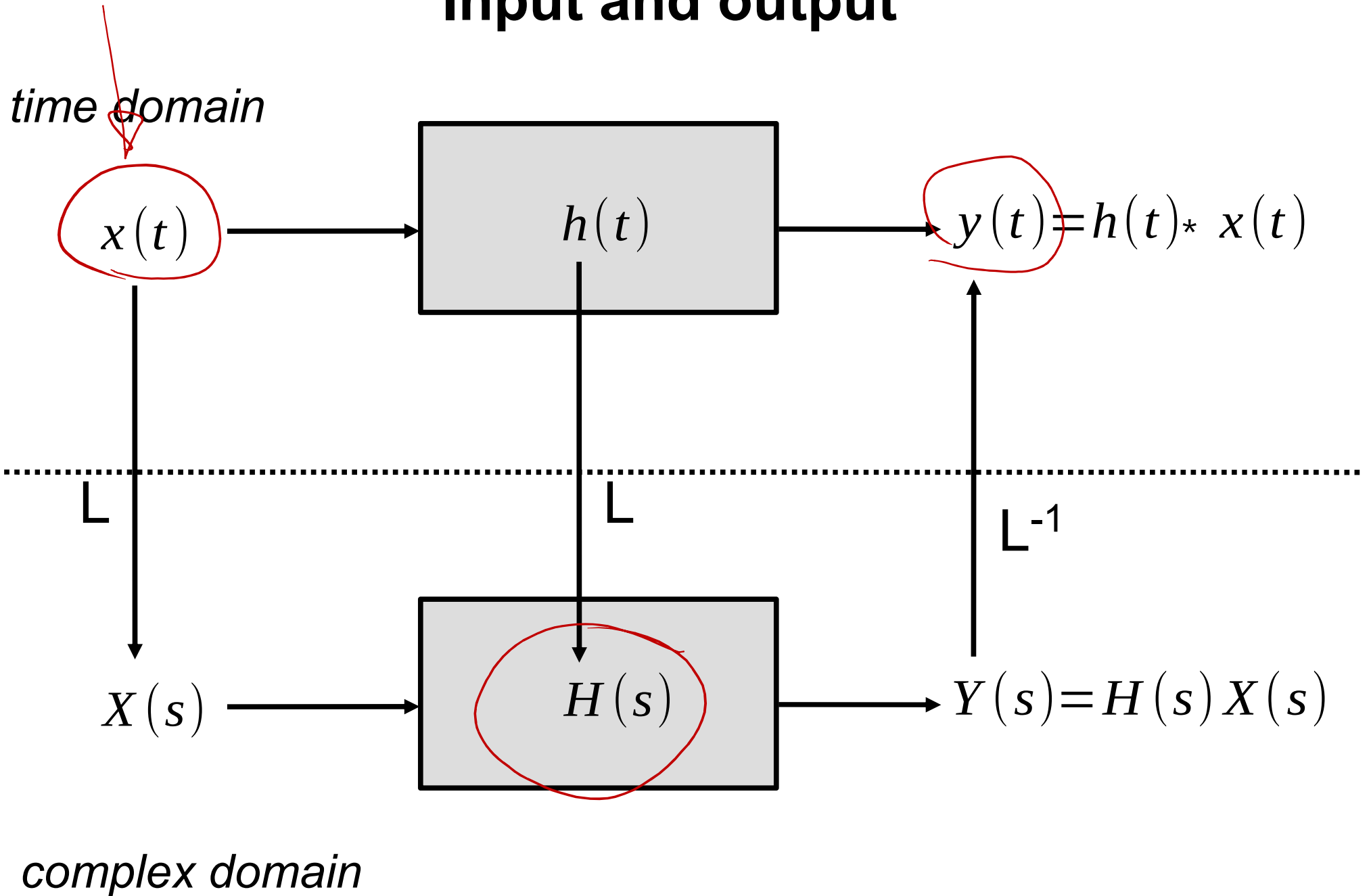
Linear time-invariant SISO system for continuous-time input signal  $x(t)$  and output  $y(t)$  in a form

Transfer function  $H(s) = \frac{Y(s)}{X(s)}$

$Y(s)$  - Laplace'a transform of an output

$X(s)$  - Laplace'a transform of an input

# Input and output



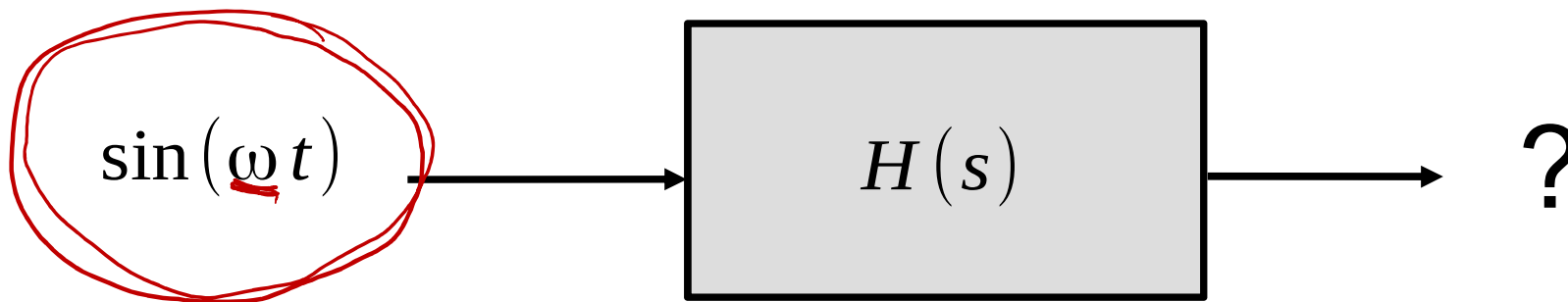
# Transfer function & frequency response

Transfer function  
(Laplace domain)

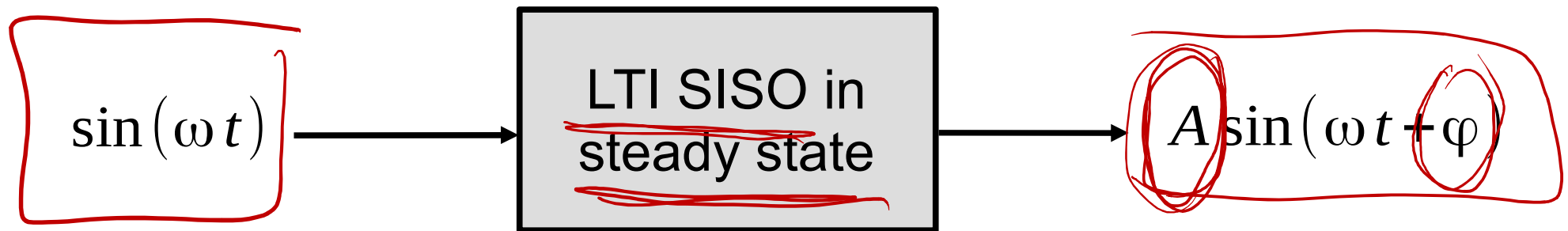
$$H(s)$$

Full system description  
(for every possible input)

# Transfer function & frequency response



# Transfer function & frequency response



# Transfer function & frequency response

Transfer function  
(Laplace domain)

$$H(s)$$

Full system description  
(for every possible input)

$\sigma = 0$

$$s = j\omega$$

Frequency response  
(Fourier domain)

$$H(j\omega)$$

Description of a system in  
steady state with harmonic  
input

# Transfer function – frequency response

input:

$$x(t) = \sin(\omega t)$$

transfer function:

$$H(s)$$

output:

$$y(t) = A \sin(\omega t + \varphi)$$



# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$     transfer function:  $H(s)$     output:  $y(t) = A \sin(\omega t + \varphi)$

$$H(s) \xrightarrow{s = j\omega} H(j\omega) = P(\omega) + jQ(\omega)$$

# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

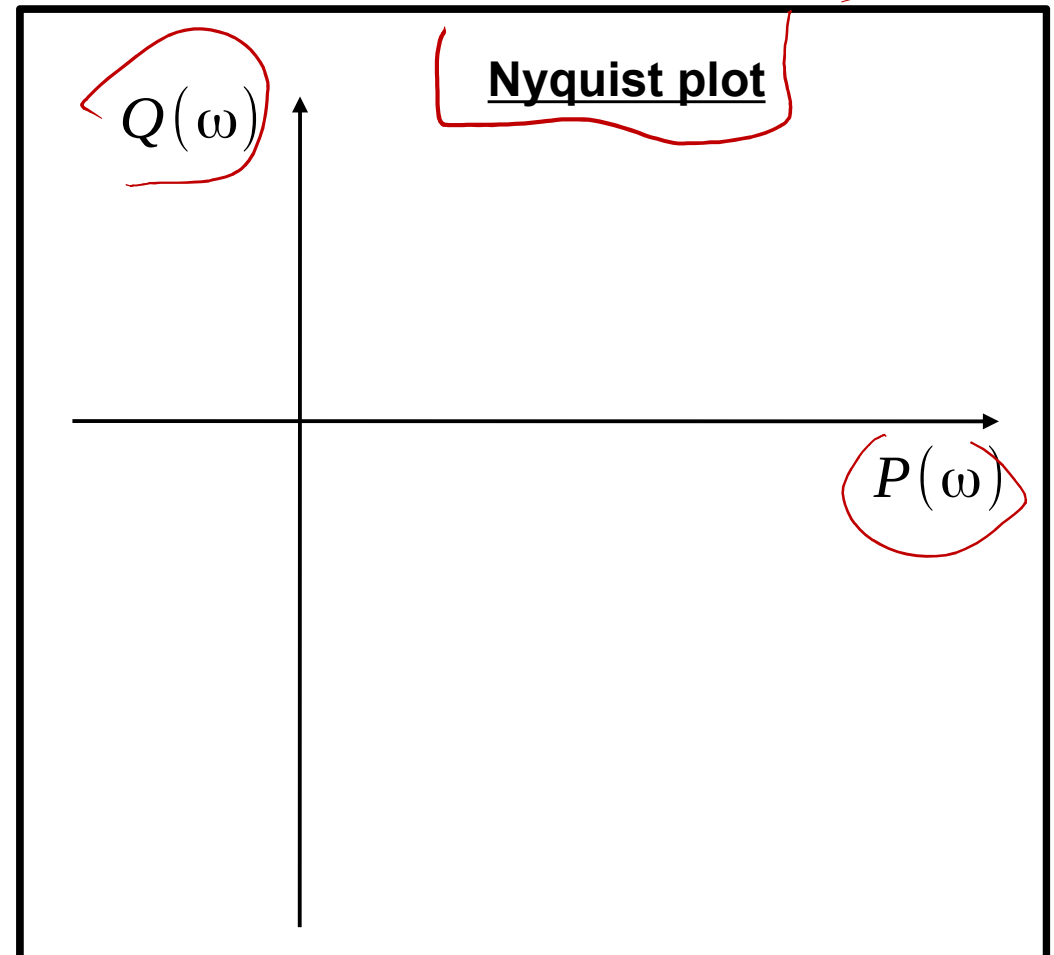
$$H(s) \xrightarrow{s=j\omega} H(j\omega) = P(\omega) + jQ(\omega)$$

# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

$$H(s) \xrightarrow{s=j\omega} H(j\omega) = P(\omega) + jQ(\omega)$$

$\omega \in (0; \infty)$



# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

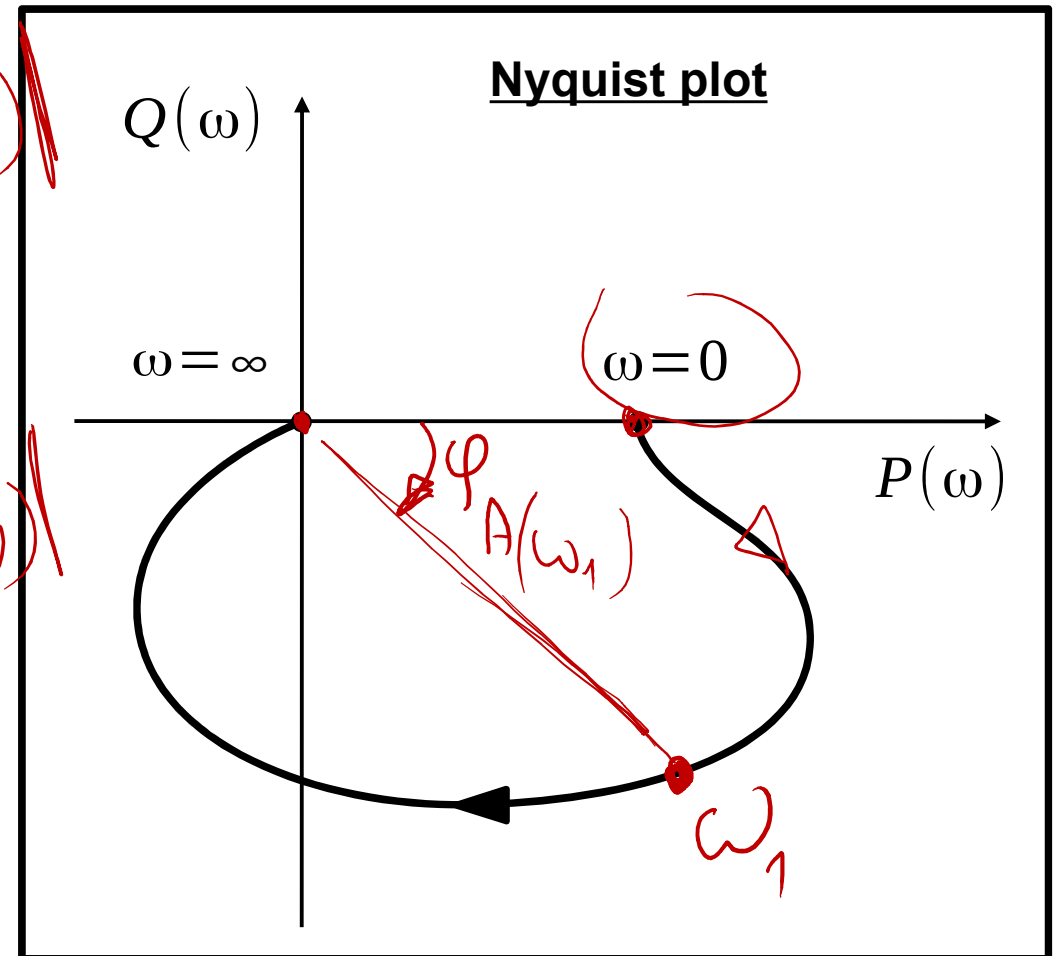
$$H(s) \xrightarrow{s=j\omega} \underline{H(j\omega)} = P(\omega) + jQ(\omega)$$

$$A(\omega) = \sqrt{P^2 + Q^2} = |H(j\omega)|$$

GAIN

$$\varphi(\omega) = \arctan \frac{Q}{P} = \text{Ang } H(j\omega)$$

delay

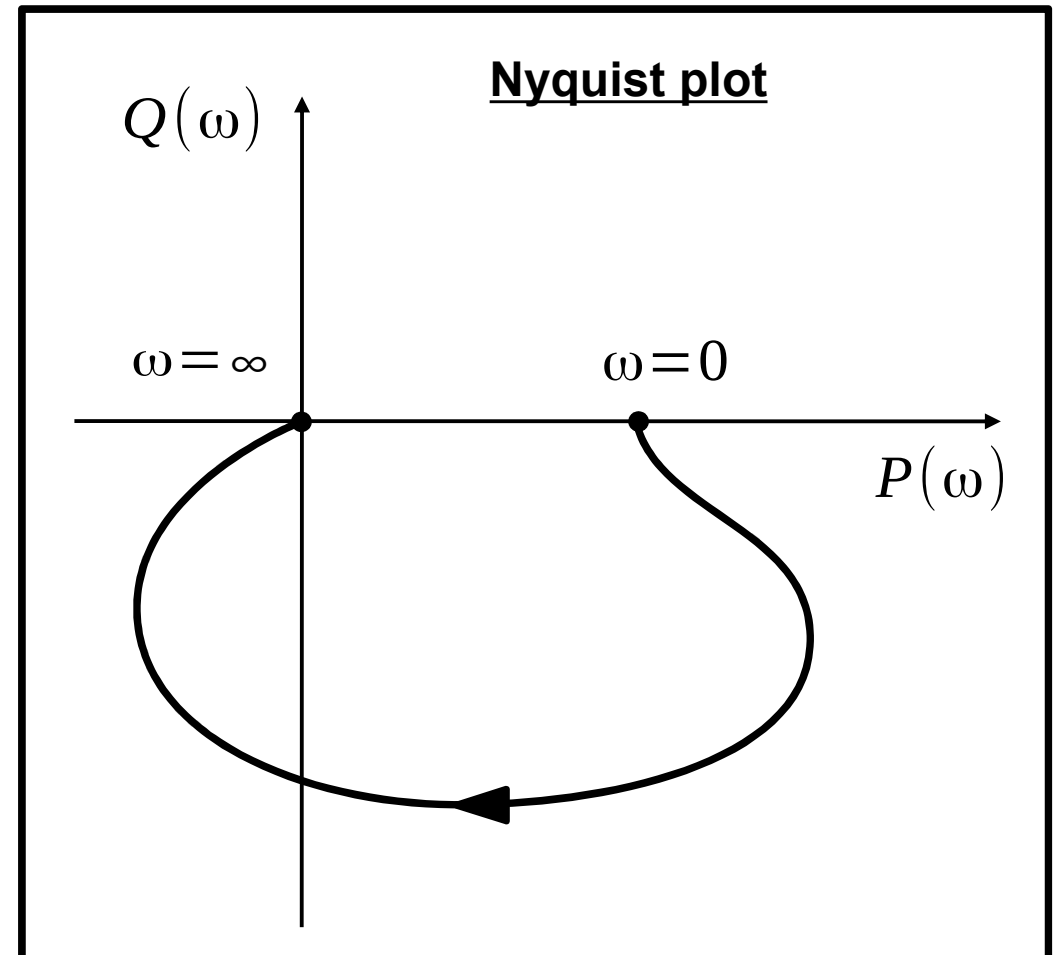


# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

$$H(s) \xrightarrow{s=j\omega} H(j\omega) = P(\omega) + jQ(\omega)$$

$$A(\omega) = |H(j\omega)| = \sqrt{P^2(\omega) + Q^2(\omega)}$$



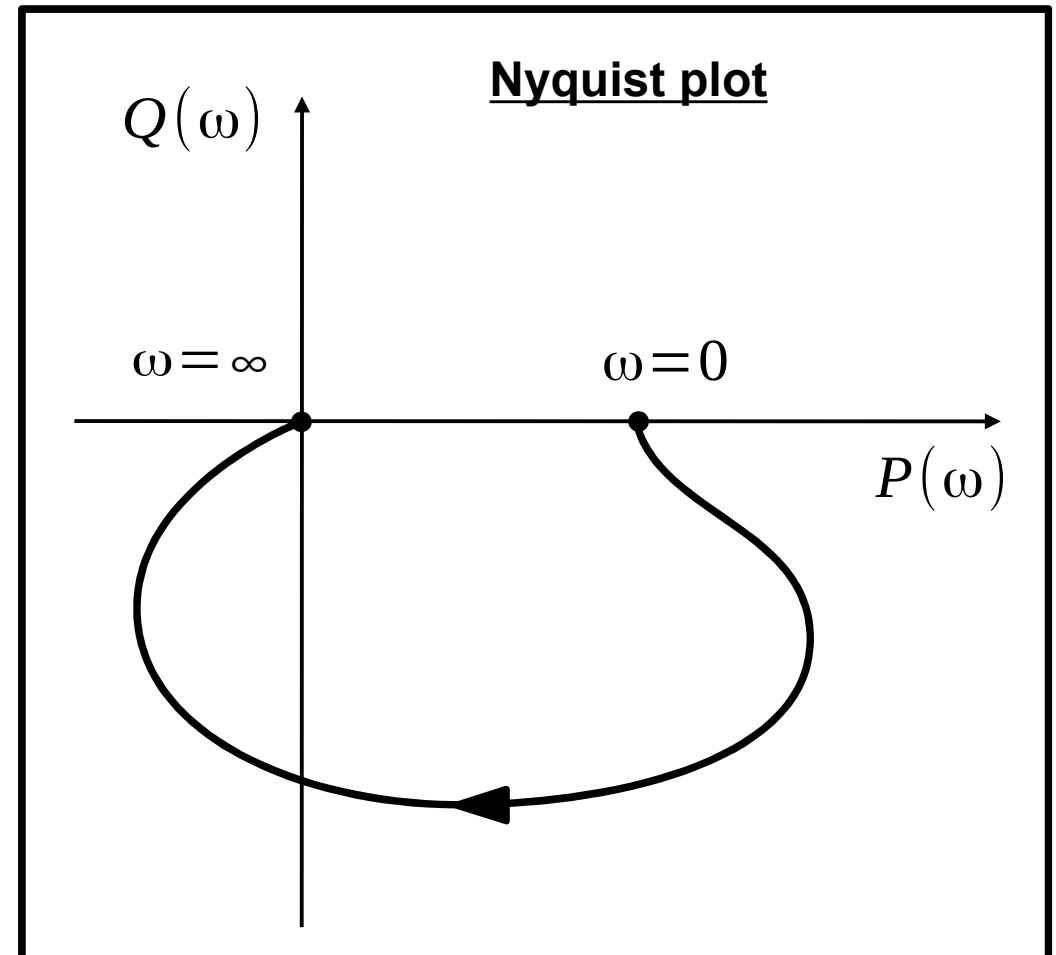
# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

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$$A(\omega) = |H(j\omega)| = \sqrt{P^2(\omega) + Q^2(\omega)}$$

$$\varphi(\omega) = \text{Arg } H(j\omega) = \arctan \frac{Q}{P}$$



# Transfer function – frequency response

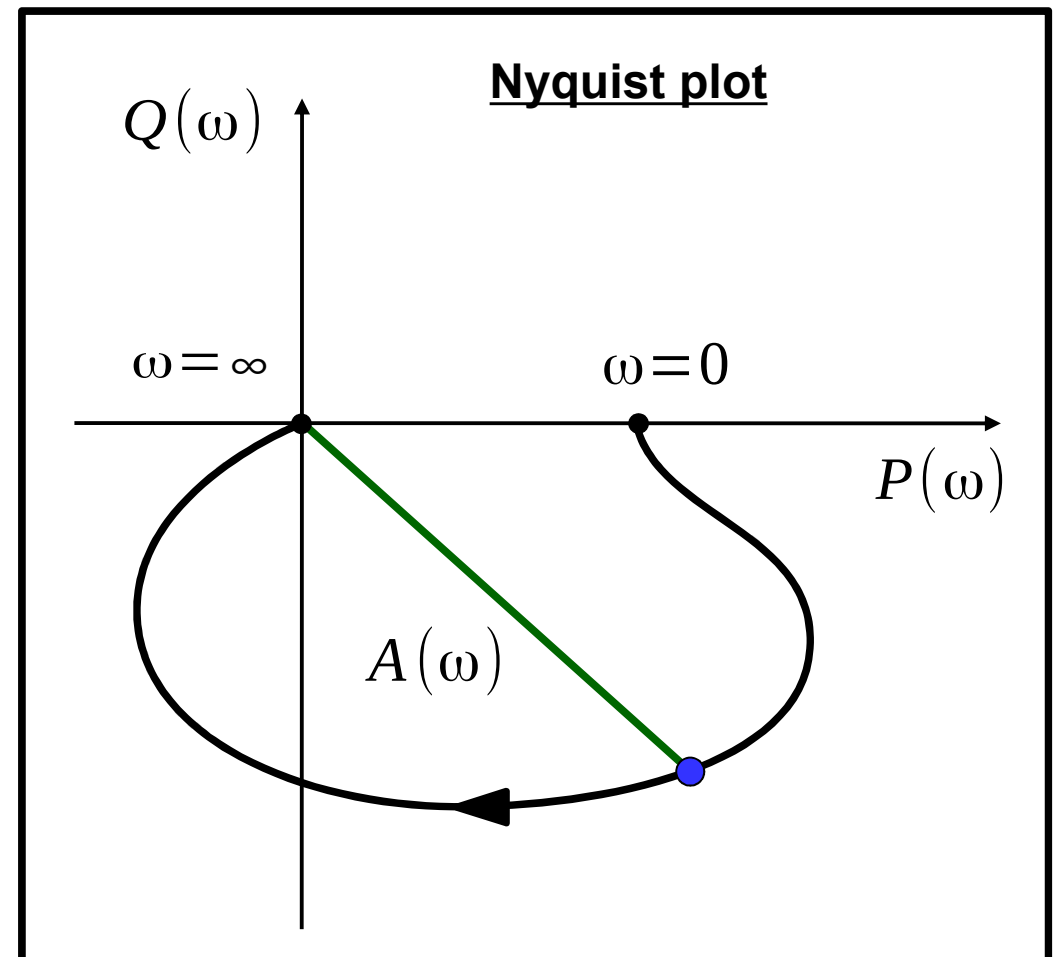
input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

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**GAIN**

$$\varphi(\omega) = \text{Arg } H(j\omega) = \arctan \frac{Q}{P}$$



# Transfer function – frequency response

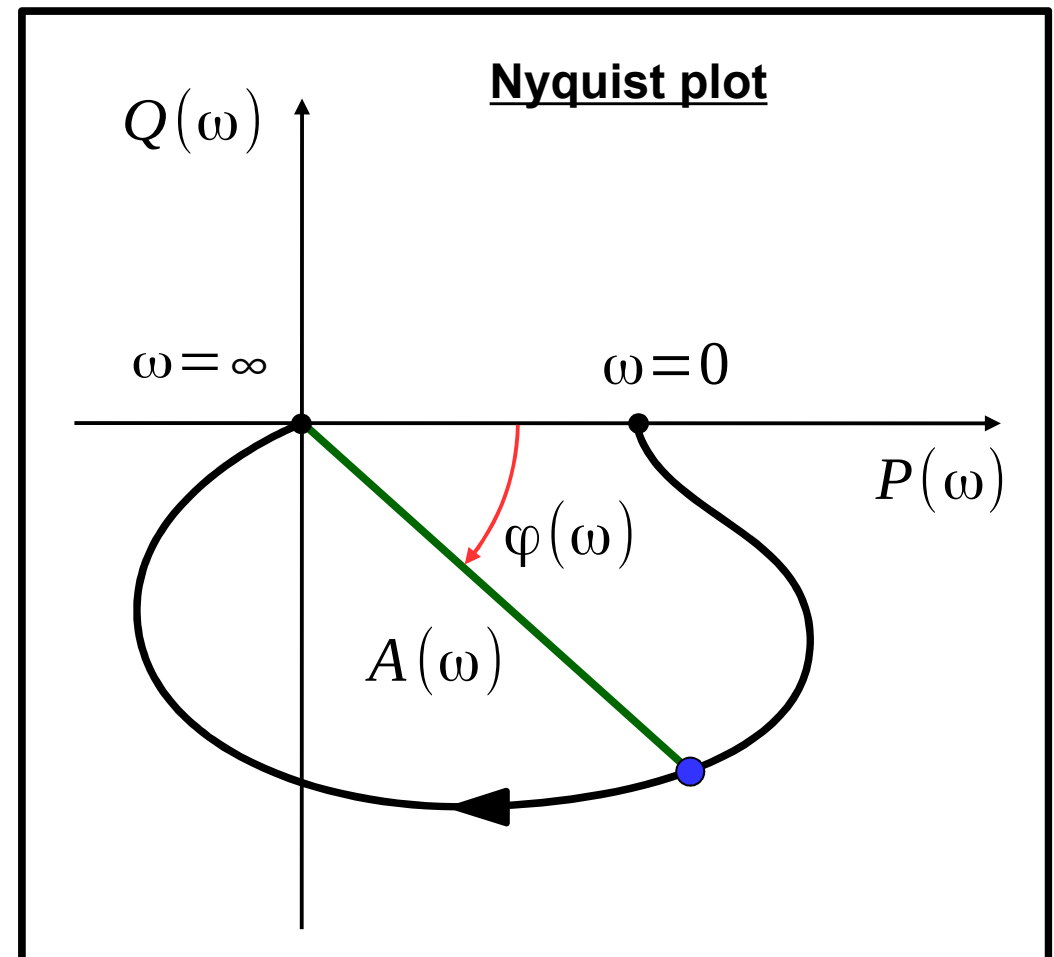
input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

$$H(s) \xrightarrow{s=j\omega} H(j\omega) = P(\omega) + jQ(\omega)$$

$$A(\omega) = |H(j\omega)| = \sqrt{P^2(\omega) + Q^2(\omega)}$$

$$\varphi(\omega) = \text{Arg } H(j\omega) = \arctan \frac{Q}{P}$$

DELAY



# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$

Bode Plot



# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$  transfer function:  $H(s)$  output:  $y(t) = A \sin(\omega t + \varphi)$

## Bode Plot

gain (magnitude) plot

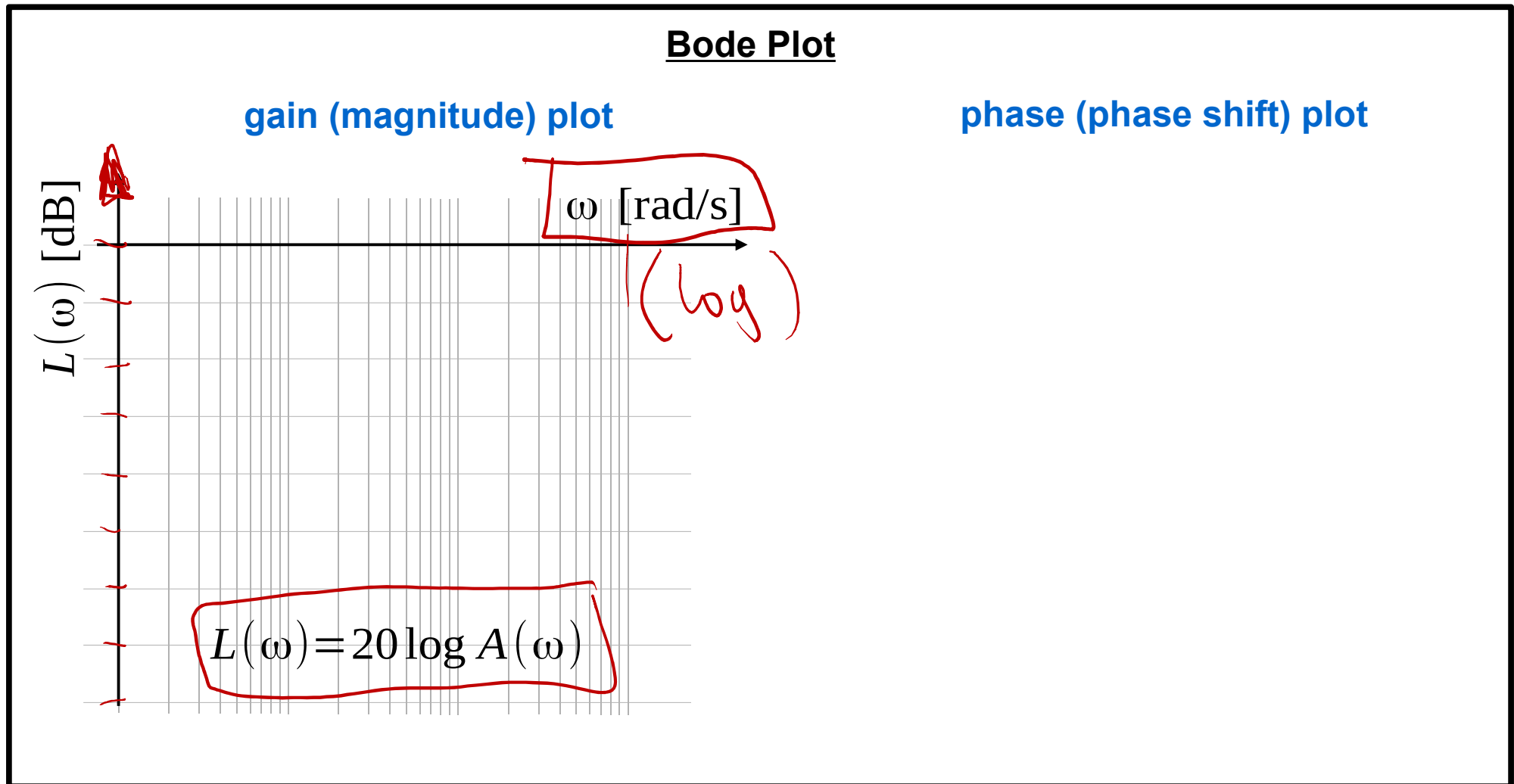
*delay*  
phase (phase shift) plot

# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$

transfer function:  $H(s)$

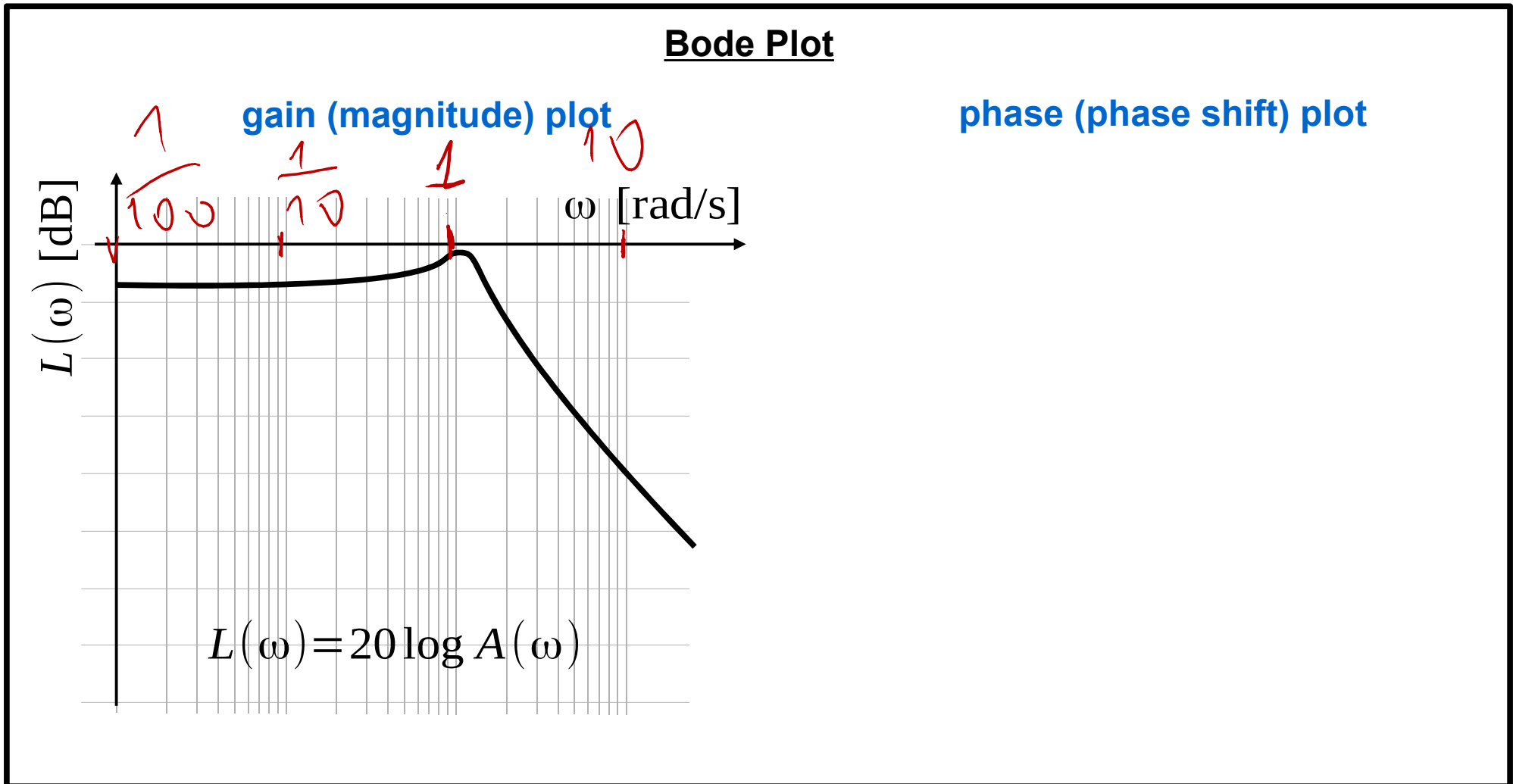
output:  $y(t) = A \sin(\omega t + \varphi)$



$$L(\omega) = 20 \cdot \log_{10} A(\omega) \text{ [dB]}$$

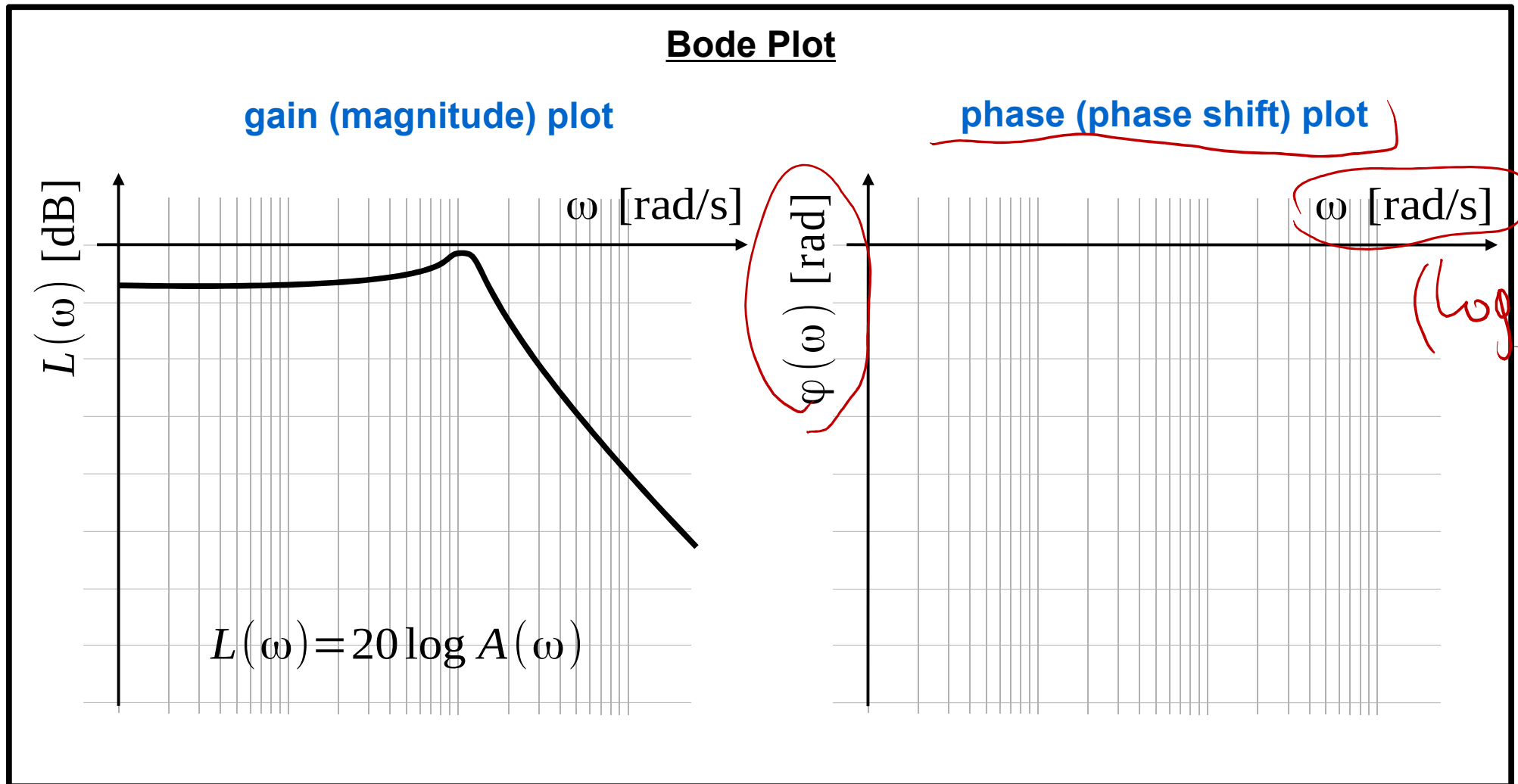
# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$



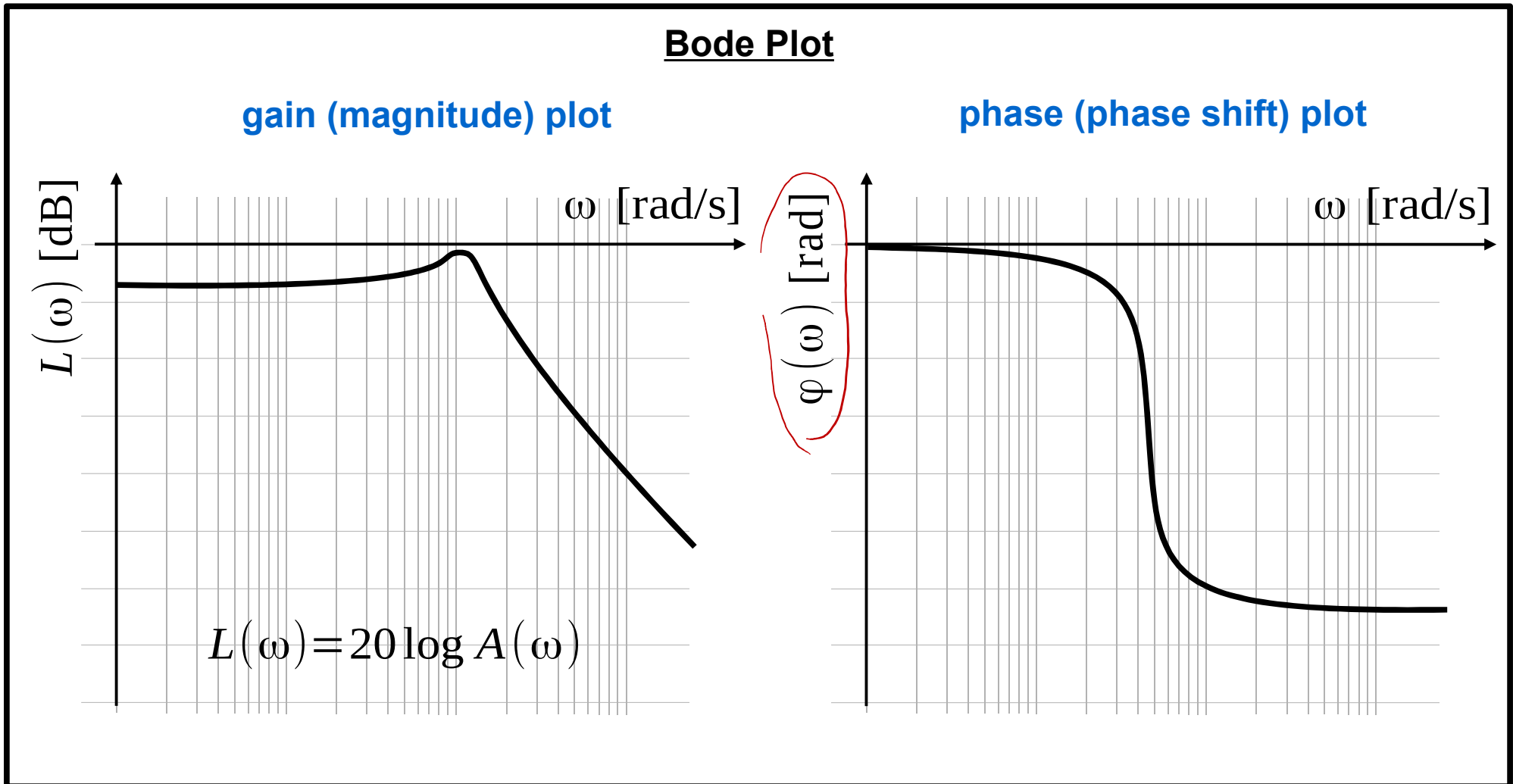
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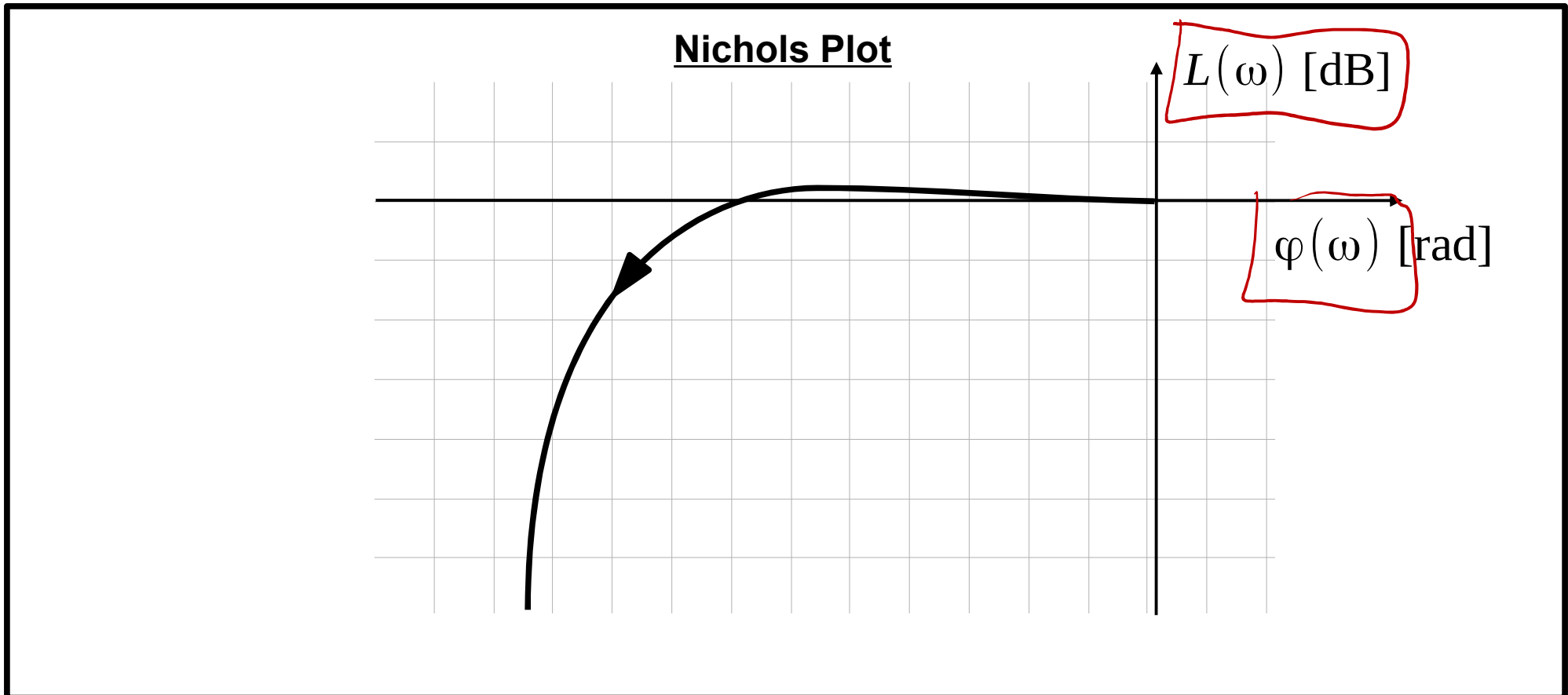
# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$



# Transfer function – frequency response

input:  $x(t) = \sin(\omega t)$       transfer function:  $H(s)$       output:  $y(t) = A \sin(\omega t + \varphi)$



# Transfer function – frequency response

20log 1

A (gain)	20logA [dB]
1000	60
100	40
10	20
1	0
0.1	-20
0.01	-40
0.001	-60

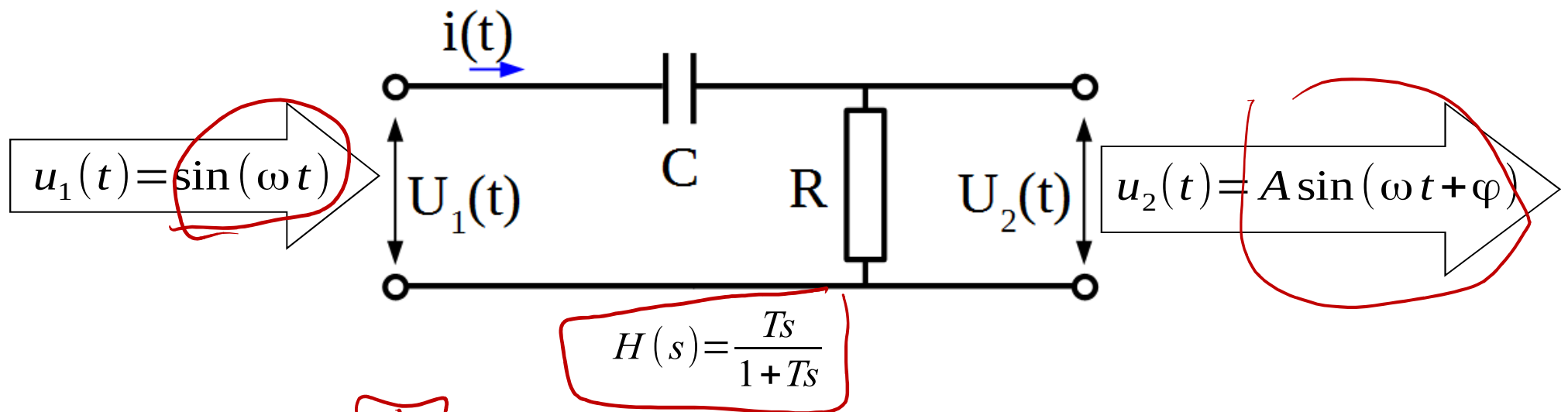
# Transfer function – frequency response

Useful for:

- audio components (amplifiers, microphones, loudspeakers, cables)
- wireless components (antennas, amplifiers)
- vibrating systems (suspensions, drivetrains)
- control systems (regulators, objects)

# Transfer function – frequency response

## RC circuit example



$$H(j\omega) = \frac{Tj\omega}{1 + Tj\omega} = P(\omega) + jQ(\omega)$$

$$\frac{Tj\omega}{(1 + Tj\omega)} \cdot \frac{1 - Tj\omega}{(1 - Tj\omega)} = \frac{Tj\omega(1 - Tj\omega)}{1^2 - T^2(j\omega)^2} = \frac{Tj\omega - T^2(j\omega)^2}{1 + T^2\omega^2}$$

# Transfer function – frequency response

## RC circuit example

$$H(s) = \frac{Ts}{1+Ts}$$

$$H(j\omega) = \frac{Tj\omega + T^2\omega^2}{1 + T^2\omega^2}$$

$$= \underbrace{\frac{T^2\omega^2}{1+T^2\omega^2}}_{P(\omega)} + j \underbrace{\frac{T\omega}{1+T^2\omega^2}}_{Q(\omega)}$$

$$A(\omega) = \sqrt{P^2 + Q^2} = \sqrt{\frac{T^4\omega^4 + T^2\omega^2}{(1+T^2\omega^2)^2}} =$$

$$= \sqrt{\frac{T^2\omega^2 \cancel{(T^2\omega^2 + 1)}}{(1+T^2\omega^2)^2}} = \frac{T\omega}{\sqrt{1+T^2\omega^2}}$$

$$L(\omega) = 20 \log A(\omega) = 20 \log \frac{T\omega}{\sqrt{1+T^2\omega^2}} = 20 \log T\omega -$$

$$\varphi(\omega) = \arctan \frac{Q}{P} = \arctan \left( \frac{1}{T\omega} \right) \quad 20 \log \sqrt{1+T^2\omega^2}$$

# Transfer function – frequency response

## RC circuit example

$$H(s) = \frac{Ts}{1 + Ts}$$

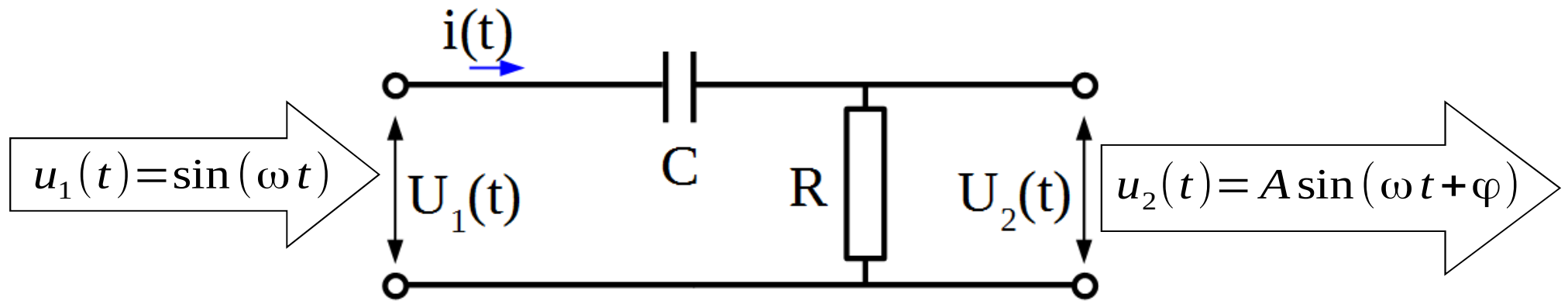
# Transfer function – frequency response

## RC circuit example

$$H(s) = \frac{Ts}{1 + Ts}$$

# Transfer function – frequency response

## RC circuit example



$$H(s) = \frac{Ts}{1+Ts}$$

$$s = j\omega$$

$$H(j\omega) = \frac{Tj\omega}{1+Tj\omega} = \frac{Tj\omega}{1+Tj\omega} \frac{1-Tj\omega}{1-Tj\omega} = \frac{Tj\omega - T^2j^2\omega^2}{1^2 - T^2j^2\omega^2} = \frac{Tj\omega + T^2\omega^2}{1^2 + T^2\omega^2} = \frac{T^2\omega^2}{1^2 + T^2\omega^2} + j \frac{T\omega}{1^2 + T^2\omega^2}$$

$$P(\omega) = \frac{T^2\omega^2}{1+T^2\omega^2} \quad Q(\omega) = \frac{T\omega}{1+T^2\omega^2} \quad A(\omega) = |H(j\omega)| = \sqrt{P^2(\omega) + Q^2(\omega)} = \frac{T\omega}{\sqrt{T^2\omega^2 + 1}}$$

$$L(\omega) = 20 \log A(\omega) = 20 \log \frac{T\omega}{\sqrt{T^2\omega^2 + 1}} = 20 \log T\omega - 20 \log \sqrt{T^2\omega^2 + 1}$$

$$\varphi(\omega) = \arg H(j\omega) = \arctan \frac{Q}{P} = \arctan \left( \frac{1}{T\omega} \right)$$

# Transfer function – frequency response

## RC circuit example

Nyquist plot

$$P(\omega) = \frac{T^2 \omega^2}{1 + T^2 \omega^2} \quad Q(\omega) = \frac{T \omega}{1 + T^2 \omega^2}$$

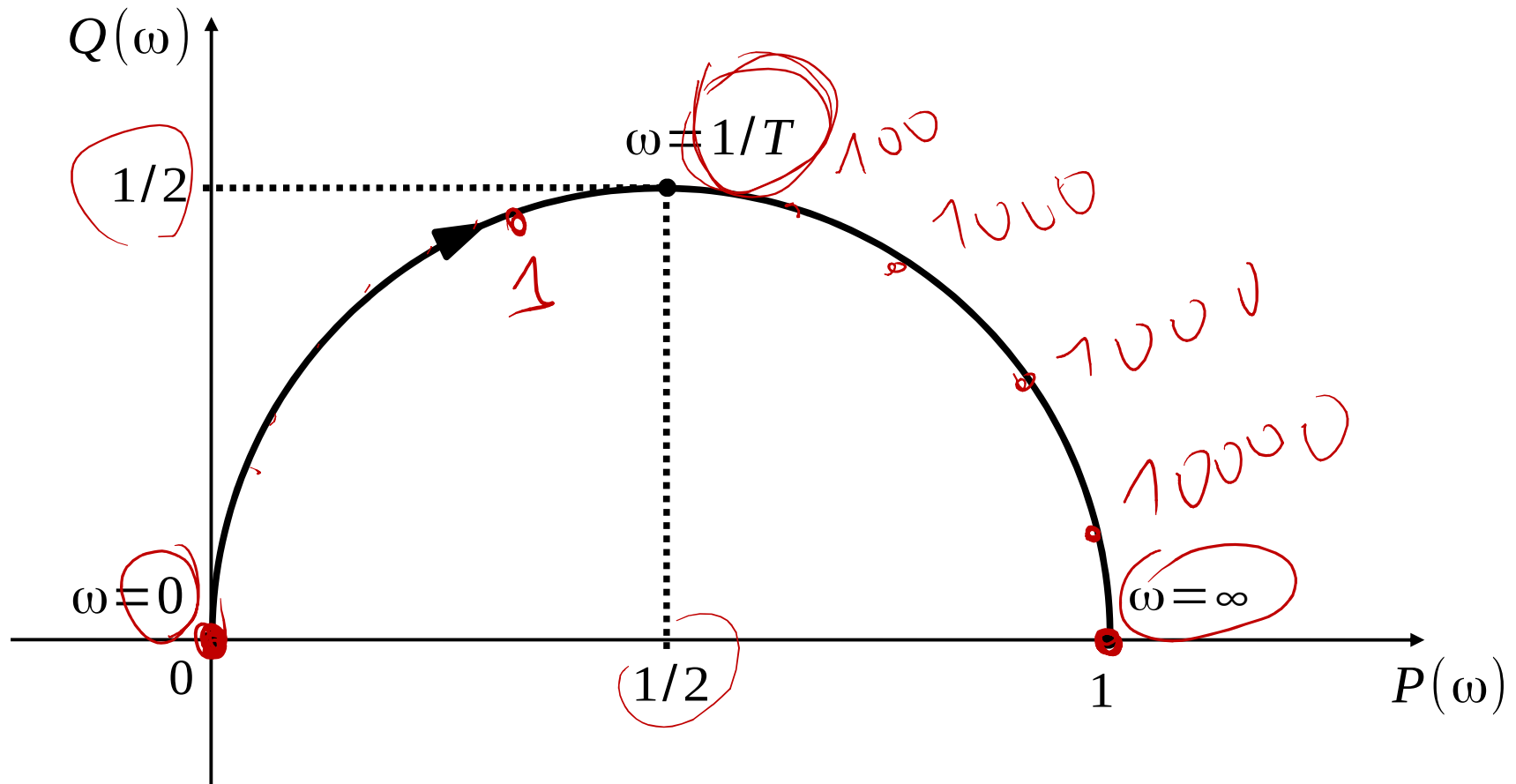
# Transfer function – frequency response

## RC circuit example

$$\left(P - \frac{1}{2}\right)^2 + Q^2 = \frac{1}{4}$$

Nyquist plot

$$P(\omega) = \frac{T^2 \omega^2}{1 + T^2 \omega^2} \quad Q(\omega) = \frac{T \omega}{1 + T^2 \omega^2}$$



# Transfer function – frequency response

## RC circuit example

$$A(\omega) = \frac{T\omega}{\sqrt{T^2\omega^2 + 1}}$$

LIN.

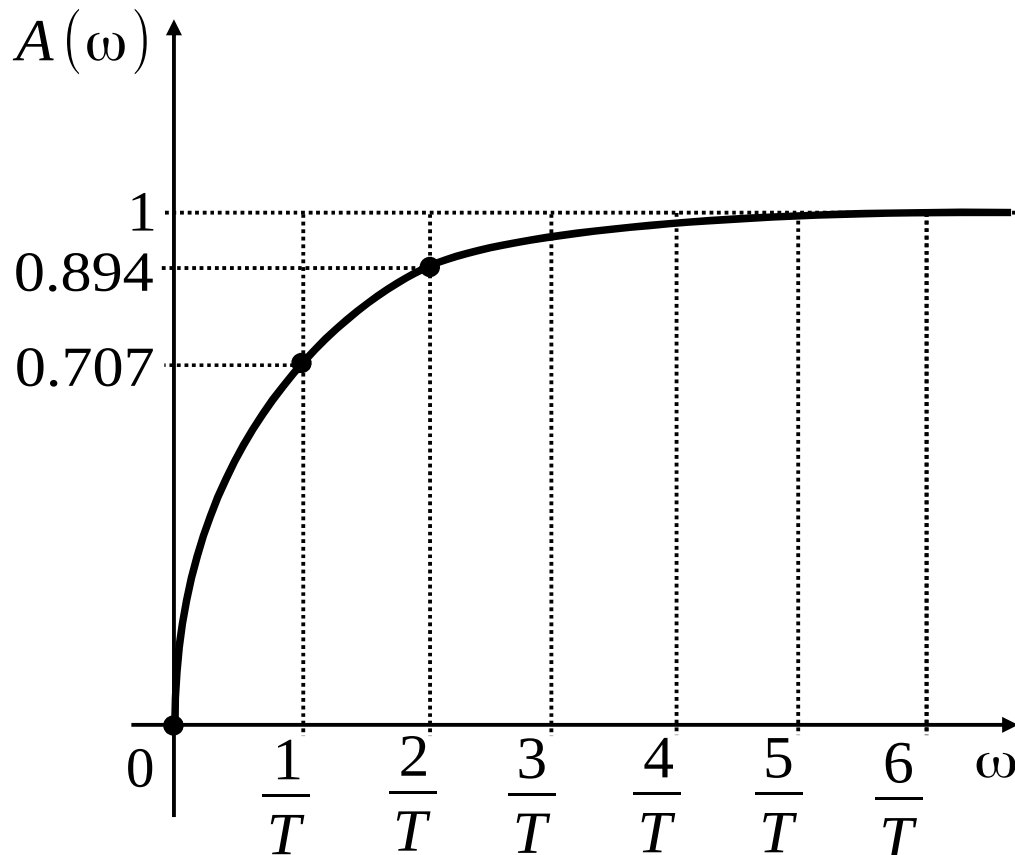
$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$

# Transfer function – frequency response

## RC circuit example

$$A(\omega) = \frac{T\omega}{\sqrt{T^2\omega^2 + 1}}$$

$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$



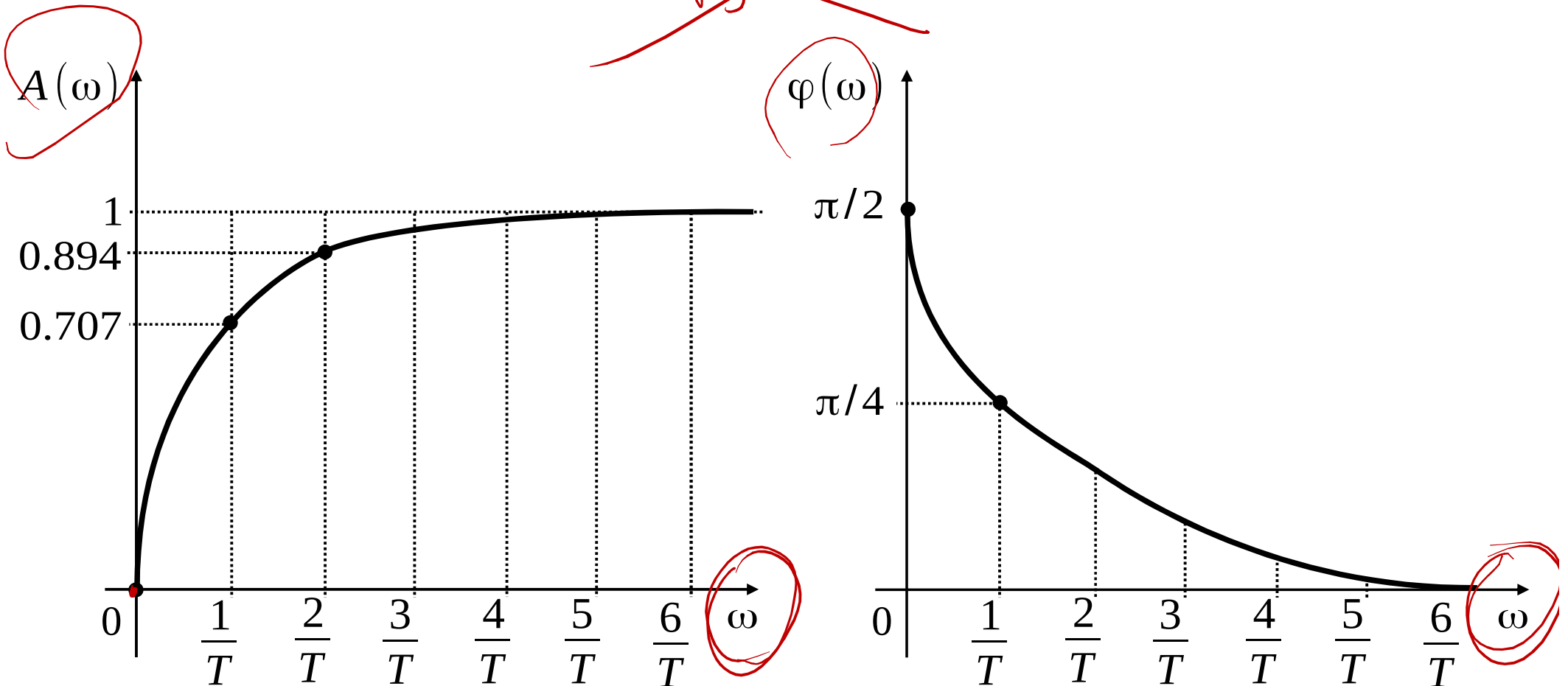
# Transfer function – frequency response

## RC circuit example

$$A(\omega) = \frac{T\omega}{\sqrt{T^2\omega^2 + 1}}$$

$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$

~~Bode~~



# Transfer function – frequency response

## RC circuit example

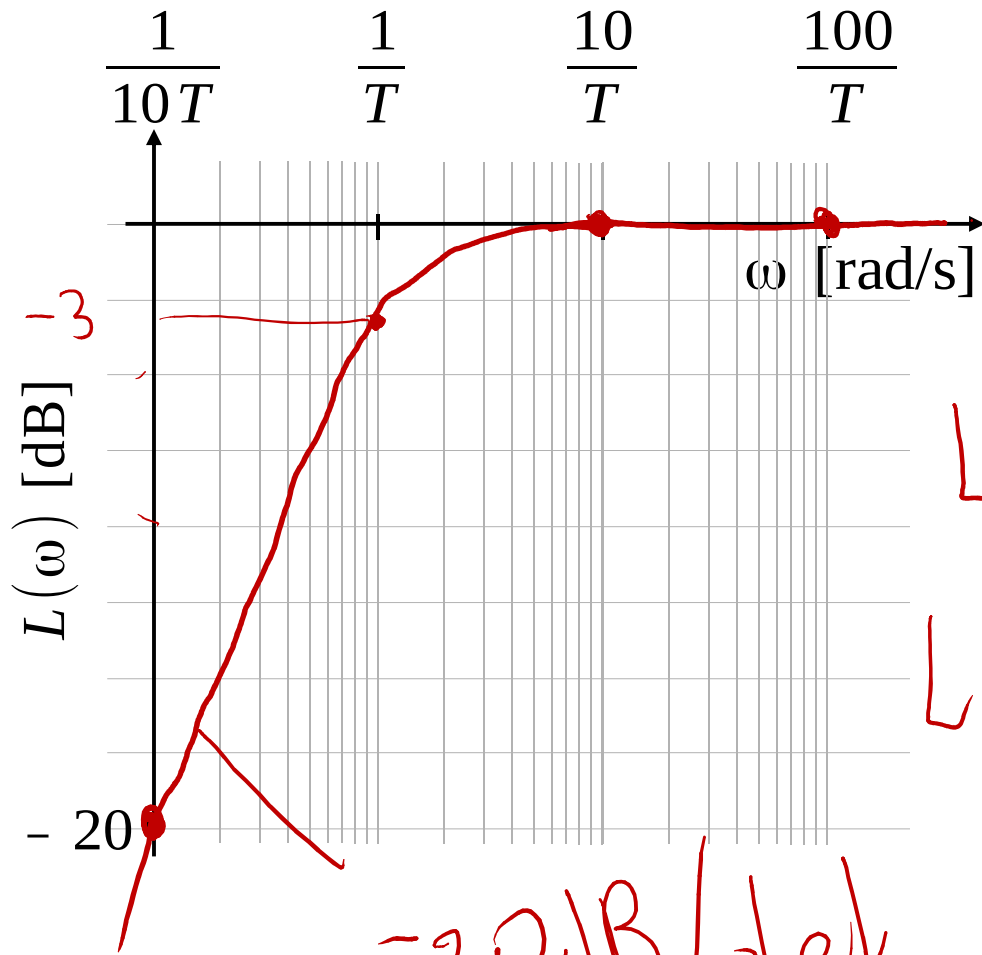
$$\underline{L(\omega)} = 20 \log T \omega - 20 \log \sqrt{T^2 \omega^2 + 1}$$

$$\underline{\varphi(\omega)} = \arctan \left( \frac{1}{T \omega} \right)$$

# Transfer function – frequency response $\approx -3$

## RC circuit example

$$L(\omega) = 20 \log T \omega - 20 \log \sqrt{T^2 \omega^2 + 1}$$



$$L\left(\omega = \frac{1}{T}\right) = 20 \log 1 - 20 \log \sqrt{2}$$

$$L\left(\omega = \frac{10}{T}\right) = 20 \log 10 - 20 \log \sqrt{101} \approx 0$$

$$L\left(\omega = \frac{100}{T}\right) = 20 \log 100 - 20 \log \sqrt{10001} = 0$$

$$L\left(\omega = \frac{1}{10T}\right) = 20 \log \frac{1}{10} - 20 \log \sqrt{\frac{101}{100}} = -20$$

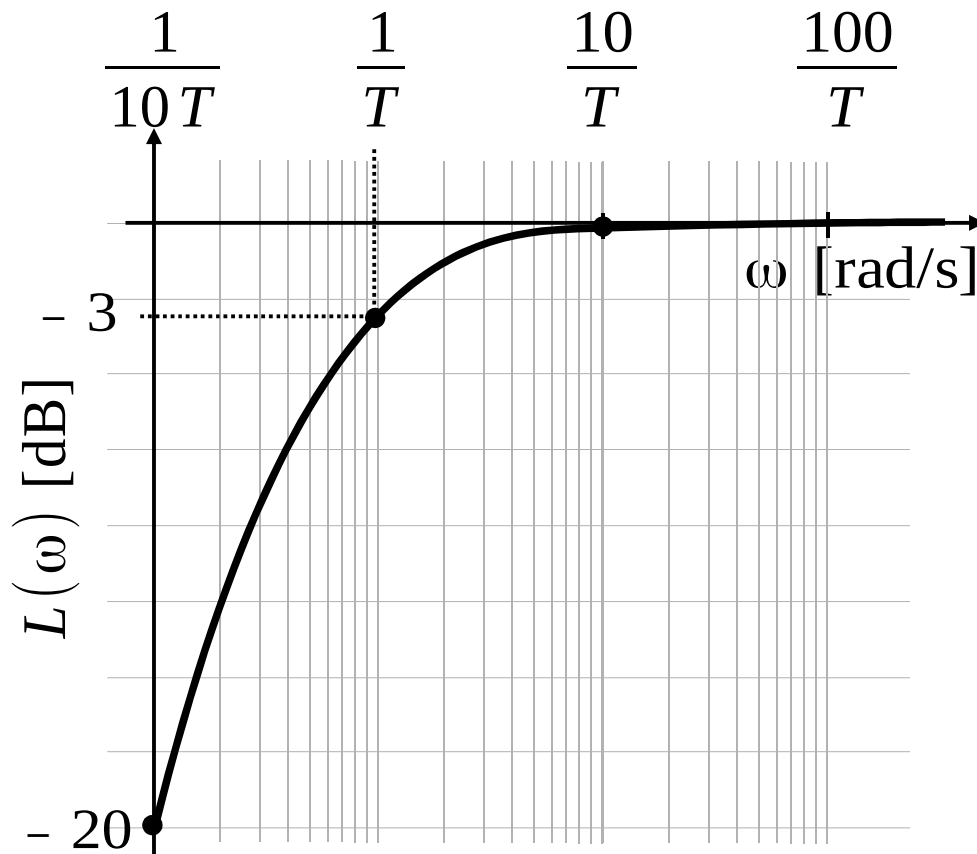
$$L\left(\omega = \frac{1}{100T}\right) = 20 \log \frac{1}{100} - 20 \log \sqrt{\frac{10001}{10000}} = -40$$

$-20 \text{ dB/dec}$

# Transfer function – frequency response

## RC circuit example

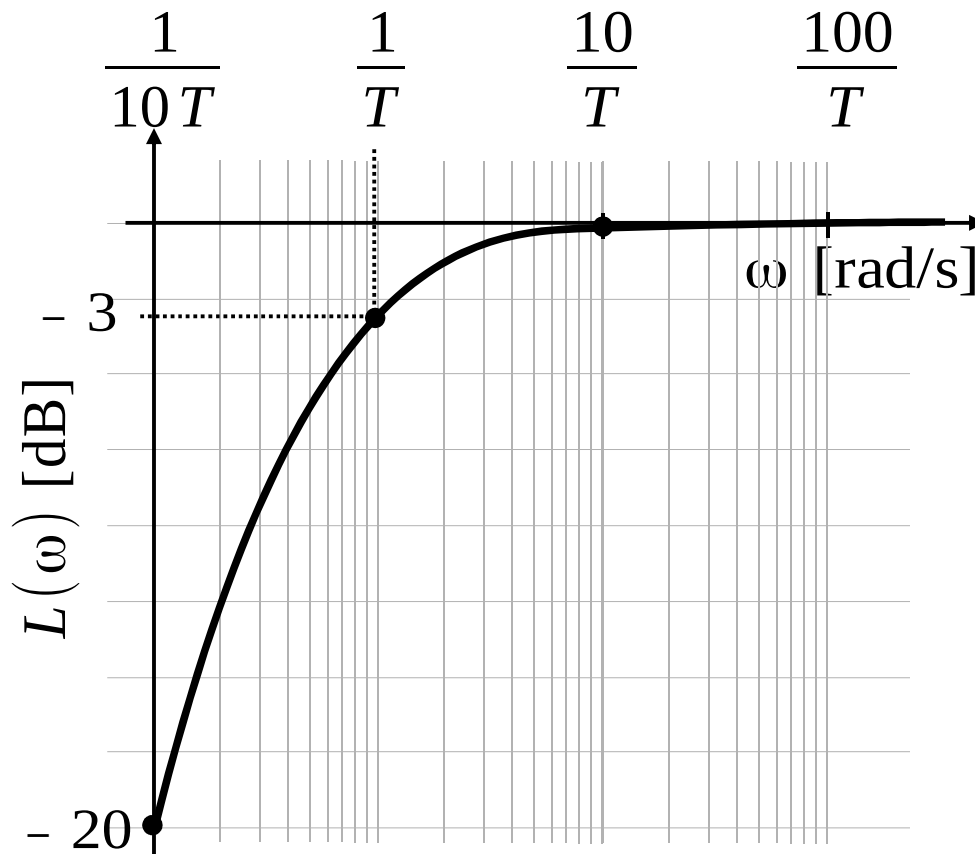
$$L(\omega) = 20 \log T \omega - 20 \log \sqrt{T^2 \omega^2 + 1}$$



# Transfer function – frequency response

## RC circuit example

$$L(\omega) = 20 \log T \omega - 20 \log \sqrt{T^2 \omega^2 + 1}$$



$$L\left(\omega = \frac{1}{T}\right) = 20 \log 1 - 20 \log \sqrt{2} \approx -3$$

$$L\left(\omega = \frac{10}{T}\right) = 20 \log 10 - 20 \log \sqrt{101} \approx 0$$

$$L\left(\omega = \frac{100}{T}\right) = 20 \log 100 - 20 \log \sqrt{10001} \approx 0$$

$$L\left(\omega = \frac{1}{10T}\right) = 20 \log 0,1 - 20 \log \sqrt{1,01} \approx -20$$

$$L\left(\omega = \frac{1}{100T}\right) = 20 \log 0,01 - 20 \log \sqrt{1,0001} \approx -40$$

# Transfer function – frequency response

## RC circuit example

$$\varphi(\omega = \frac{1}{T}) = \arctan(1) = \frac{\pi}{4} = 45^\circ$$

$$\varphi(\omega = \frac{10}{T}) = \arctan(\frac{1}{10}) \approx 6^\circ$$

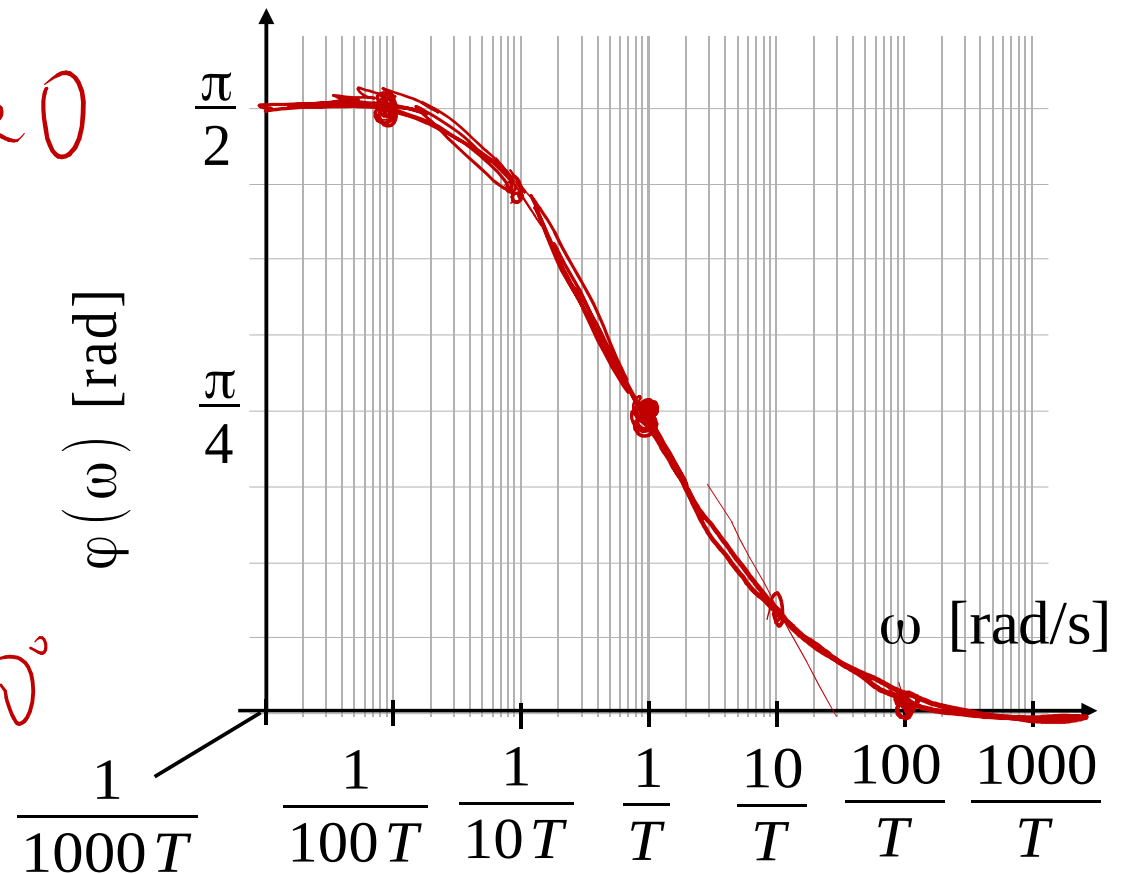
$$\varphi(\omega = \frac{100}{T}) = \arctan(\frac{1}{100}) \approx 0,7^\circ \approx 0$$

$$\varphi(\omega = \frac{1000}{T}) = 0$$

$$\varphi(\omega = \frac{1}{10T}) = \arctan(10) \approx 84^\circ$$

$$\varphi(\omega = \frac{1}{100T}) = \arctan(100) \approx 90^\circ$$

$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$



# Transfer function – frequency response

## RC circuit example

$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$



# Transfer function – frequency response

## RC circuit example

$$\varphi(\omega) = \arctan\left(\frac{1}{T\omega}\right)$$

$$\varphi\left(\omega = \frac{1}{T}\right) = \arctan 1 = \frac{\pi}{4} = 45^\circ$$

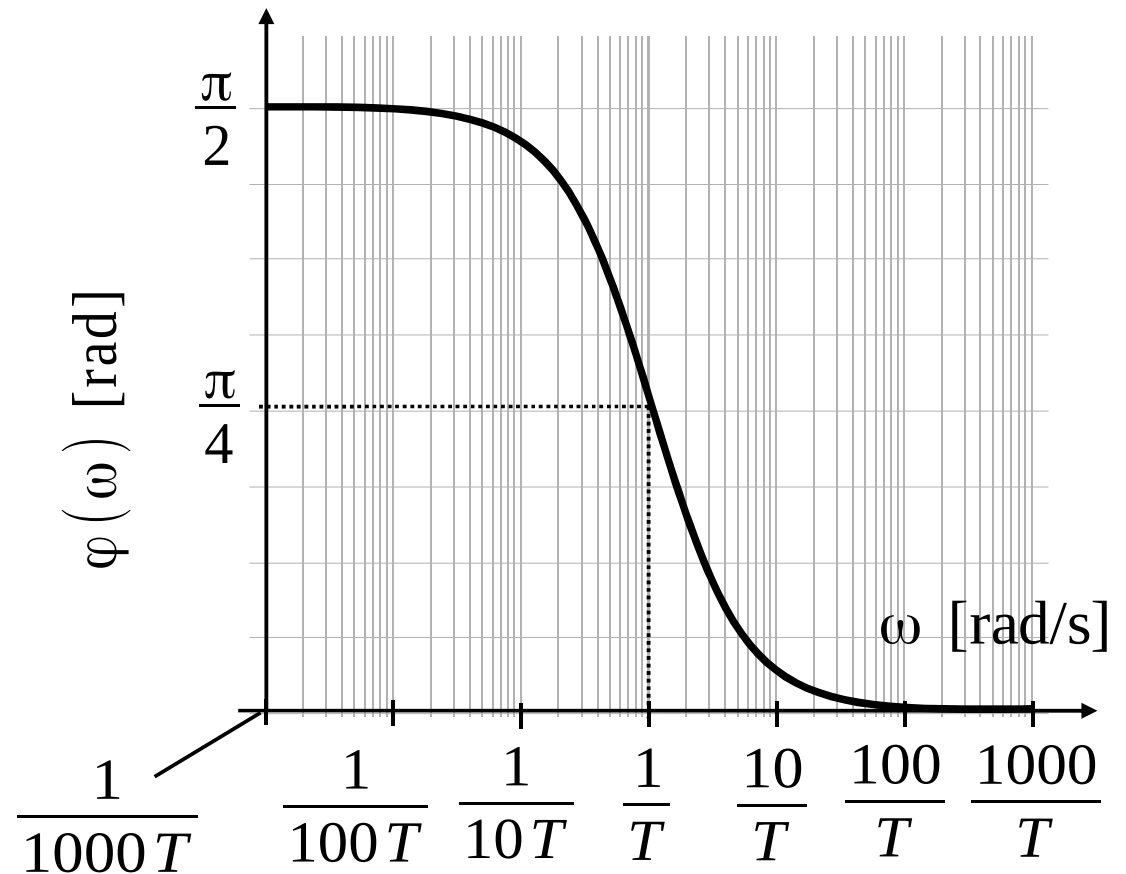
$$\varphi\left(\omega = \frac{10}{T}\right) = \arctan \frac{1}{10} = 0,1 = 5,71^\circ$$

$$\varphi\left(\omega = \frac{100}{T}\right) = \arctan \frac{1}{100} = 0,01 = 0,57^\circ$$

$$\varphi\left(\omega = \frac{1000}{T}\right) = \arctan \frac{1}{1000} = 0,001 = 0,06^\circ$$

$$\varphi\left(\omega = \frac{1}{10T}\right) = \arctan 10 = 1,47 = 84,29^\circ$$

$$\varphi\left(\omega = \frac{1}{100T}\right) = \arctan 100 = 1,56 = 89,43^\circ$$



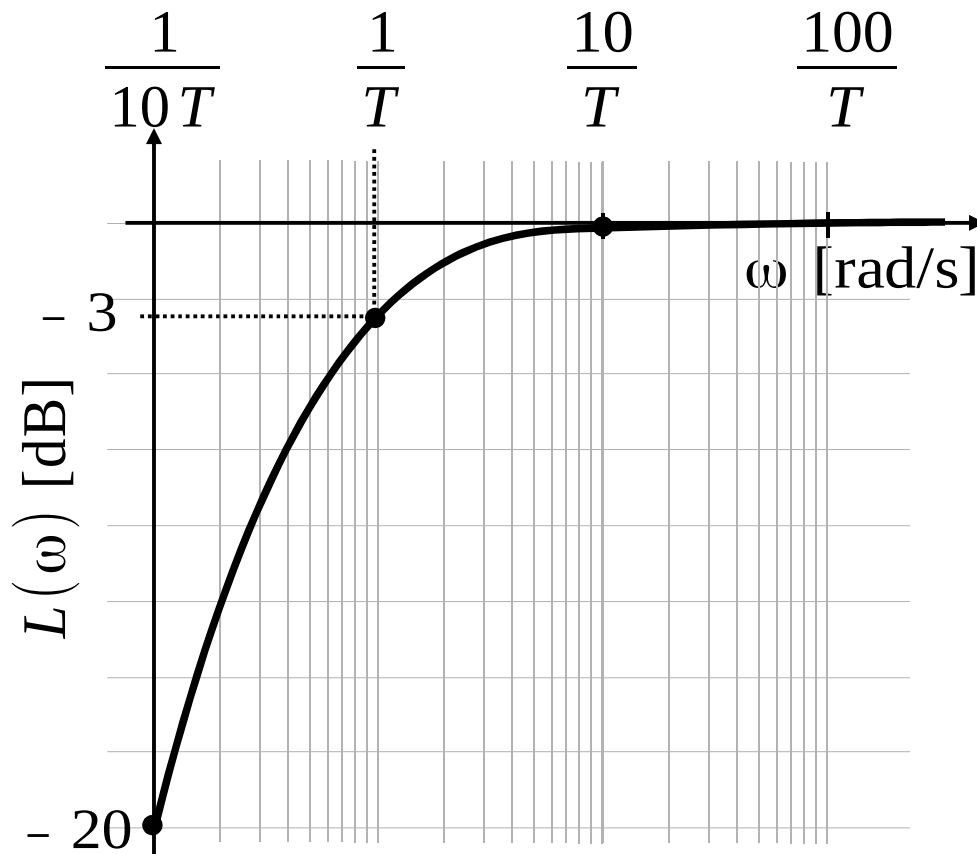
# Transfer function – frequency response

## RC circuit example

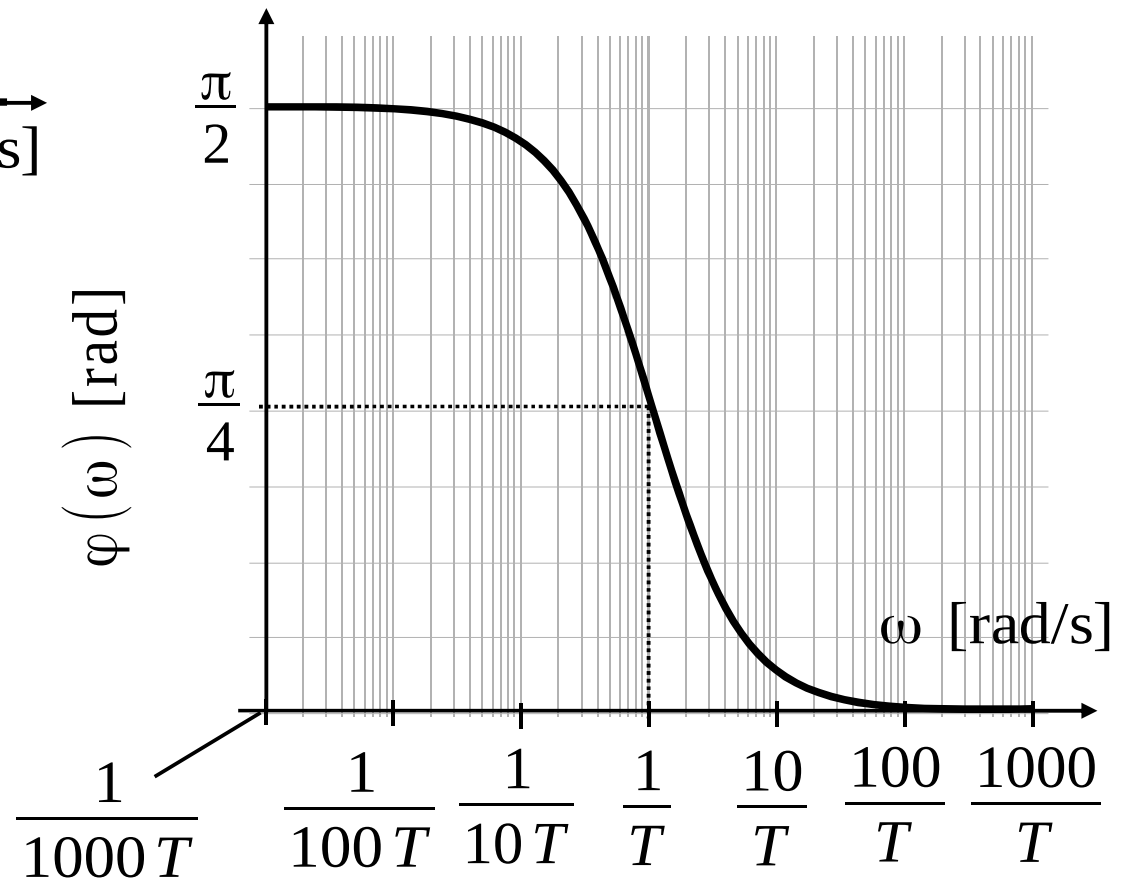
$$L(\omega) = 20 \log T \omega - 20 \log \sqrt{T^2 \omega^2 + 1}$$

$$\varphi(\omega) = \arctan\left(\frac{1}{T \omega}\right)$$

Bode plot

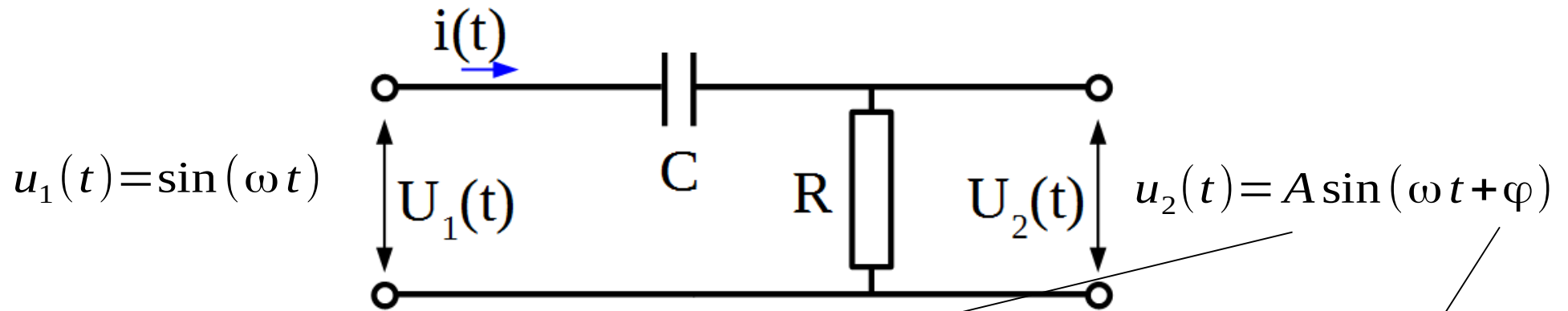


$$\omega = 2\pi f$$



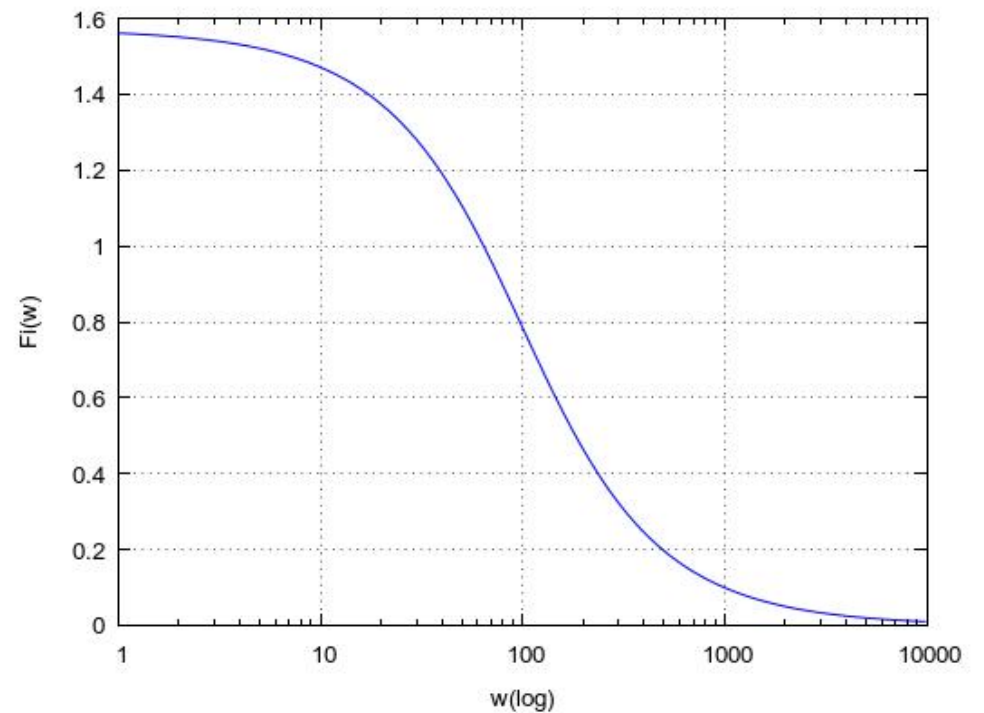
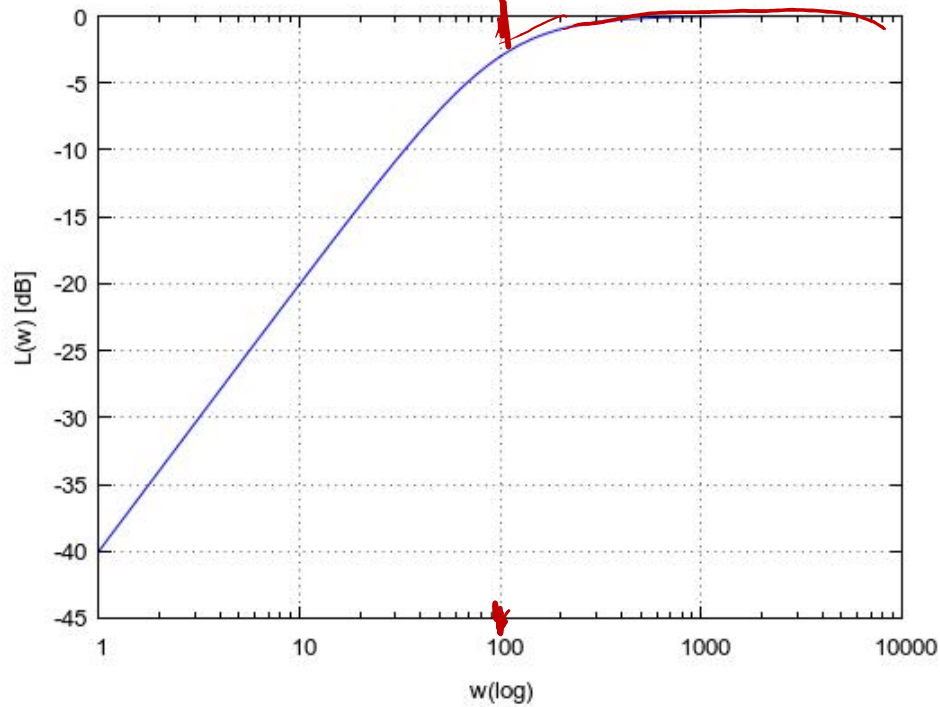
# Transfer function – frequency response

## RC circuit example



Example:

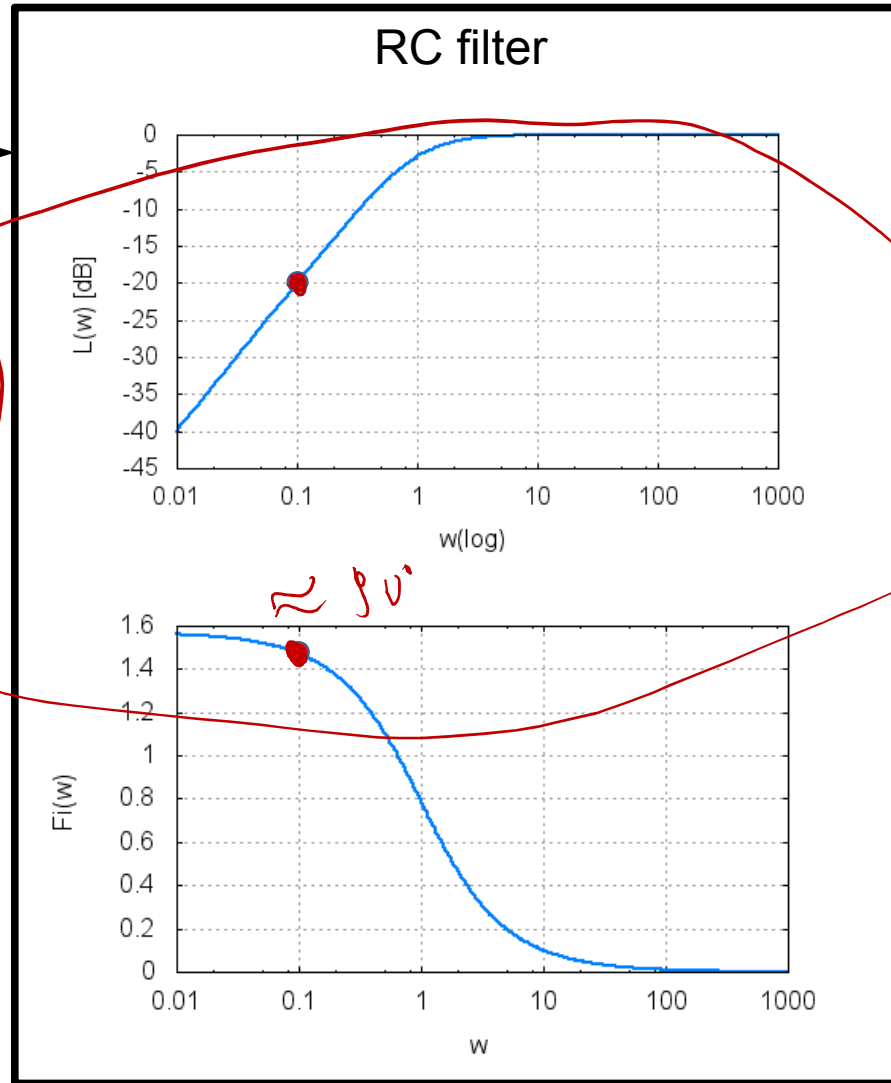
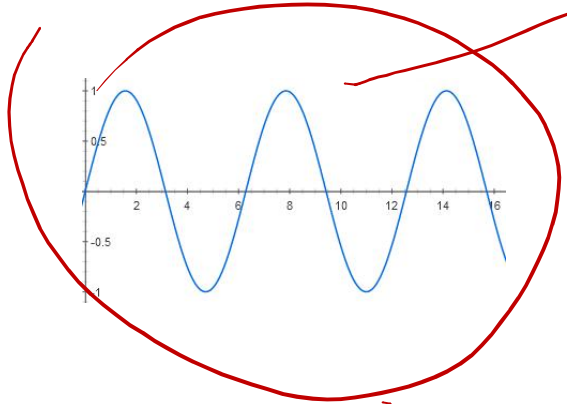
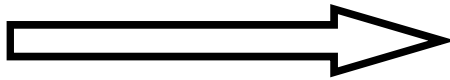
$R = 1 \text{ k}\Omega$ ,  $C = 10 \mu\text{F}$



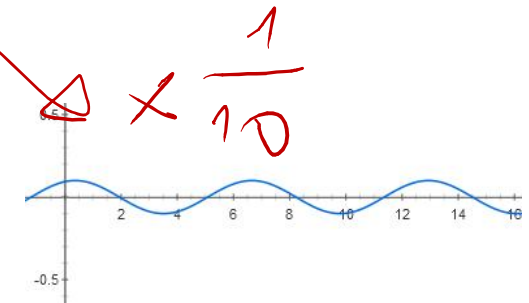
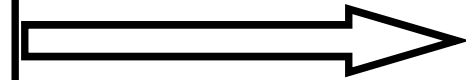
# Transfer function – frequency response

## RC circuit example

$$u_1(t) = \sin(\omega t)$$

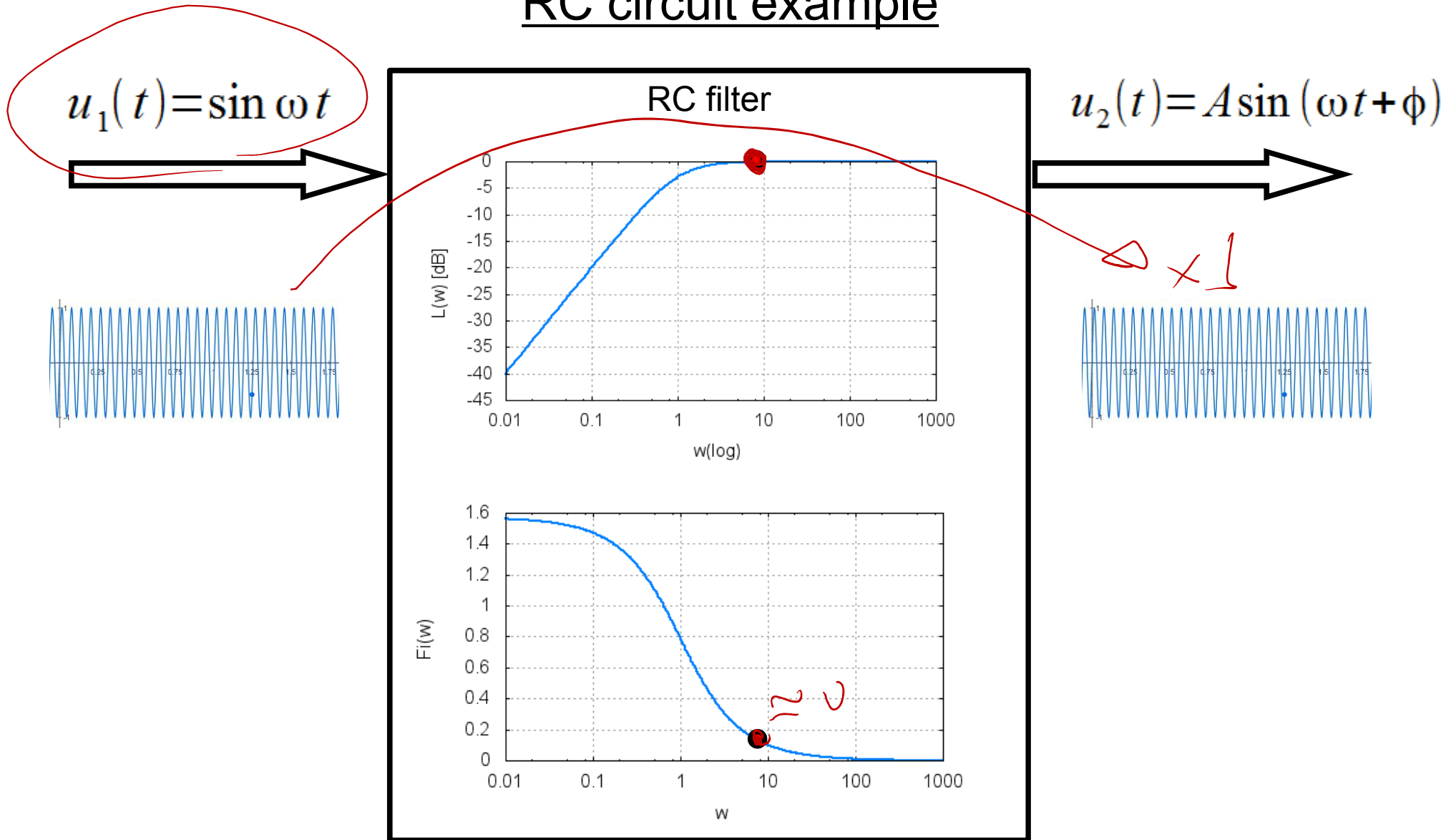


$$u_2(t) = A \sin(\omega t + \varphi)$$



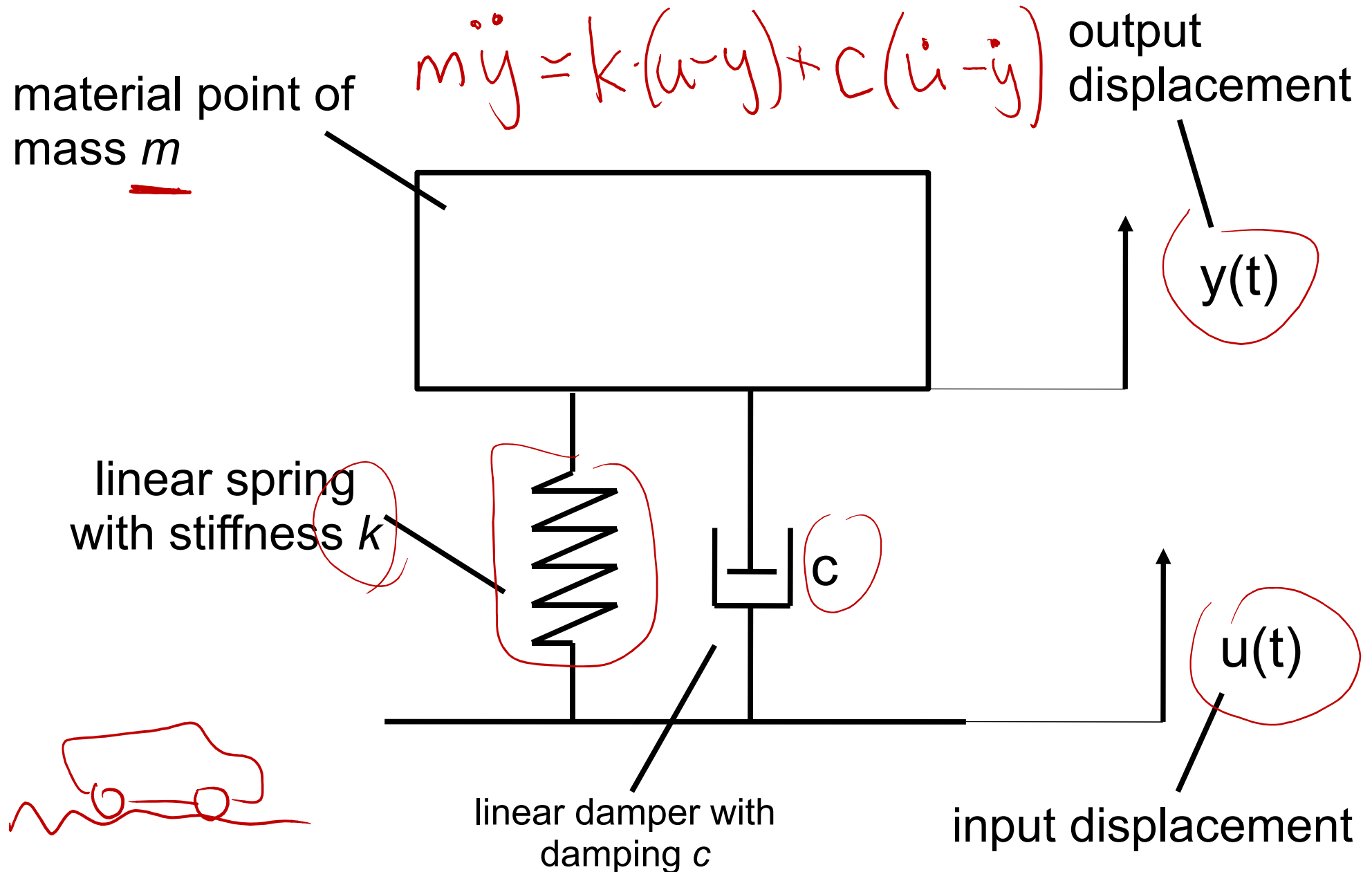
# Transfer function – frequency response

## RC circuit example



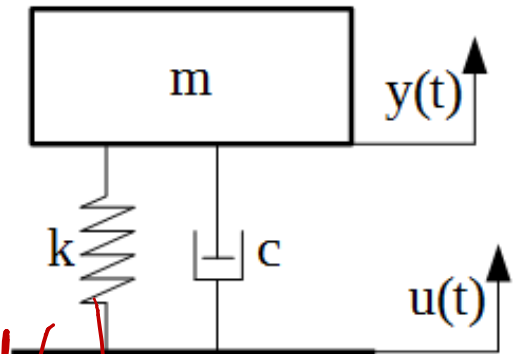
# Transfer function – frequency response

## Example 2 - vibrating system



# Transfer function – frequency response

## Example 2 - vibrating system



$$m\ddot{y} + c\dot{y} + ky = k'u + c\dot{u}$$

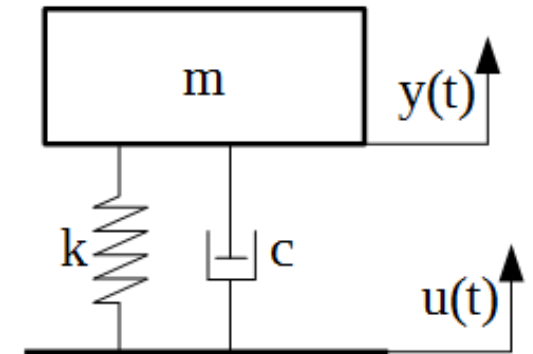
$$ms^2 Y(s) + c \cdot s \cdot Y(s) + k Y(s) = k'U(s) + csU(s)$$

$$H(s) = \frac{Y(s)}{U(s)} = \frac{k + cs}{ms^2 + cs + k}$$

$$H(j\omega) = \frac{k + cj\omega}{-m\omega^2 + cj\omega + k} = \frac{k + jc\omega}{(k - m\omega^2) + jc\omega} \cdot \frac{(k + jc\omega)(k - m\omega^2 - jc\omega)}{(k - m\omega^2) - jc\omega}$$
$$= \frac{(k + jc\omega)(k - m\omega^2) - (k + jc\omega)jc\omega}{(k - m\omega^2)^2 + c^2\omega^2} = P(\omega) + jQ(\omega)$$

# Transfer function – frequency response

## Example 2 - vibrating system



$$m \ddot{y}(t) + c \dot{y}(t) + k y(t) = c \dot{u}(t) + k u(t)$$

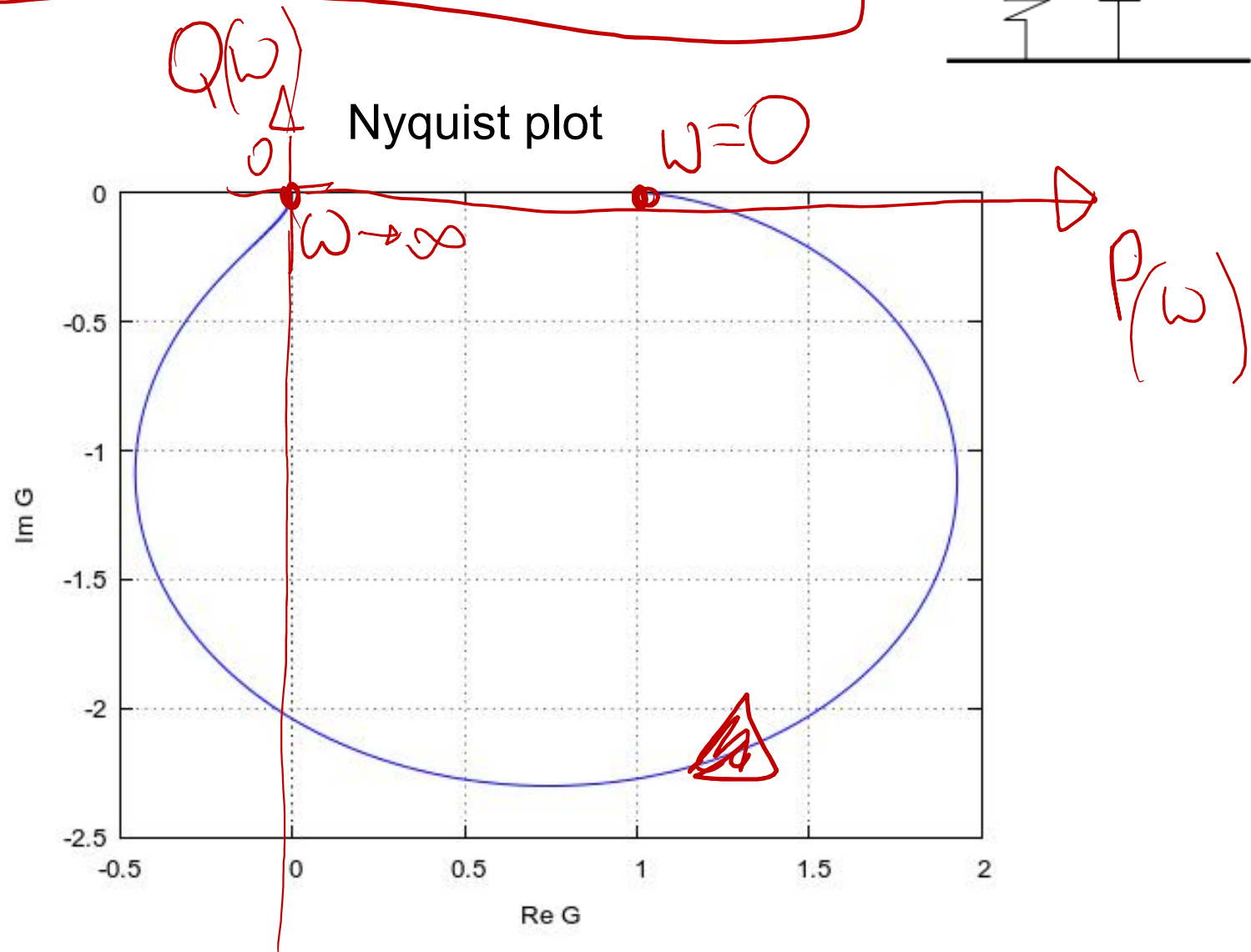
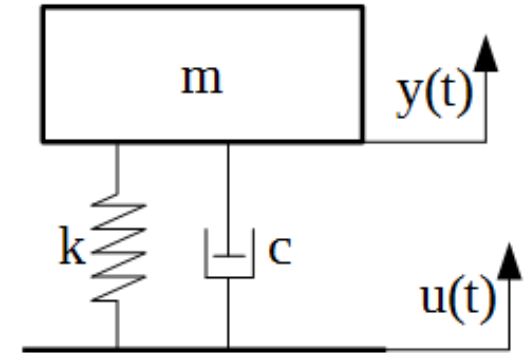
$$H(s) = \frac{cs + k}{ms^2 + cs + k}$$

$$P(\omega) = \frac{k^2 + c^2 \omega^2 - km\omega^2}{(k - m\omega^2)^2 + c^2 \omega^2}, \quad Q(\omega) = \frac{-cm\omega^3}{(k - m\omega^2)^2 + c^2 \omega^2}$$

# Transfer function – frequency response

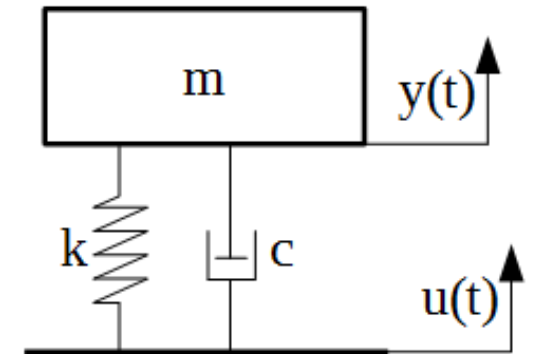
## Example 2 - vibrating system

Plots for:  $m = 300 \text{ kg}$ ,  $c = 800 \frac{\text{Ns}}{\text{m}}$ ,  $k = 11000 \frac{\text{N}}{\text{m}}$

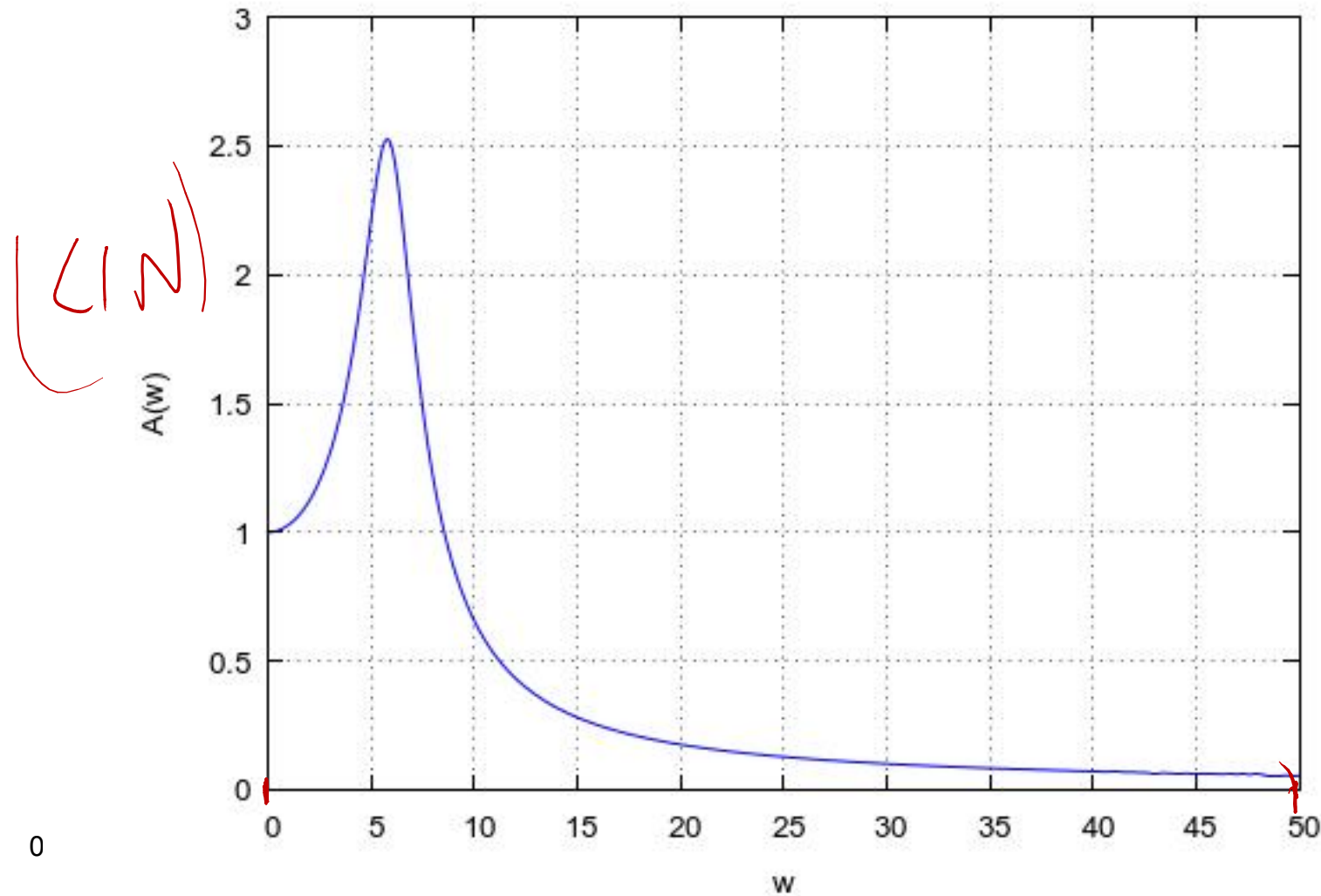


# Transfer function – frequency response

## Example 2 - vibrating system

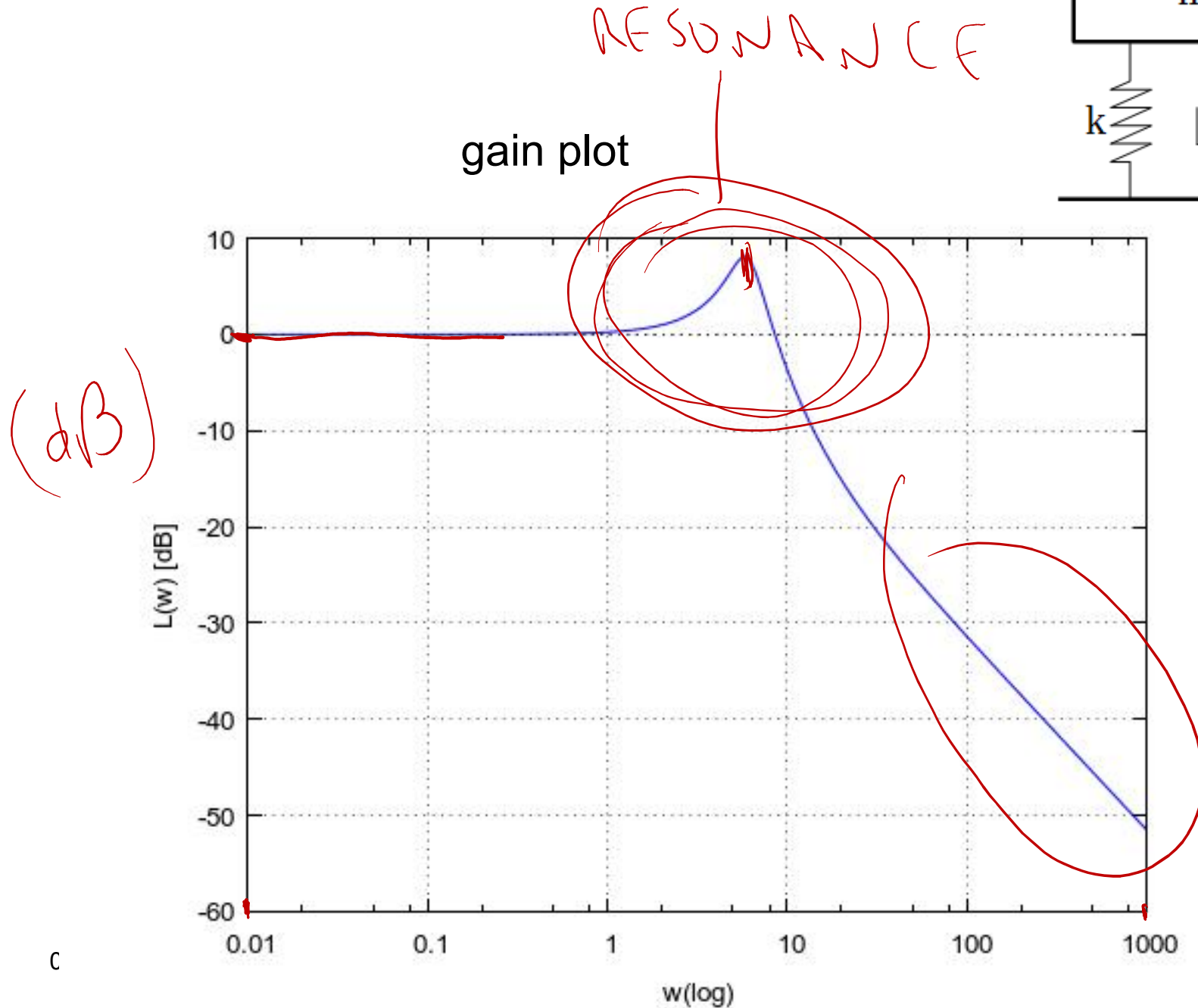
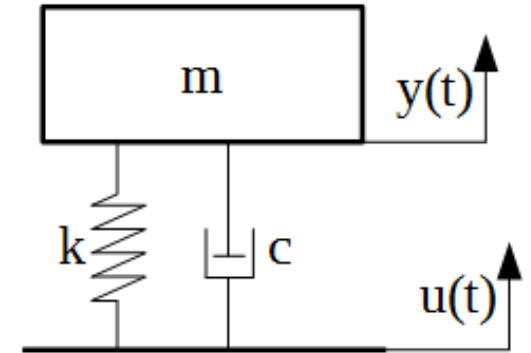


gain plot



# Transfer function – frequency response

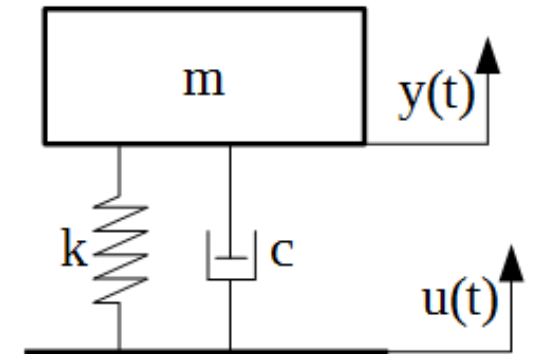
## Example 2 - vibrating system



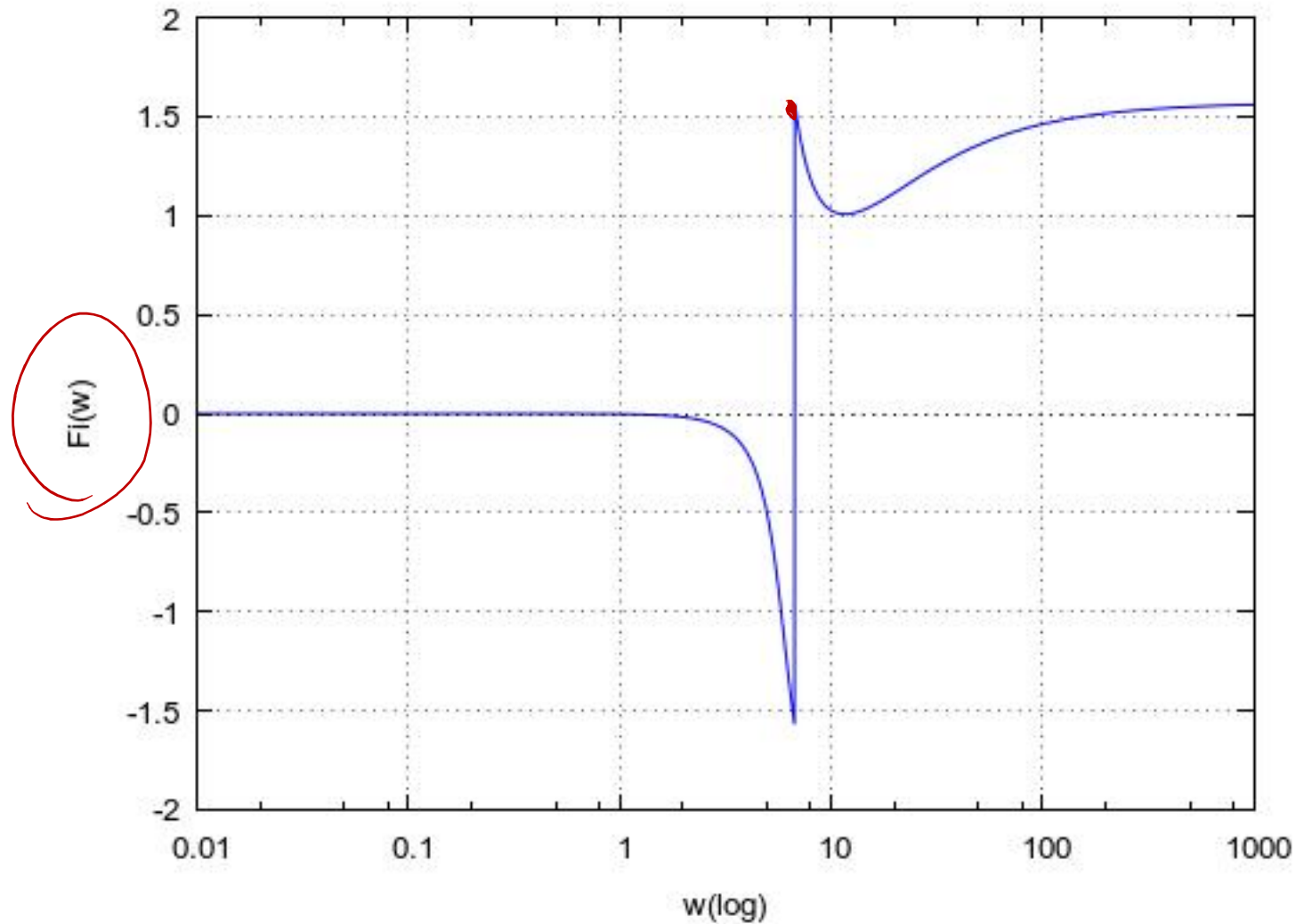
$w$  (log)

# Transfer function – frequency response

## Example 2 - vibrating system

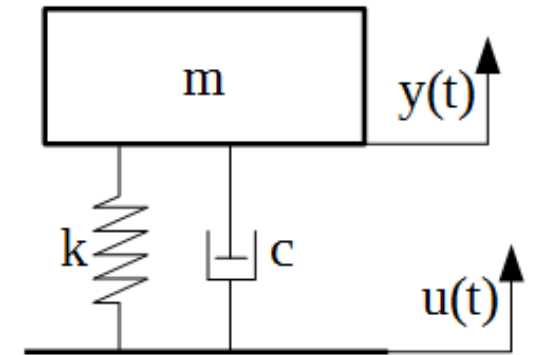


phase plot with „atan”

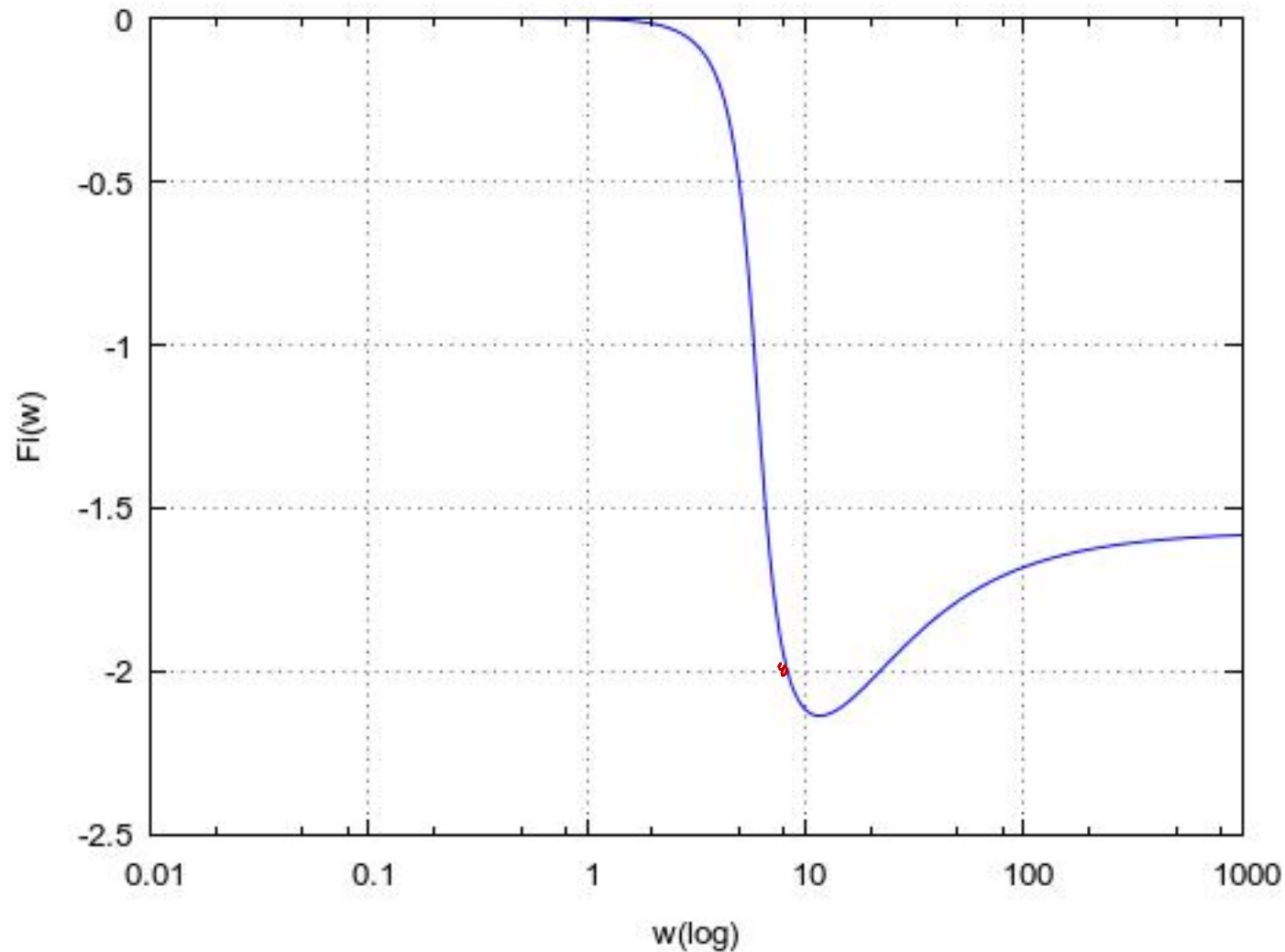


# Transfer function – frequency response

## Example 2 - vibrating system

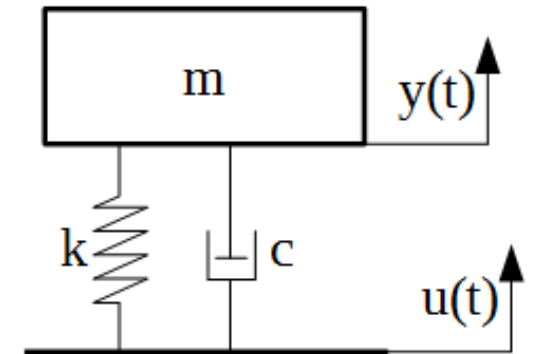


phase plot with „atan2”

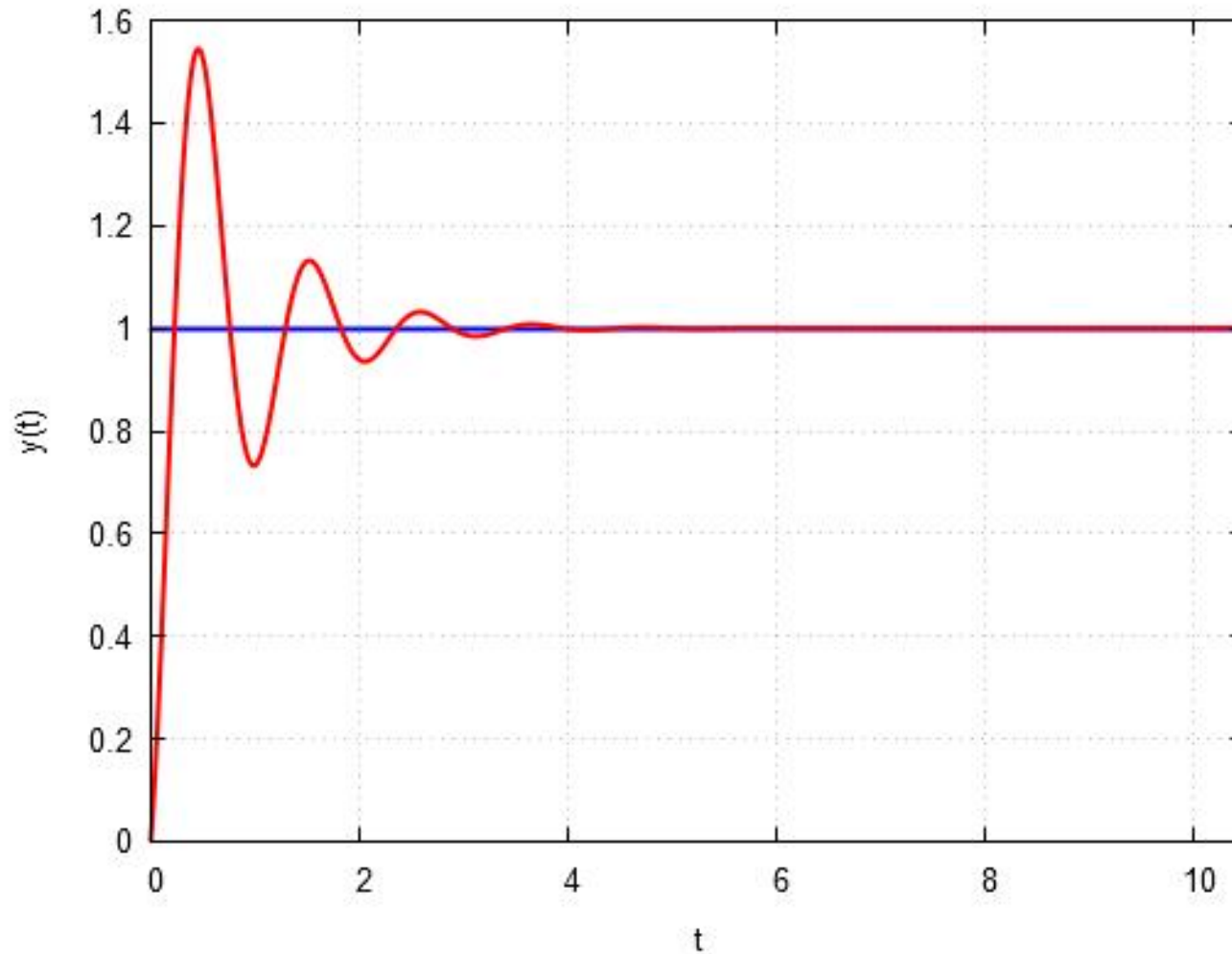


# Transfer function – frequency response

## Example 2 - vibrating system



Step response



# Classification of basic automatic systems

Element name	Equation	Transfer function
<u>proportional</u>	$y(t) = ku(t)$ <p><i>OUTPUT</i> <math>k \in \mathbb{R}</math></p>	$k$
<u>first order (inertial)</u>	$T \frac{dy(t)}{dt} + y(t) = ku(t)$	$\frac{k}{Ts + 1}$ <p><math>k \in \mathbb{R}</math> <math>T \in \mathbb{R}_+</math></p>
<u>integrator</u>	$y(t) = k \int_0^t u(t) dt$ <p>or</p> $\frac{dy(t)}{dt} = ku(t)$	$\frac{k}{s}$ <p><math>k \in \mathbb{R}</math></p>

# Classification of basic automatic systems

Element name	Equation	Transfer function
<p>derivative</p>	$y(t) = k \frac{du(t)}{dt}$	$k \in \mathbb{R}$ $ks$
<p>derivative with inertia</p>	$T \frac{dy(t)}{dt} + y(t) = k \frac{du(t)}{dt}$	$\frac{ks}{Ts + 1}$

$k \in \mathbb{R}$   
 $T \in \mathbb{R}_+$

# Classification of basic automatic systems

Element name	Equation	Transfer function
<u>delay</u>	$y(t) = u(t - \tau)$ <p style="text-align: right; color: red;"><math>\tau \in \mathbb{R}_+</math> [s]</p>	$e^{-\tau s}$
<u>second order (oscillator)</u>	$T_1^2 \frac{d^2 y(t)}{dt^2} + T_2 \frac{dy(t)}{dt} + y(t) = ku(t)$	$\frac{k}{T_1^2 s^2 + T_2 s + 1}$

$k \in \mathbb{R}; T_1, T_2 \in \mathbb{R}_+$

# Proportional element

1. Element equation:  $y(t) = ku(t)$   $u(t)$  - input,  $y(t)$  - output

$$\mathcal{L} \rightarrow Y(s) = k \cdot U(s)$$

$$H(s) = \frac{Y(s)}{U(s)} = k$$

$$H(j\omega) = k; \quad P(\omega) = k; \quad Q(\omega) = 0$$

$$A = \sqrt{P^2 + Q^2} = |k| \rightarrow L(\omega) [\text{dB}] = 20 \log |k|$$

$$\varphi = \arctan \frac{Q}{P} = \arctan(0) = 0$$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):

for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$

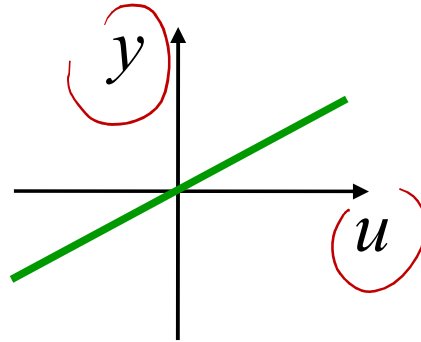
input = const.  
 $y = k \cdot u$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



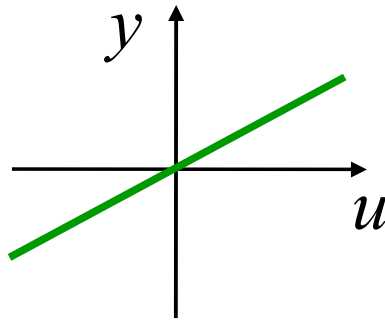
for  $k > 0$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



for  $k > 0$

---

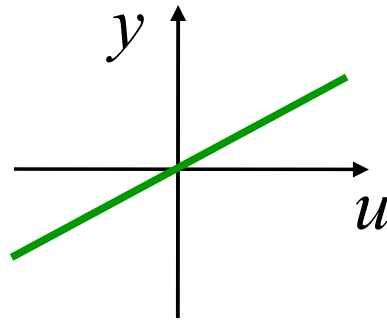
3. Transfer function:

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



for  $k > 0$

---

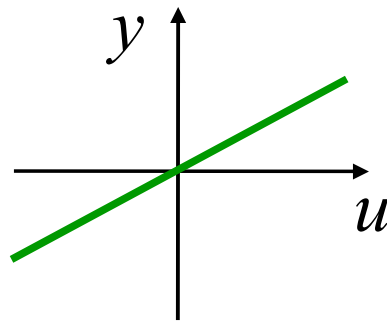
3. Transfer function:  $H(s) = k$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



for  $k > 0$

---

3. Transfer function:  $H(s) = k$

---

4. Step response:  $U(s) = u_0 \frac{1}{s}$

for  $u(t) = u_0 1(t)$

$$Y(s) = H(s) \cdot U(s) = k \cdot u_0 \frac{1}{s}$$

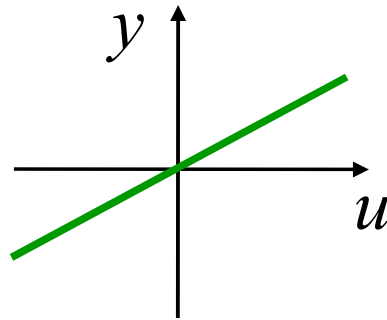
$$y(t) = \mathcal{L}^{-1} \left\{ k u_0 \frac{1}{s} \right\} = k u_0 1(t)$$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



for  $k > 0$

---

3. Transfer function:  $H(s) = k$

---

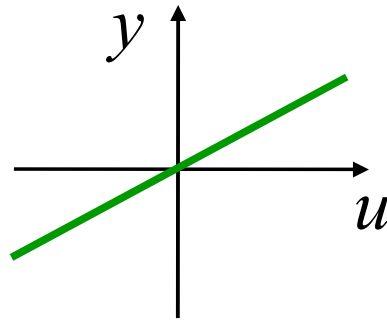
4. Step response:  $y(t) = ku_0 1(t)$       for  $u(t) = u_0 1(t)$

# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



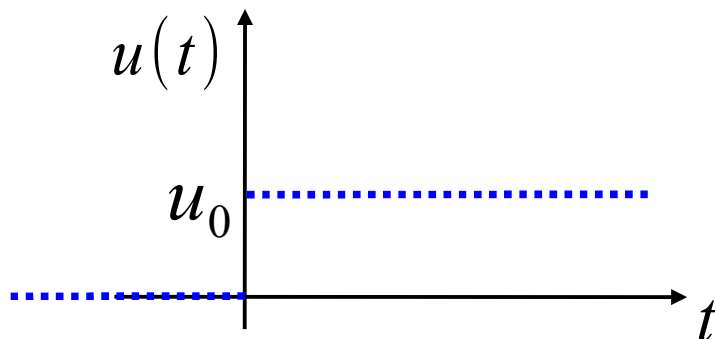
for  $k > 0$

---

3. Transfer function:  $H(s) = k$

---

4. Step response:  $y(t) = k u_0 1(t)$       for  $u(t) = u_0 1(t)$

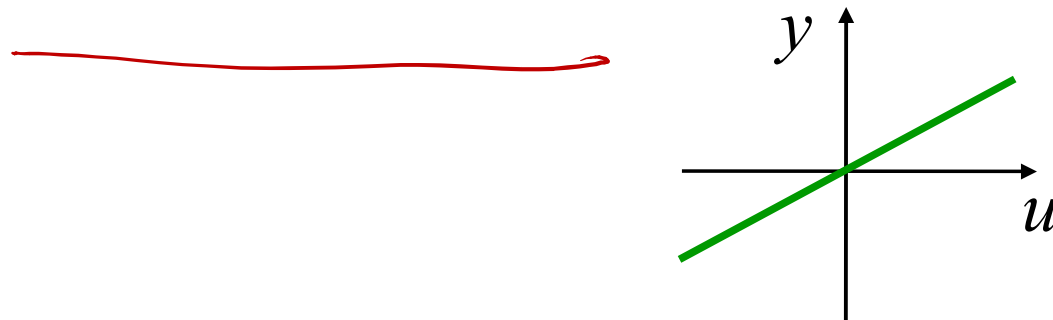


# Proportional element

1. Element equation:  $y(t) = ku(t)$        $u(t)$  - input,  $y(t)$  - output

---

2. Static characteristic (steady state):  $y = ku$       for  $\frac{dy}{dt} = 0 \wedge \frac{du}{dt} = 0$



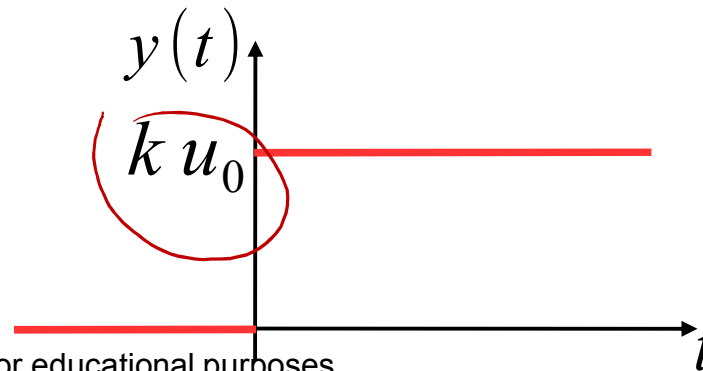
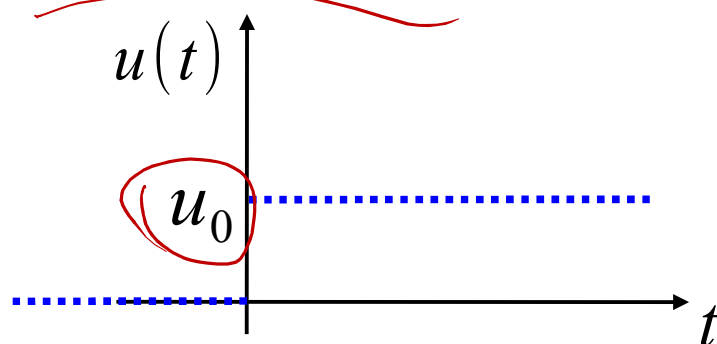
for  $k > 0$

---

3. Transfer function:  $H(s) = k$

---

4. Step response:  $y(t) = k u_0 1(t)$       for  $u(t) = u_0 1(t)$



# Proportional element

---

## 5. Frequency response:

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

---

6. Nyquist plot:

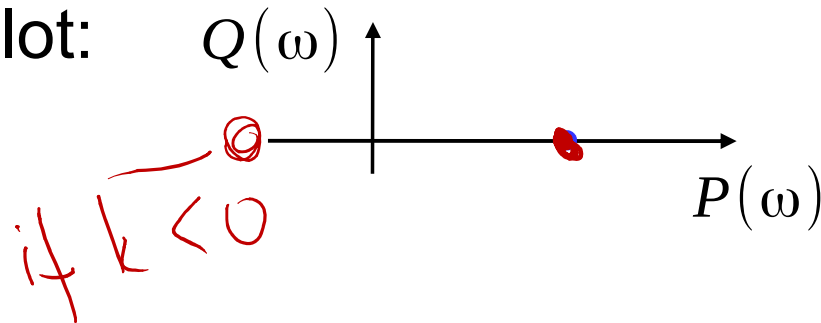
# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

---

6. Nyquist plot:



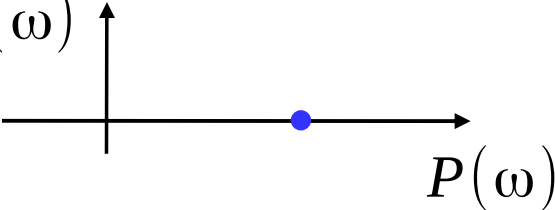
for  $k > 0$

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

---

6. Nyquist plot:   $Q(\omega)$  for  $k > 0$   
 $P(\omega)$

---

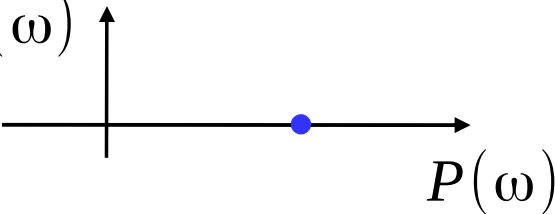
7. Bode plot:

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

---

6. Nyquist plot:       for  $k > 0$

---

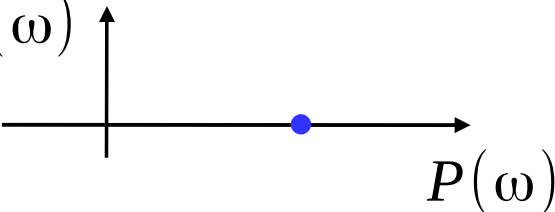
7. Bode plot:  $A(\omega) = \sqrt{P^2 + Q^2} = |k|$   
 $L(\omega) = 20 \log A(\omega)$

# Proportional element

---

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

---

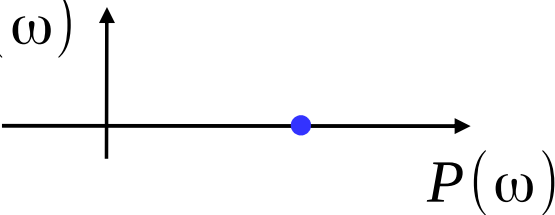
6. Nyquist plot:       for  $k > 0$

---

7. Bode plot:  $A(\omega) = \sqrt{P^2 + Q^2} = |k|$        $\varphi(\omega) = \arctan \frac{Q}{P} = \begin{cases} 0, & \text{dla } k \geq 0 \\ \pi, & \text{dla } k < 0 \end{cases}$   
 $L(\omega) = 20 \log A(\omega)$

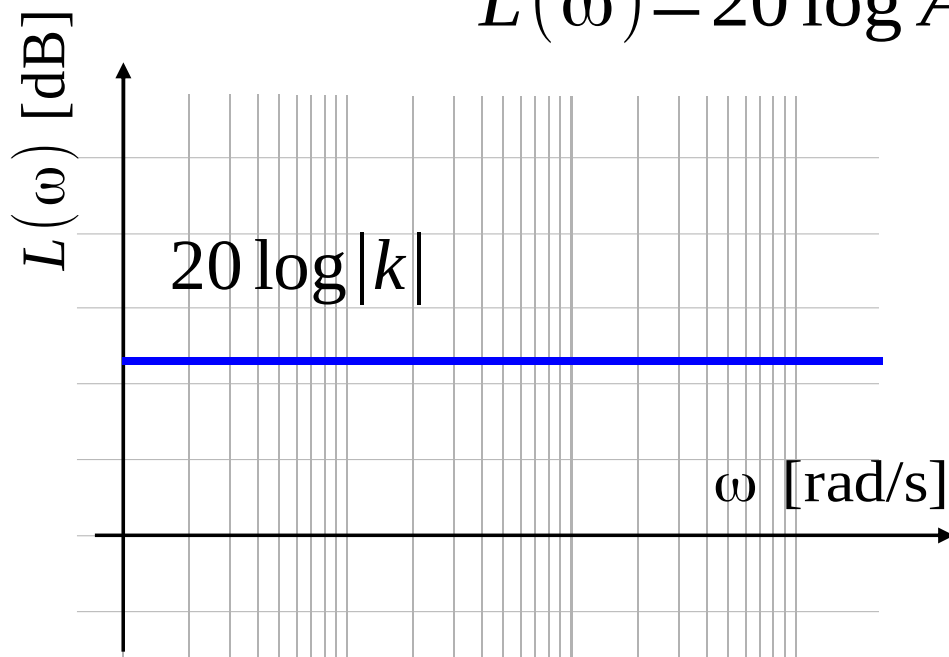
# Proportional element

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

6. Nyquist plot:  for  $k > 0$

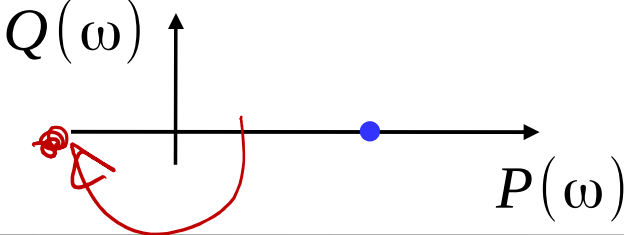
The Nyquist plot shows a horizontal axis labeled  $P(\omega)$  and a vertical axis labeled  $Q(\omega)$ . A single blue dot is plotted on the positive  $P(\omega)$  axis, representing the frequency response for  $k > 0$ .

7. Bode plot:  $A(\omega) = \sqrt{P^2 + Q^2} = |k|$        $\varphi(\omega) = \arctan \frac{Q}{P} = \begin{cases} 0, & \text{dla } k \geq 0 \\ \pi, & \text{dla } k < 0 \end{cases}$   
 $L(\omega) = 20 \log A(\omega)$

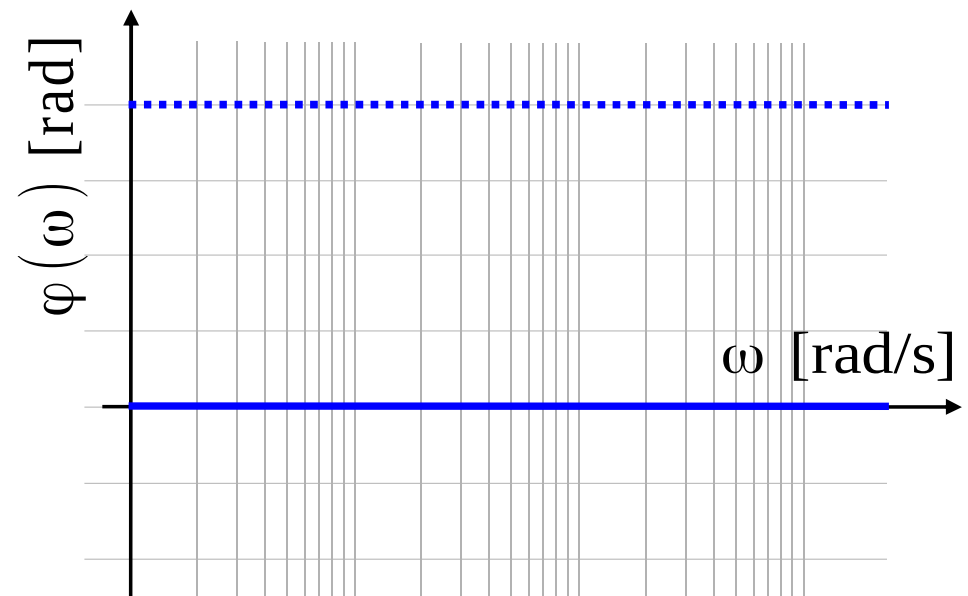
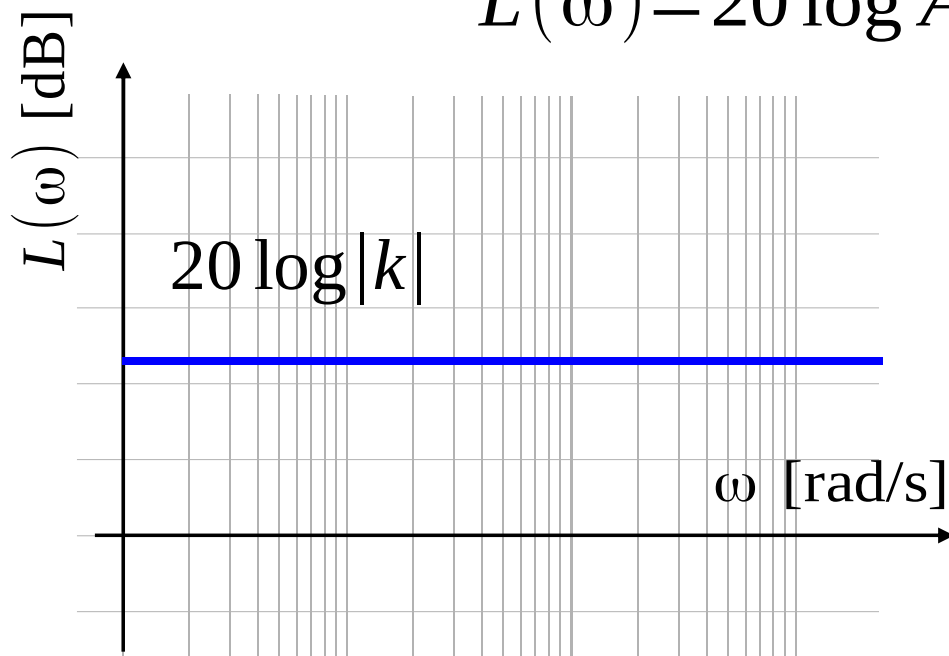


# Proportional element

5. Frequency response:  $H(j\omega) = k$        $P(\omega) = k$ ,  $Q(\omega) = 0$

6. Nyquist plot:  for  $k > 0$

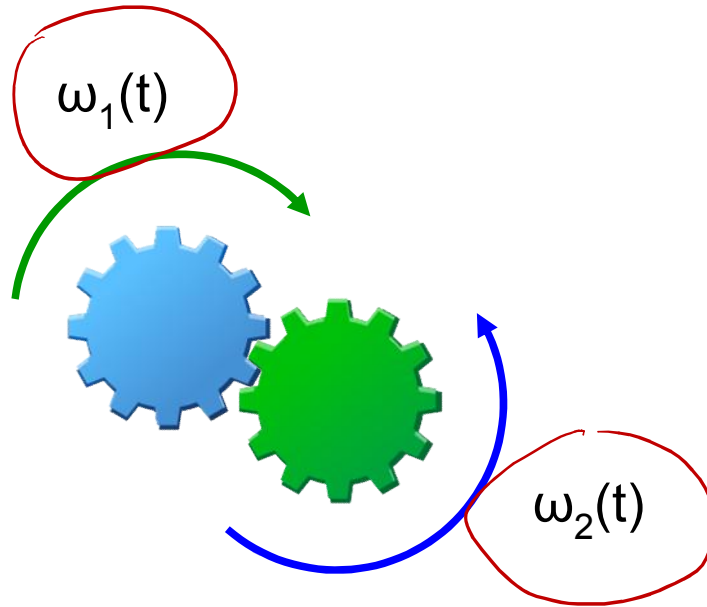
7. Bode plot:  $A(\omega) = \sqrt{P^2 + Q^2} = |k|$        $\varphi(\omega) = \arctan \frac{Q}{P} = \begin{cases} 0, & \text{dla } k \geq 0 \\ \pi, & \text{dla } k < 0 \end{cases}$   
 $L(\omega) = 20 \log A(\omega)$



# Proportional element

## Examples

1

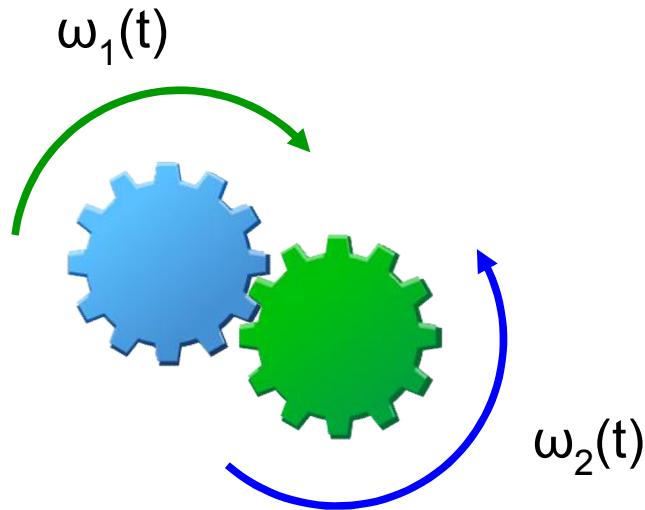


GEARBOX:  
input – angular velocity  $\omega_1(t)$   
output – angular velocity  $\omega_2(t)$

# Proportional element

## Examples

1

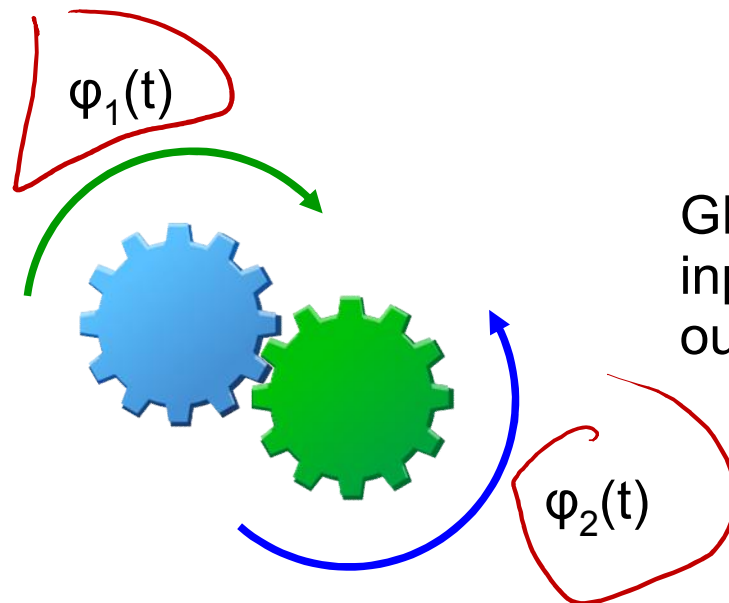


GEARBOX:

input – angular velocity  $\omega_1(t)$

output – angular velocity  $\omega_2(t)$

2



GEARBOX:

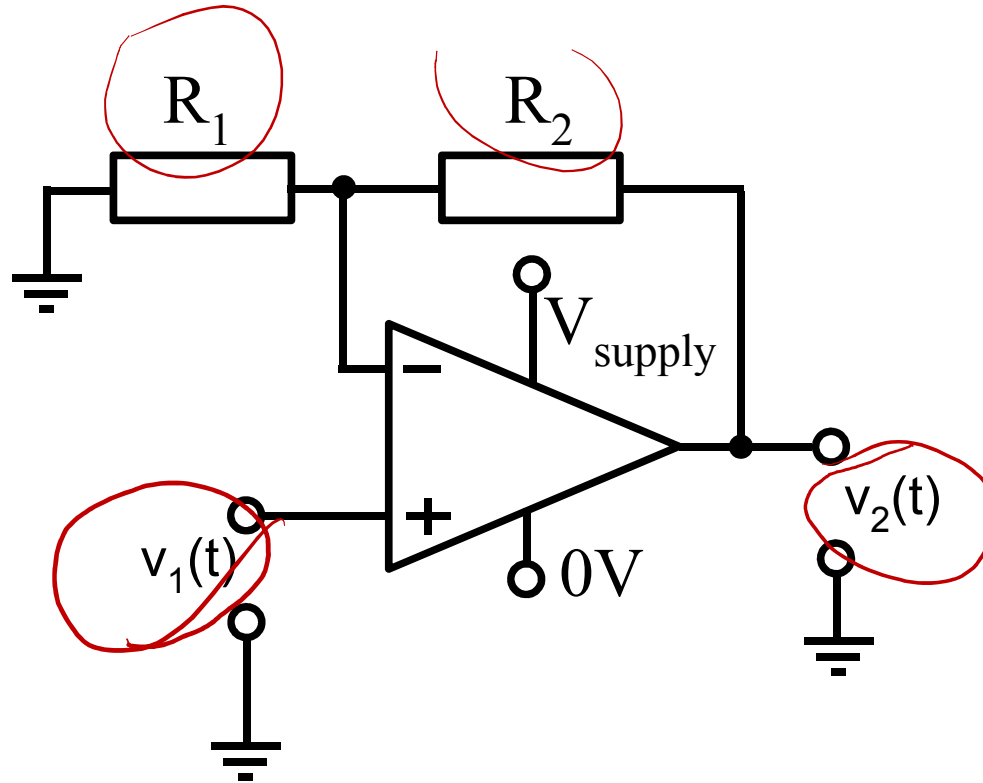
input – rotation angle  $\varphi_1(t)$

output – rotation angle  $\varphi_2(t)$

# Proportional element

## Examples

3



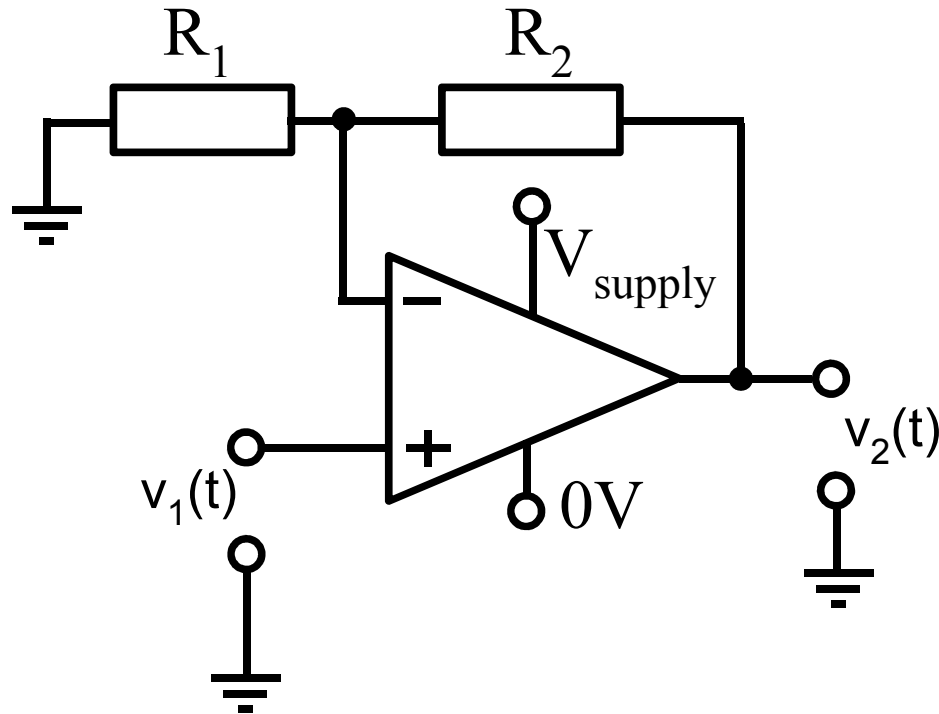
OPERATIONAL AMPLIFIER:  
input – voltage  $v_1(t)$   
output – voltage  $v_2(t)$

$$v_2(t) = v_1(t) \left( 1 + \frac{R_2}{R_1} \right)$$

# Proportional element

## Examples

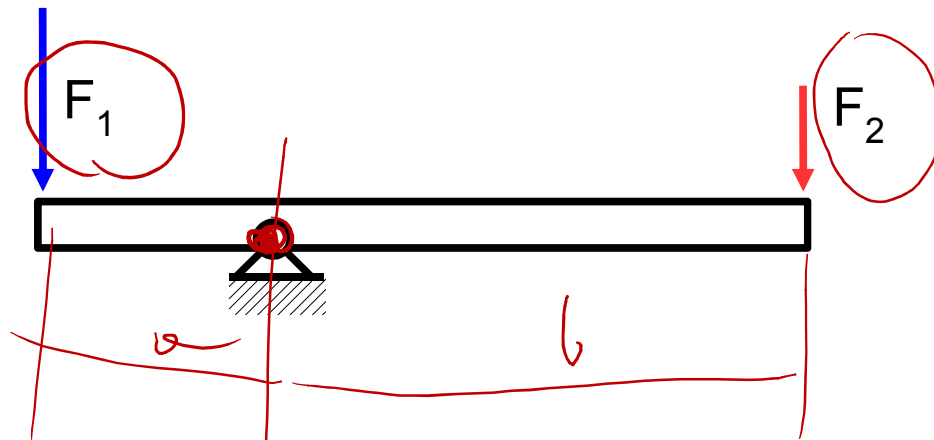
3



OPERATIONAL AMPLIFIER:  
input – voltage  $v_1(t)$   
output – voltage  $v_2(t)$

$$v_2(t) = v_1(t) \left( 1 + \frac{R_2}{R_1} \right)$$

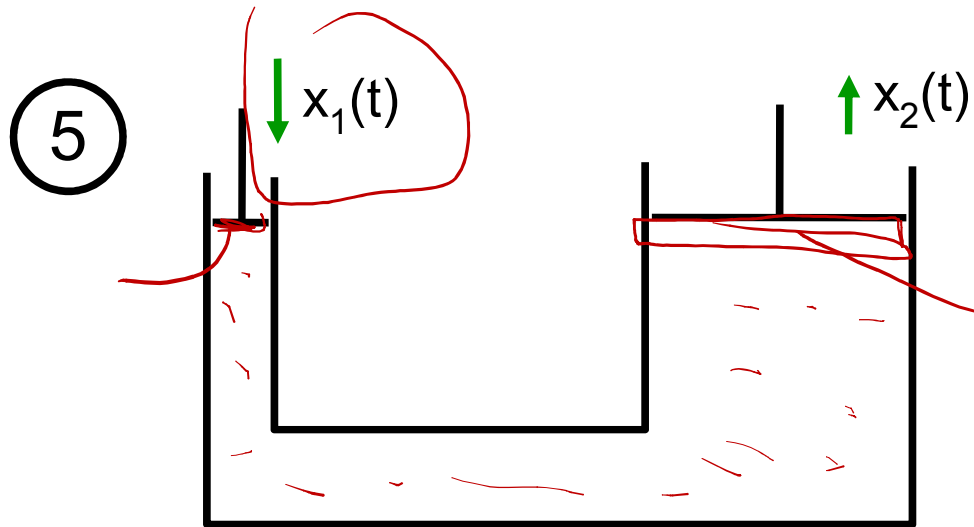
4



BEAM in steady state:  
input – force  $F_1$   
output – force  $F_2$

# Proportional element

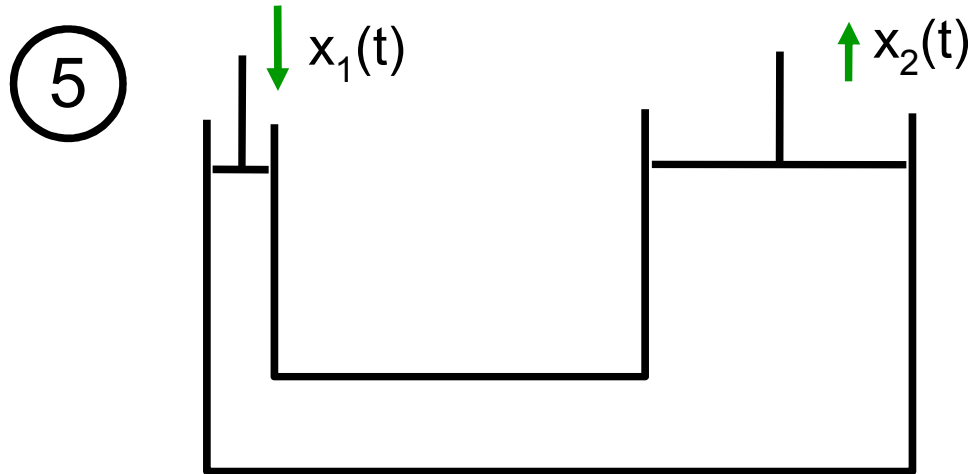
## Examples



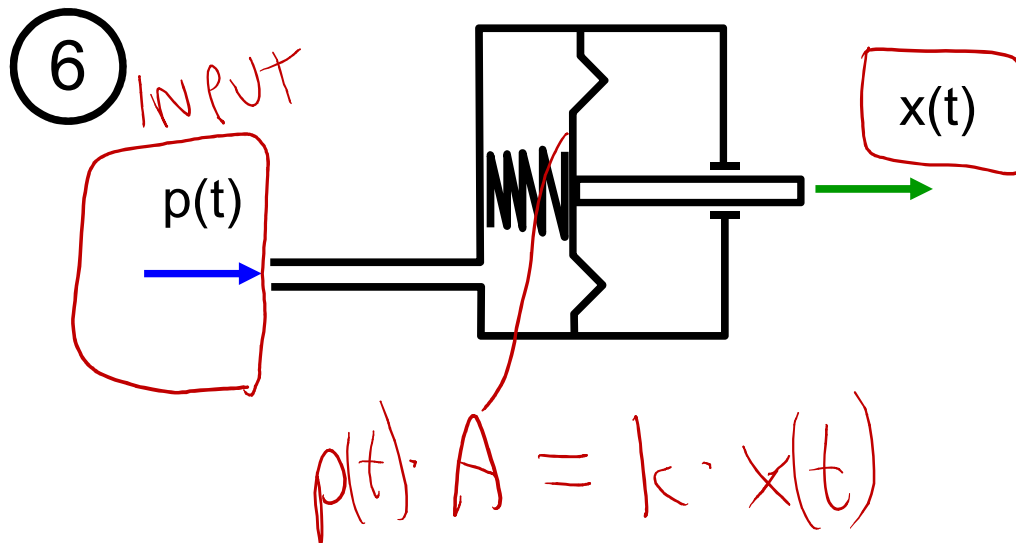
HYDRAULIC LEVER:  
input – displacement  $x_1(t)$   
output – displacement  $x_2(t)$

# Proportional element

## Examples



HYDRAULIC LEVER:  
input – displacement  $x_1(t)$   
output – displacement  $x_2(t)$



PRESSURE ACTUATOR:  
input – pressure  $p_1(t)$   
output – displacement  $x(t)$

# WolframAlpha



transfer function  $(8*s+4)/(2*s^4+7*s^3+11*s^2+19*s+6)$



[Examples](#) [Random](#)

Input interpretation:

systems model

transfer function 
$$\frac{4 + 8 s}{6 + 19 s + 11 s^2 + 7 s^3 + 2 s^4}$$

# WolframAlpha



partial fraction decomposition  $s/(s^3+4*s^2+5*s+2)$



[Examples](#) [Random](#)

Assuming "s" is a variable | Use as a [unit](#) instead

Input

partial fractions

$$\frac{s}{s^3 + 4s^2 + 5s + 2}$$

Result:

[Step-by-step solution](#)

$$\frac{s}{s^3 + 4s^2 + 5s + 2} = -\frac{2}{s + 2} + \frac{2}{s + 1} - \frac{1}{(s + 1)^2}$$

# WolframAlpha



inverse laplace transform  $s/(s^3 + 4s^2 + 5s + 2)$



Examples Random

Assuming "s" is a variable | Use as a [unit](#) instead

Input:

$$\mathcal{L}_s^{-1}\left[\frac{s}{s^3 + 4s^2 + 5s + 2}\right](t)$$

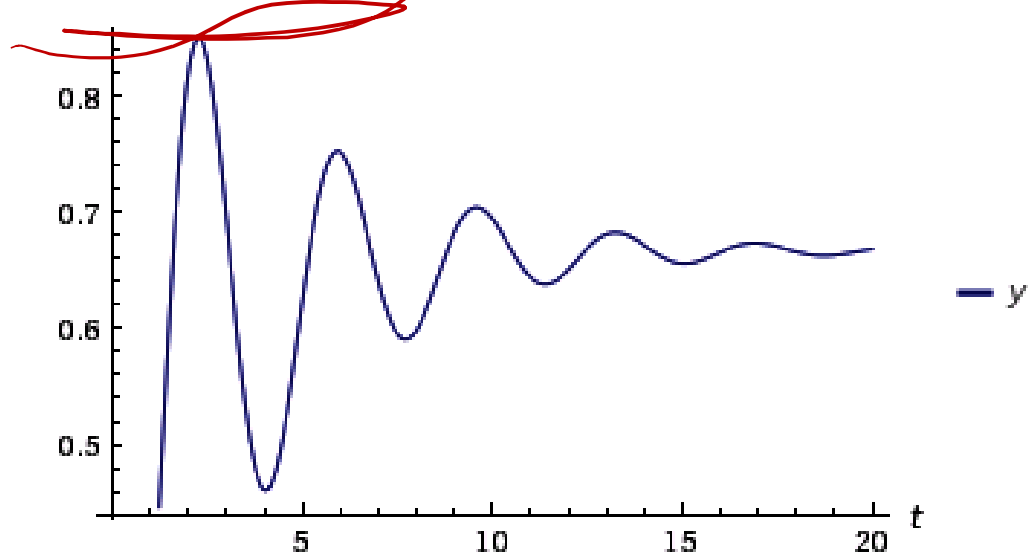
$\mathcal{L}_s^{-1}[f(s)](t)$  is the inverse Laplace transform of  $f(s)$  with real variable  $t$

Result:

$$-e^{-2t} (e^t t - 2e^t + 2)$$

# WolframAlpha

Unit step response plot:



Less time

More time

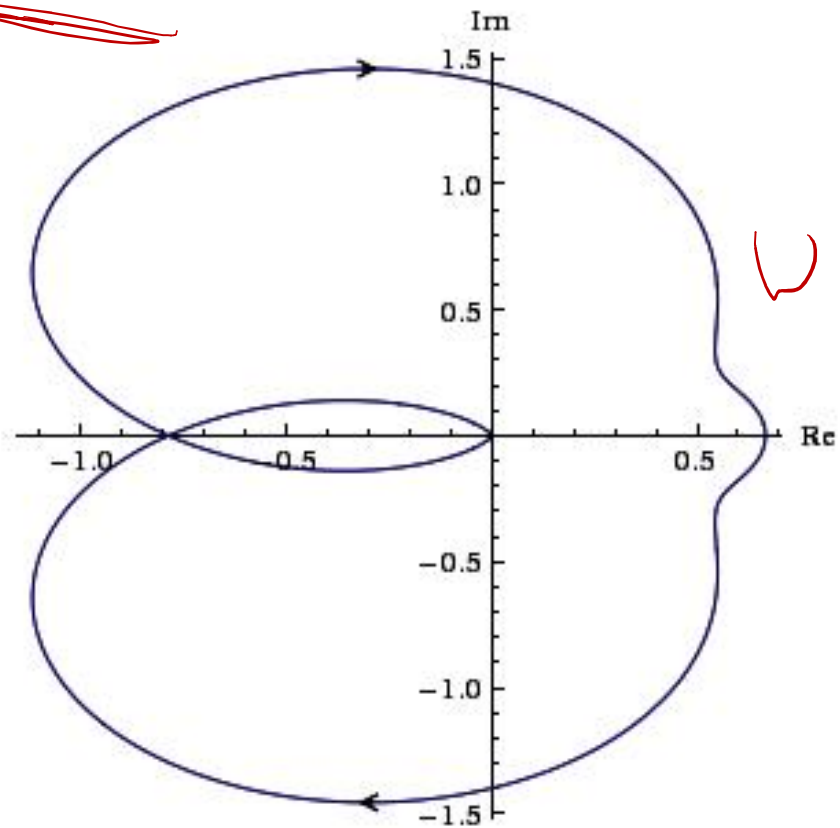
Unit step ▼

# WolframAlpha

Nyquist plot:

Show Nyquist grid

Show stability margins



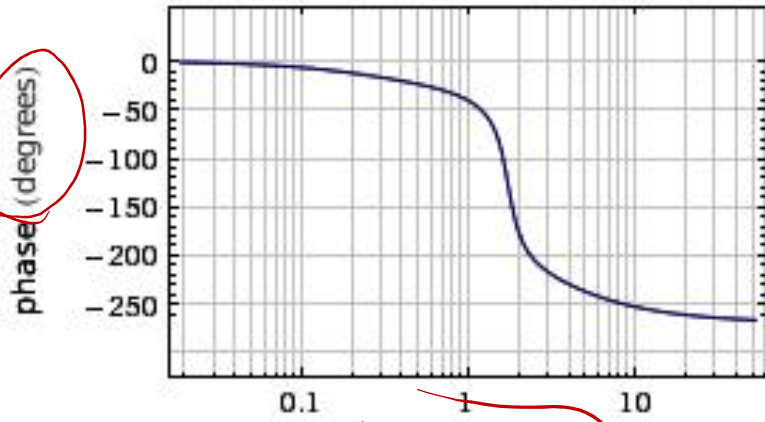
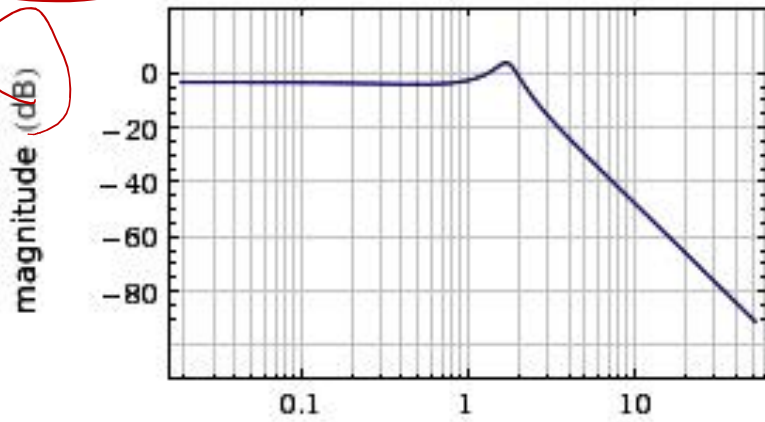
$$w = (-\infty, +\infty)$$

$$\frac{Y(s)}{U(s)}$$

# WolframAlpha

Bode plot:

Show stability margins



frequency Hz

$\frac{Y(s)}{U(s)}$